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NUMERO: 2308A

ANNO: 2018

# **A P P U N T I**

STUDENTE: Chiforeanu Loredana

MATERIA: Structural Mechanics II - Lusas guide - Prof. Cornetti  
-Sapora.pdf

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**POLITECNICO DI TORINO**

**MASTER COURSE IN CIVIL ENGINEERING**



**Course of STRUCTURAL MECHANICS II, YEAR 2017/2018**

# Lusas notes

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## How to analyse a structure in Lusas step by step

**Professor:** Cornetti Pietro  
**Author:** Loredana Mihaela Chiforeanu

**In this report you can find:**

- exercises solved in class
- explanations step by step
- images of each part and final results
- examples of exam exercises

**My other notes of Master courses:**

- Theory of Structural Mechanics II + Exam 2018, prof. P.Cornetti / Sapora, year 2017/18
- Theory and design of reinforced and prestressed concrete structures, prof. A.P. Fantilli, year 2017/2018
- Theory of Hydrology, prof. S. Tamea, year 2017/2018
- Exercises of Hydrology, prof. S. Tamea, year 2017/2018
- Exercises of Construction of Roads, Railways and Airports, prof. E. Santagata / Dalmazzo/ Riviera, year 2017/18

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- **Intro: calculation of solicitations' diagrams in a given thin beam ("trave sottile")**

The solution of the problem is an approximation because a closed form solution doesn't exist.

Since the 60's of the last century, the finite element method is the most used.

This method consists in discretization and interpolation:

- discretize the structure to simplify it introducing a finite number of nodes and a finite number of elements: from infinite degrees of freedom we pass to a finite number of degrees of freedom: this is done through a mesh;

- the calculation is made on the nodes, in terms of displacements, so we find the nodal displacements. Then from displacements we calculate stresses and other quantities;

- we pass from partial differential equations to ordinary differential equations, having a much easier solution.

$I_{zz}$  is the moment of inertia in the vertical plane around x axis and along z axis.

$M_z$  is a linear bending moment and  $T_y$  is its first derivative (shear constant).

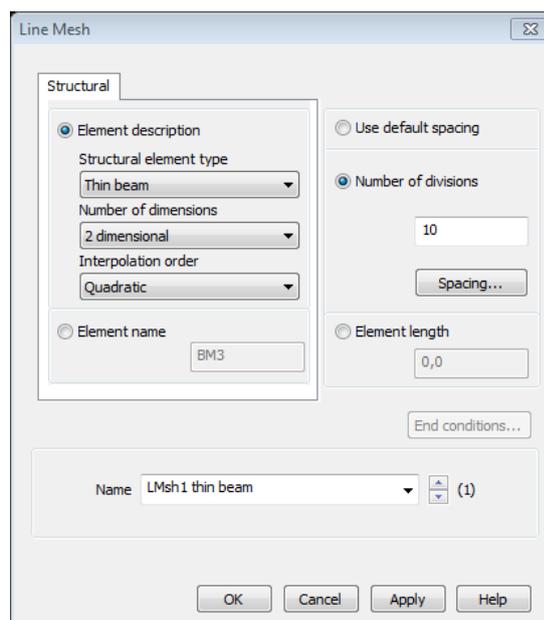
The distribution of  $\tau_{xy}$  is quadratic.

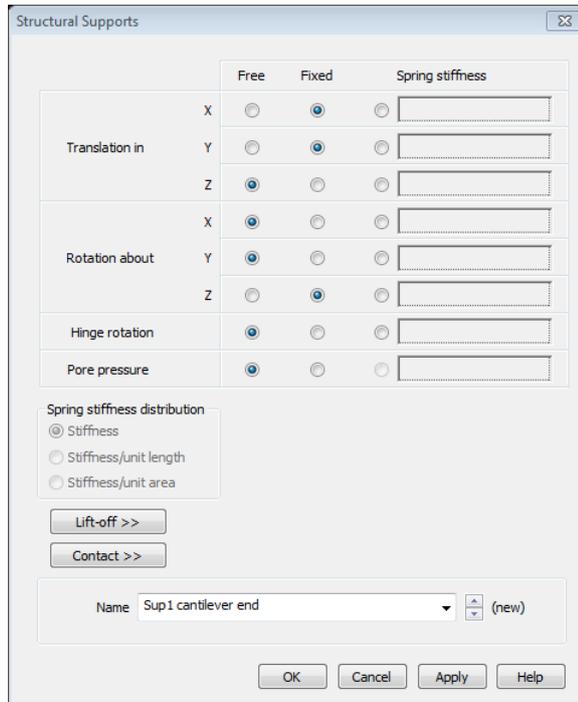
In Lusas we can import a file from Autocad.

- **How to draw in Lusas:**

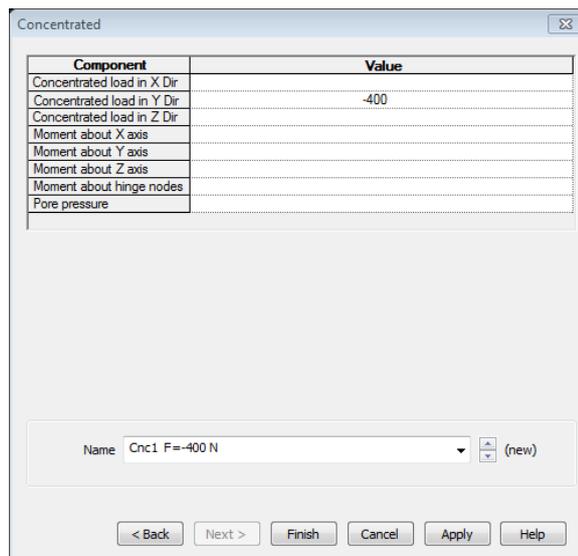
- Start from 2 points and a line in between:

- 1) Introduce coordinates of points (*new point*)  
(pay attention to units that you choose from the beginning!)
- 2) Select the 2 points and click *line*
- 3) Mesh the geometry: *attributes, mesh line, thin beam, 2D, interpolation order: quadratic, number of divisions: 10* (10 cm long each finite element, in total there are 11 nodes), *name: thin beam*

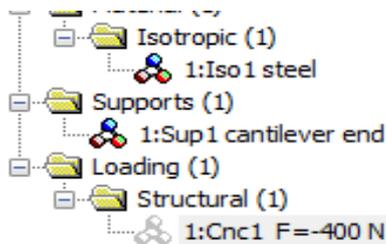




7) Loads: *attributes, loading, concentrated, next, y: -400 (because the force is downwards but y axis is upwards), apply, finish*



8) After defining the properties we have to select the beam and assign them to it: *left click, assign;* (the three cubes color).



!!! The supports must be assigned to the points (not to line in this case)

- **Calculation of stress in a given rectangular cross section**

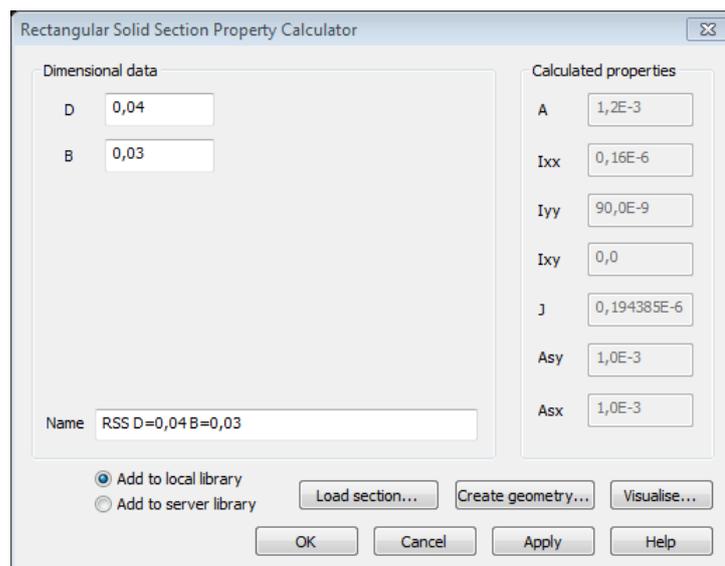
- All assigned attributes are taken away from the thin beam (but the attributes remain in the memory): *Attributes, deassign, from all.*
- Create a new **Thick beam** (“trave spessa”): *attributes, mesh, line, thick beam, 2 dimensional, linear interpolation, number of subdivisions 10, apply, ok.*

!!! If you increase the number of elements, there is a value after which the solution doesn't change visibly, in fact the results will differ only for negligible centesimal numbers.

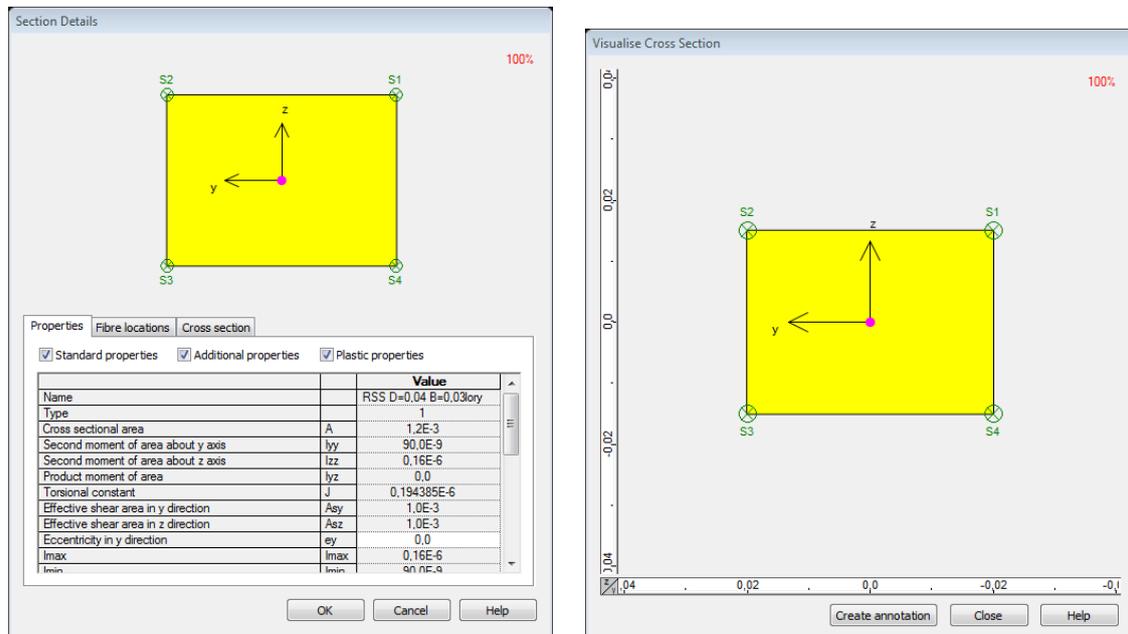
- Assign all properties to the thick beam: *right click, assign to all.*  
Now all the cube-triangles near the thick beam (in the attributes window) are colored.
- Change the line attribute: *attributes, line, 2D thick beam.*

!!! “Prima di assegnare il tipo vincolo nodale e la forza concentrata bisogna selezionare il nodo sul quale si vuole agire, altrimenti ciò verrà applicato a tutti gli 11 nodi” – Before defining the constraints and applying concentrated forces, you have to select the node, otherwise they will be applied automatically on all the 11 nodes of the structure.

- [To not visualize the deformed structure: *right click, deformed mesh* (it must be deselected).]
- Create cross section: *Utilities, sections property calculator, rectangular, solid, add to local library* (the Excel file will be saved in the current folder and it can be used only there (server library is for all folders), *apply, ok.*



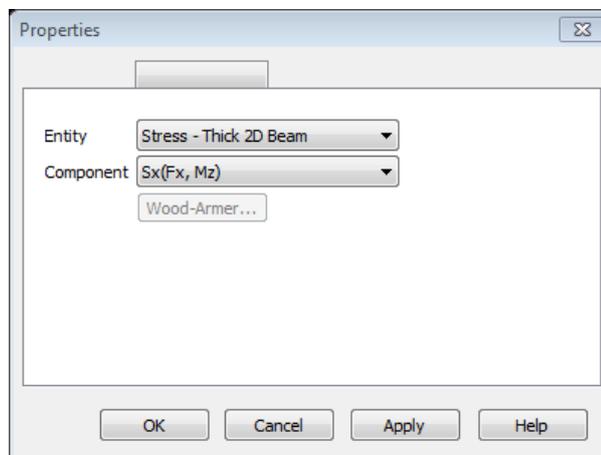
- Assign the section created to the thick beam: *Geometric line, right click on thick beam, definition from library, user sections*, (you can choose the rotation of the angle: *rotation about centroid 0°*).



**RUN!!!**

• **How to see different diagrams:**

you can see the different diagrams and values of stress, reactions (M,N,T) by changing the properties: *layers*, right click on diagram, entity: .... , component: .... .



- Change font, colors, dimensions: *right click on the elements in the main window, font.*

Diagram Mz and reaction Fy:



-activate/deactivate the flashing and change vista to see in 3D and turn to the 2D



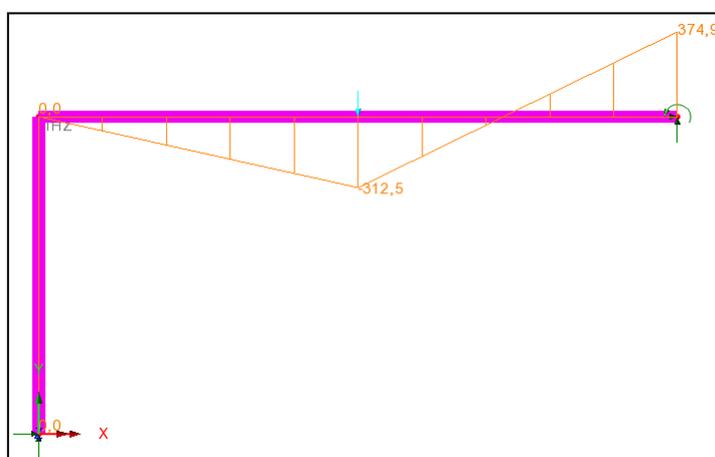
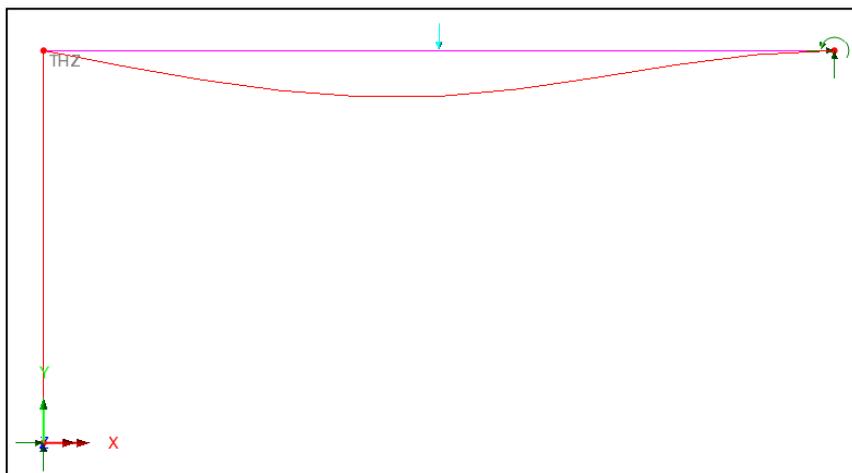
- deassign from selection the mesh, click on end conditions...: start of line: fixed; end of line: rotation around z:pinned (end conditions... is available only for thick beams)

-material, isotropic

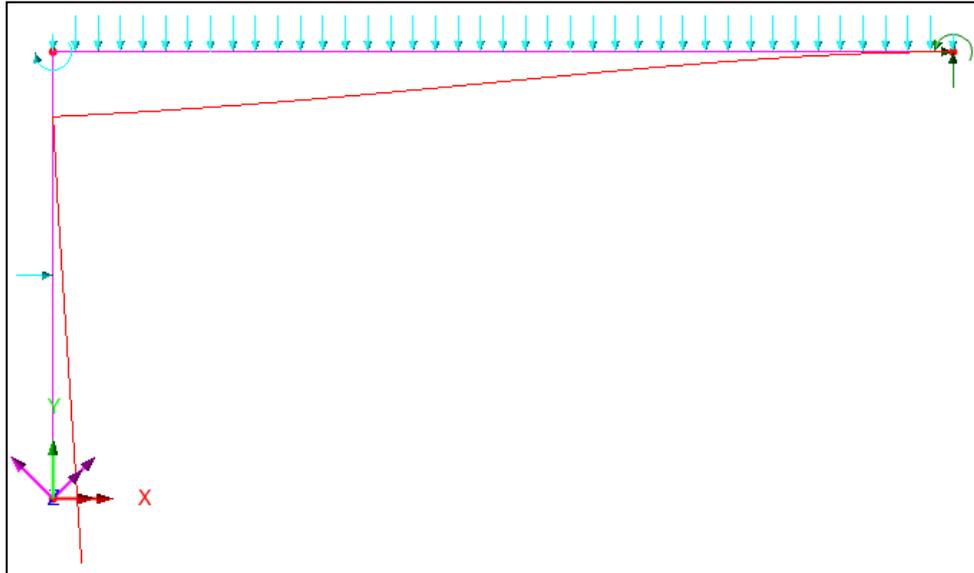
-support ,one is pinned and one is fixed (node 1 and node 3)

-loading, internal beam point, next, global load direction, load position about beam axis, distance type parametric , so you can write  $L/2=0.5$ ; PX,PY are components:  $P_y=-1000$ , name, apply, finish

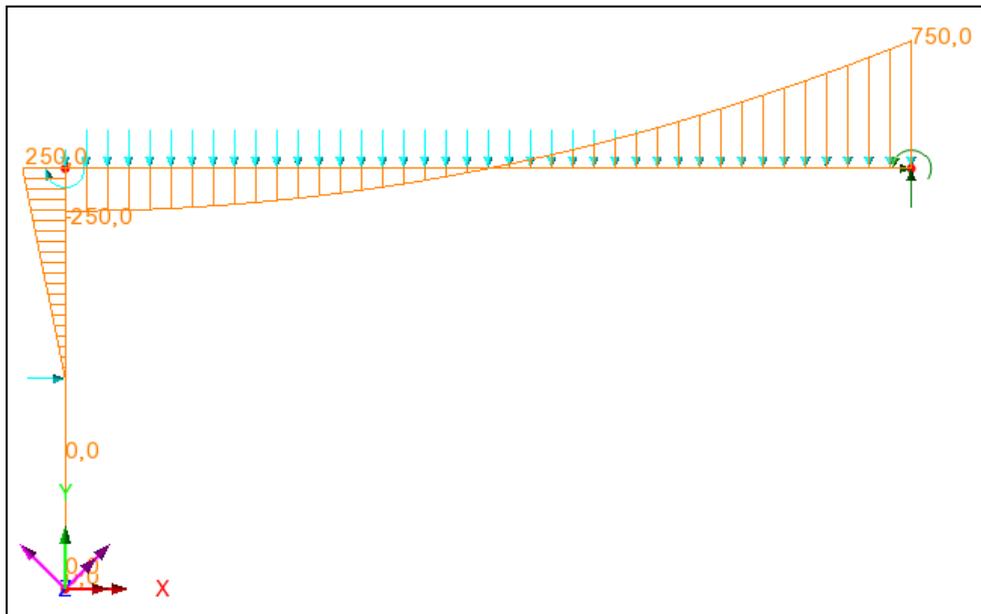
**RUN!!!**



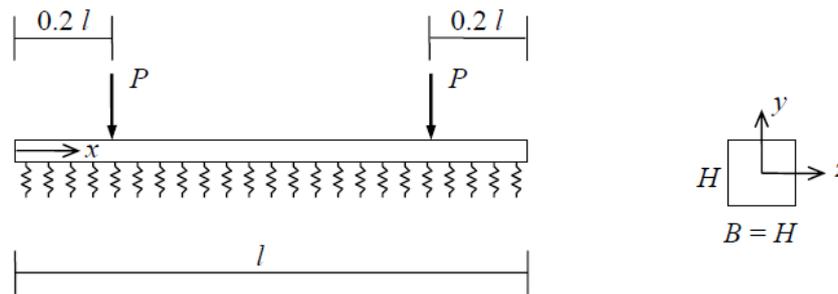
- Deformed mesh:



- Moment  $M_z$  diagram:



# Beam on elastic foundation – Winkler



Data:

$l = 10 \text{ m}$

$B = H = 0.4 \text{ m}$  ( $A = 0.16 \text{ m}^2$ ;  $I_z = 2.13\text{E-}3 \text{ m}^4$ )

$P = 1\text{E}5 \text{ N}$

$E = 30\text{E}9 \text{ Pa}$ ,  $\nu = 0.18$  (concrete)

$K = 4\text{E}7 \text{ Pa}$  (soil stiffness) => Spring stiffness distribution in LUSAS: use "Stiffness/unit length"

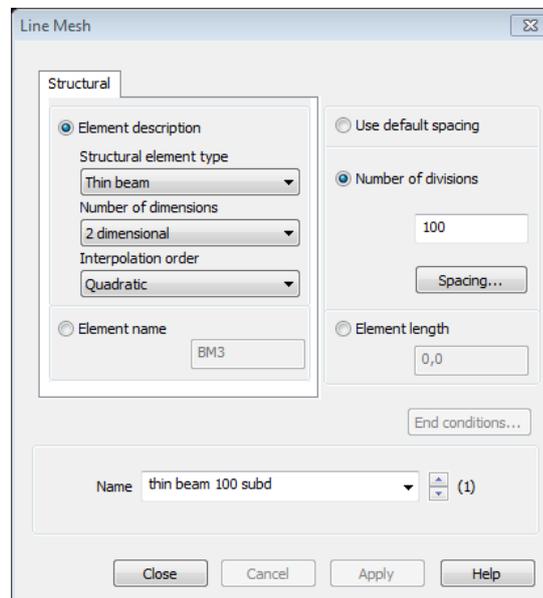
Extreme cases:

1<sup>st</sup> case:  $K = 4\text{E}10 \text{ Pa}$

2<sup>nd</sup> case:  $K = 4\text{E}3 \text{ Pa}$

- **How to draw and assign the soil:**

- Draw a line and assign mesh



- Assign geometry and material

- Assign the loads

Internal Beam Point

Load direction:  Global  Element local

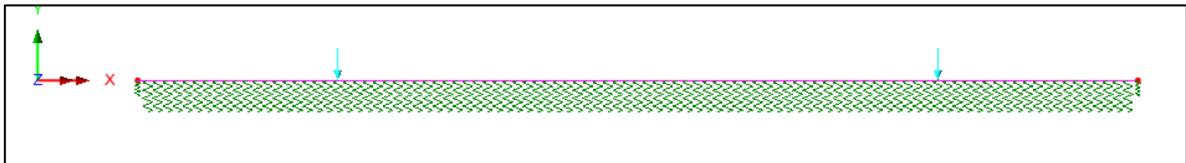
Load position:  About beam axis  About nodal line

Distance type:  Parametric  Actual

	Distance	PX	PY	PZ	MX	MY	MZ
1	0,2	0,0	-100,0E3	0,0	0,0	0,0	0,0
2	0,8	0,0	-100,0E3	0,0	0,0	0,0	0,0
3							

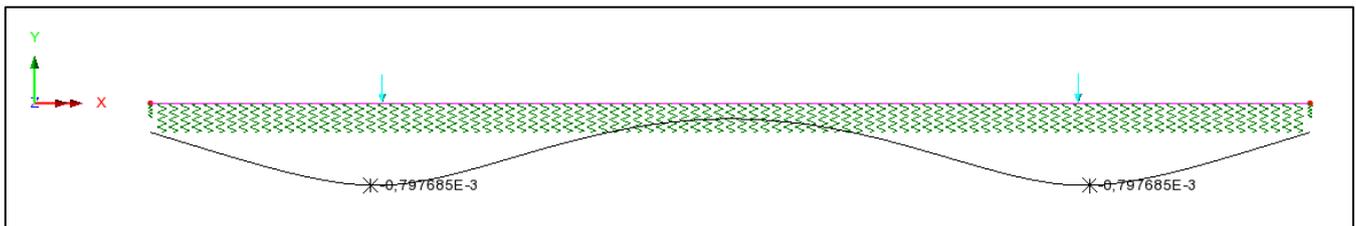
Name: P = -1E5 N (1)

Close Cancel Apply Help

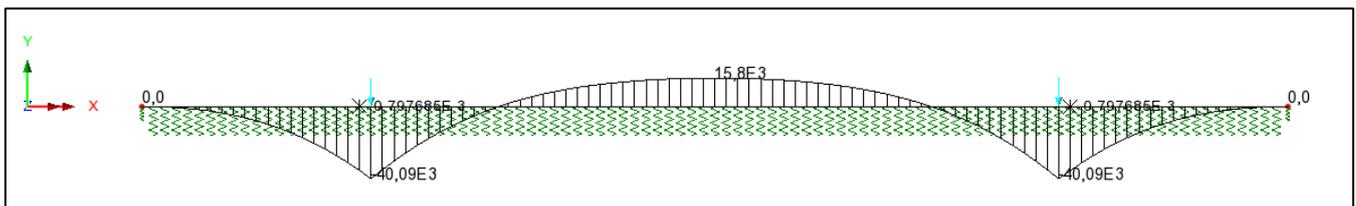


**RUN!!!**

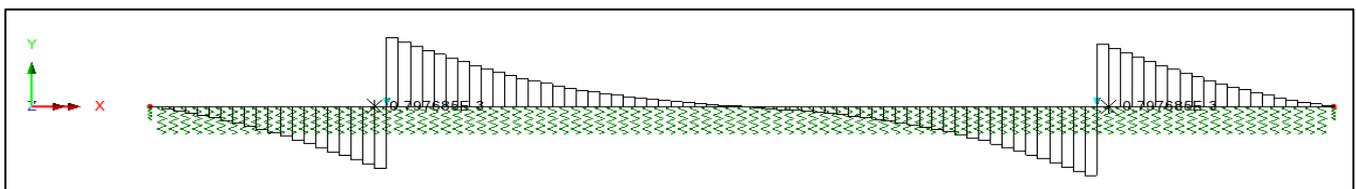
Deformed mesh:

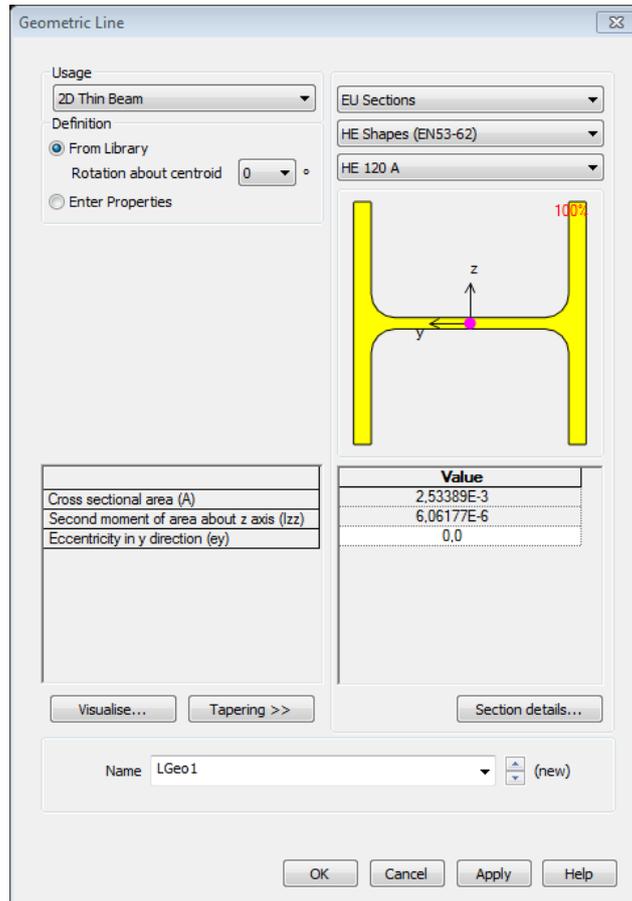


Moment diagram Mz:



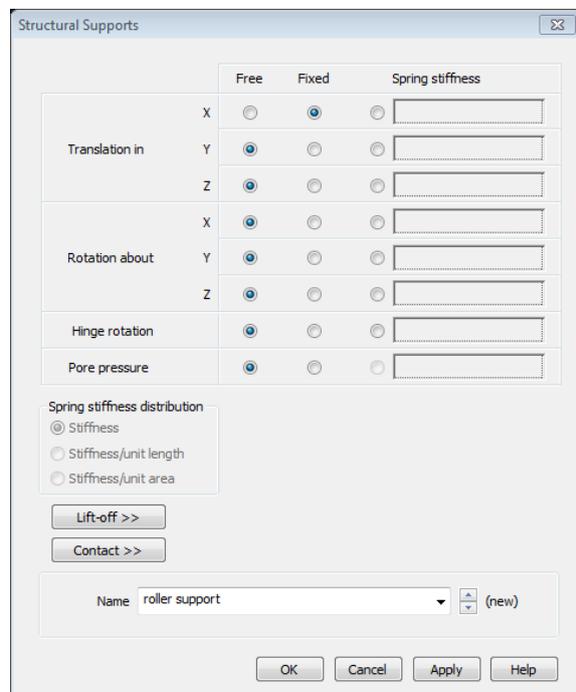
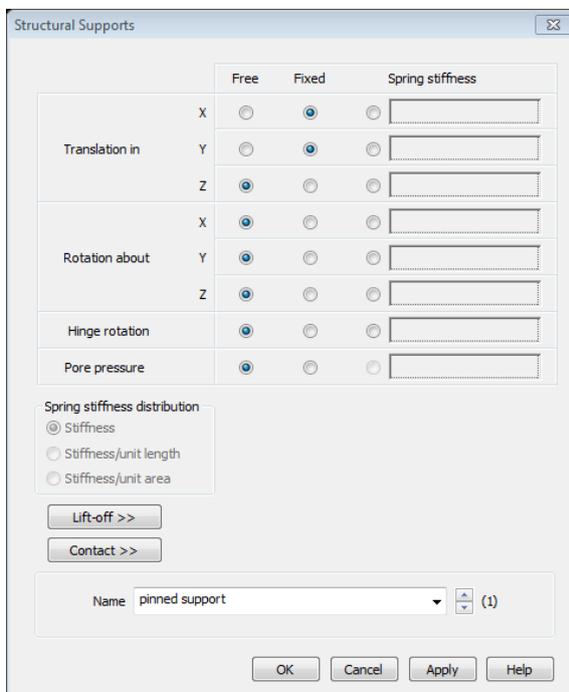
Shear Fy:



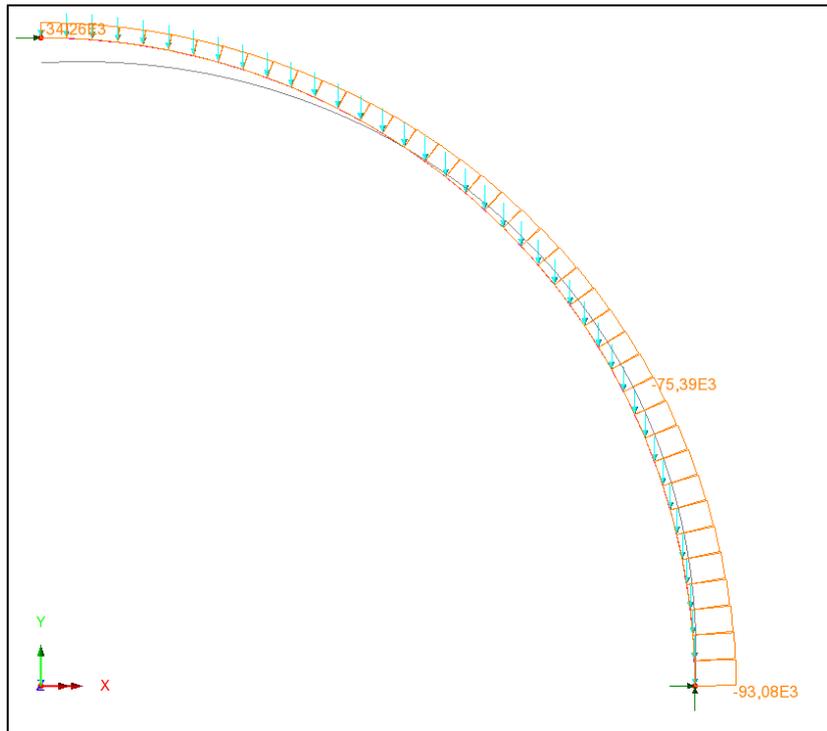


-Attrib, materials, isotropic,...

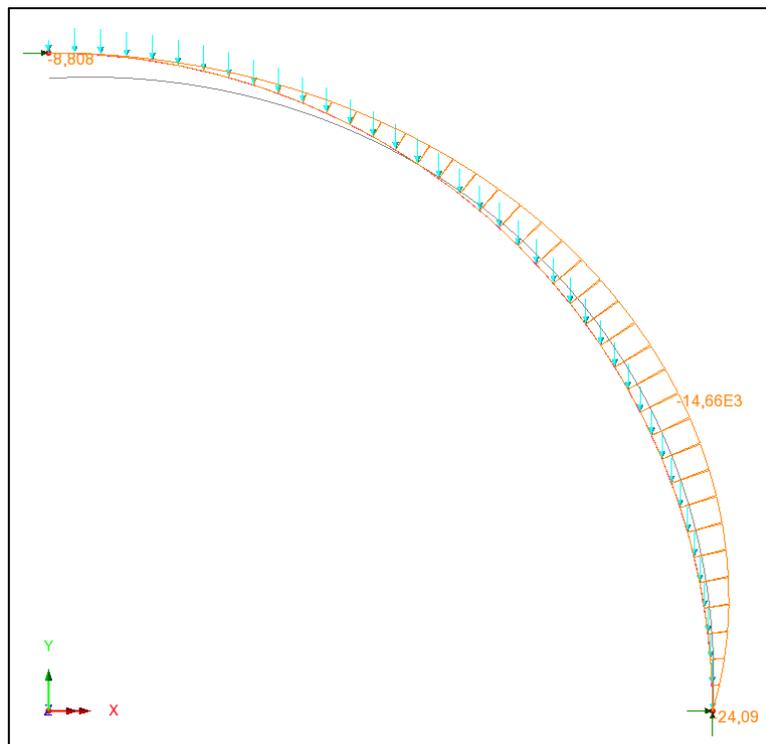
-Attrib, supports,...



**Compression Fx:**

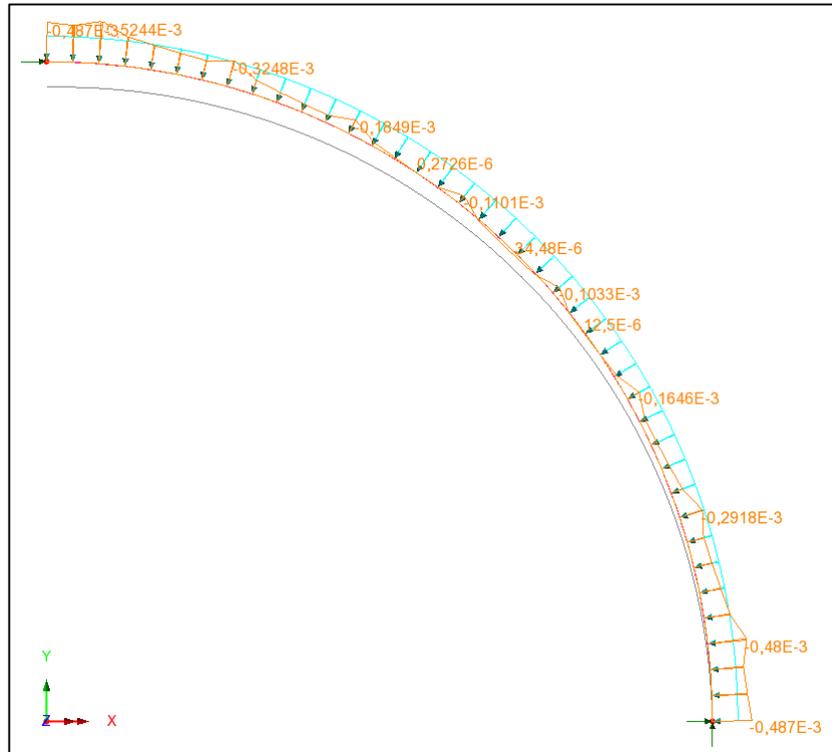


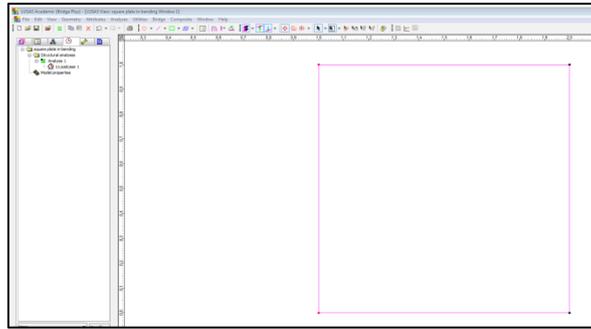
**Bending moment Mz:**



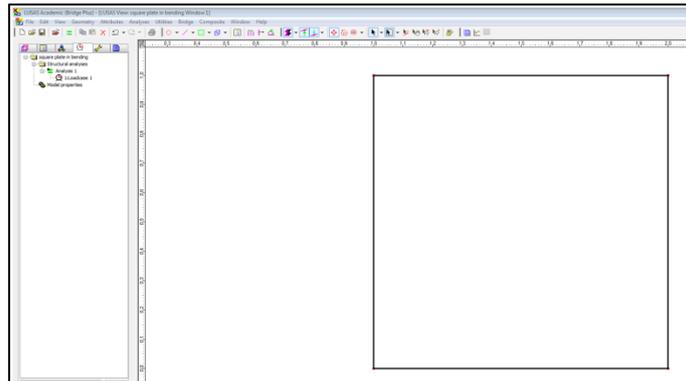
**RUN!!!**

**Moment Mz:**

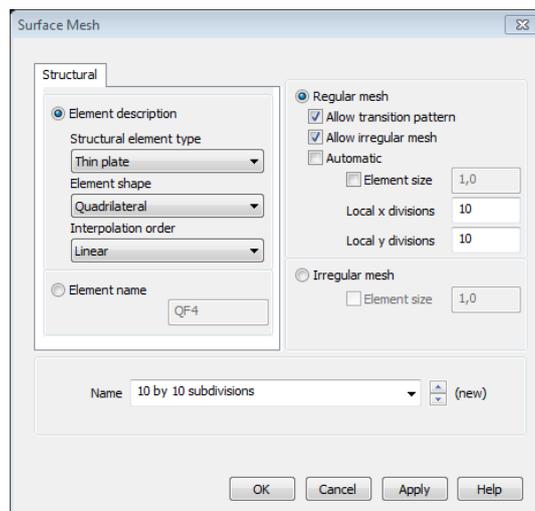




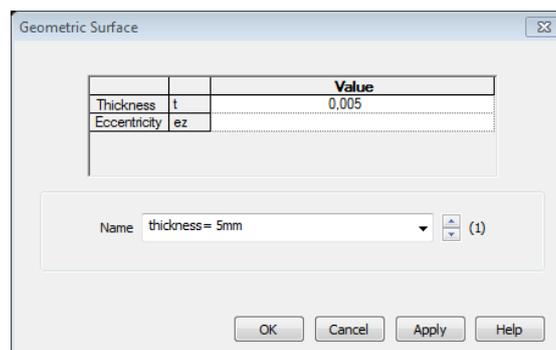
-create a surface: select 2 lines, *geometry, lines, by joining*

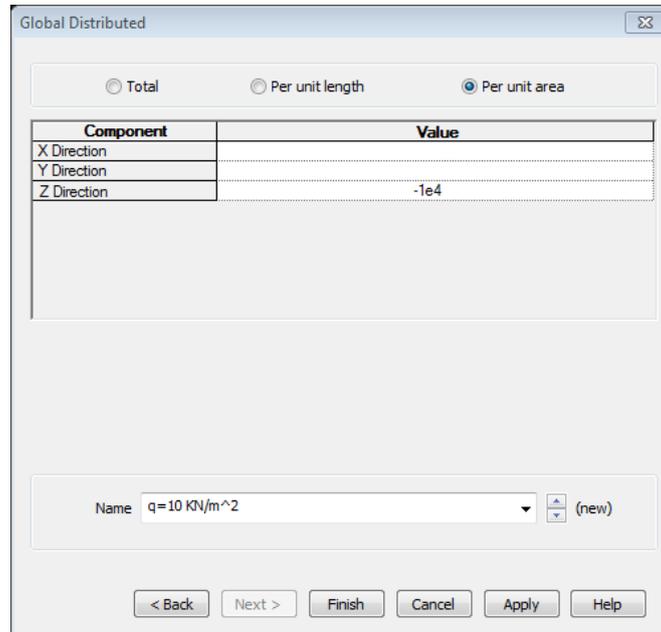


-attributes, mesh, (with regular spacing), surface, thin plate:



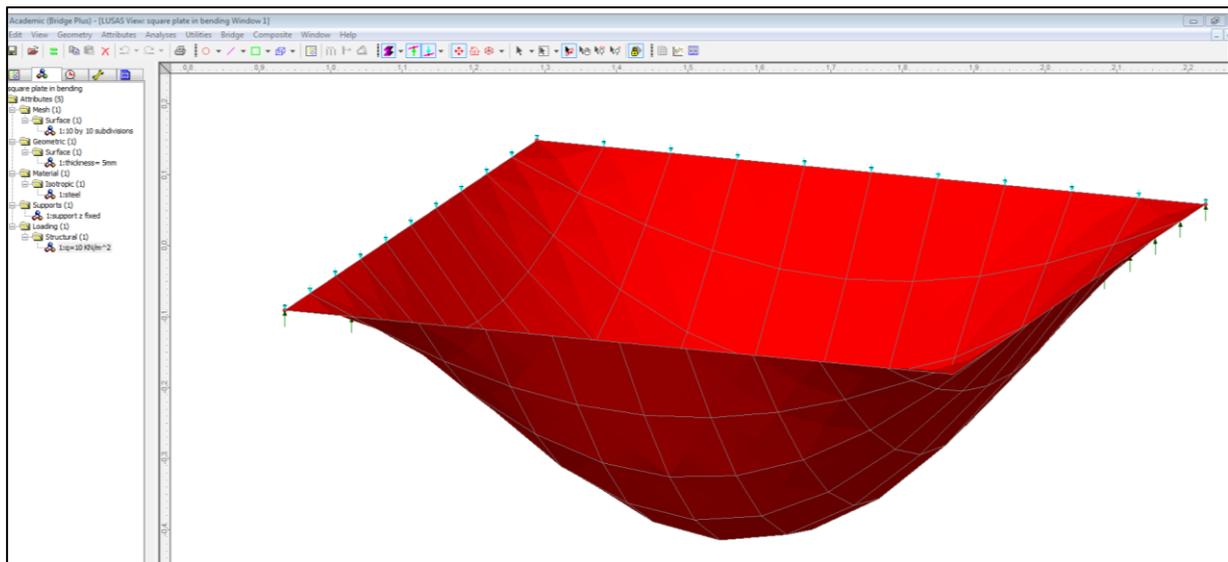
-attributes, geometry, surface:





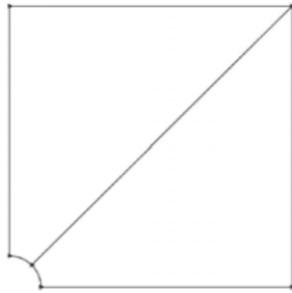
**RUN!!!**

- deformed mesh: magnitude 100 mm, mesh: solid



- **How to draw 1/4 of plate:**

Exploiting the symmetry of the problem, model one fourth of plate:



-Insert points: write the coordinates of the corners squared plate and we can find the 3 nodes of the arch by sweeping 2 times the point (0,5; 0): *geometry, line, by sweeping, rotate, 45°, ok.*

**Enter Coordinates** ✕

Grid style  
 3 columns

	X	Y	Z
1	0,5	0	0
2	4,5	0	0
3	4,5	4,5	0
4	0	4,5	0

Local coordinate  
 Global coordinates ▼

Set as active local coordinate

**Sweep**

Translate   
  Rotate   
  Mirror   
  Scale

Angle:

About axis:
   
 X-axis
   
 Y-axis
   
 Z-axis
   
 Specified

Origin of axis

X	Y	Z
<input type="text" value="0,0"/>	<input type="text" value="0,0"/>	<input type="text" value="0,0"/>

(+ve angles anti-clockwise about the axis)

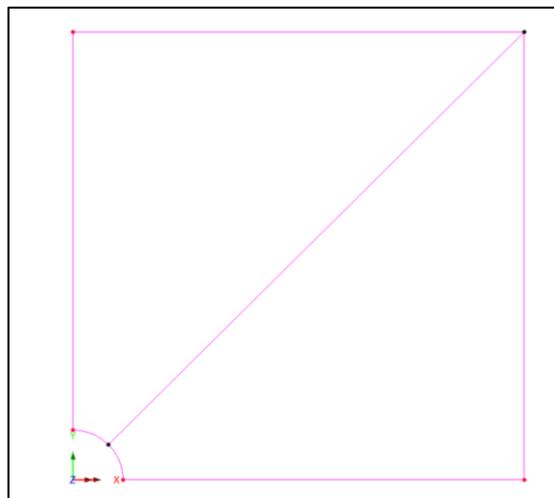
Sweep type  
 Minor arc   
  Major arc   
  Straight

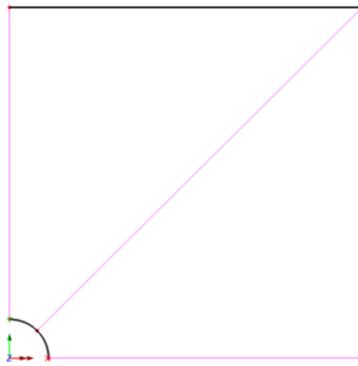
Order of geometry to create  
 Line   
  Surface   
  Volume

Transformations generated from memory selection  
 No transformations generated Use

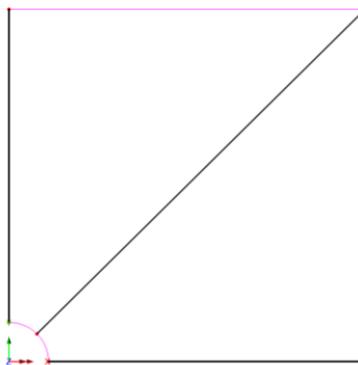
Name:  (new)

-Connect all points by a line in a counter-clockwise direction: *select 2 nodes, new line*



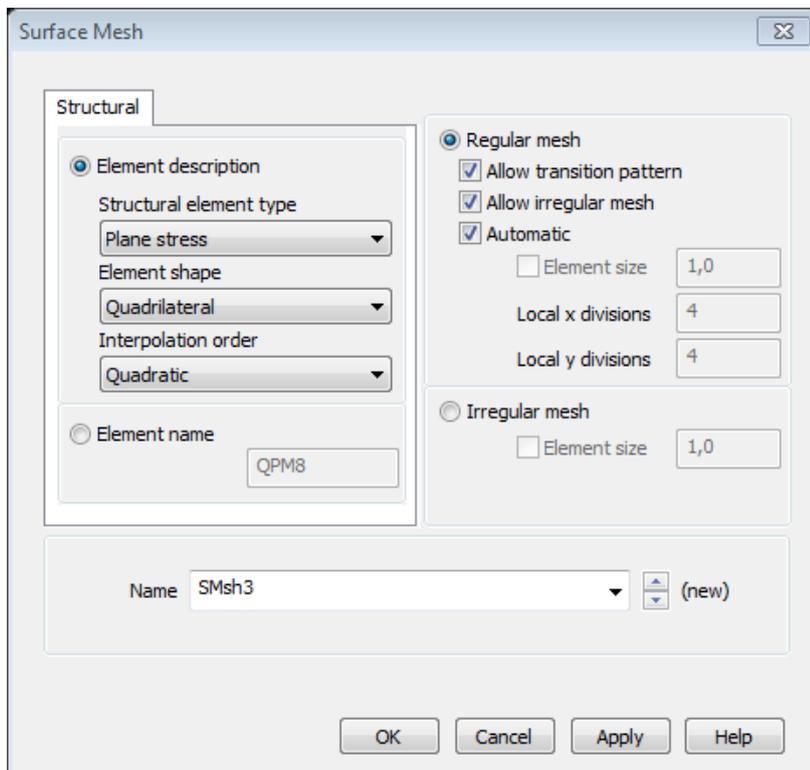


.... And the 9 subdivisions mesh to:

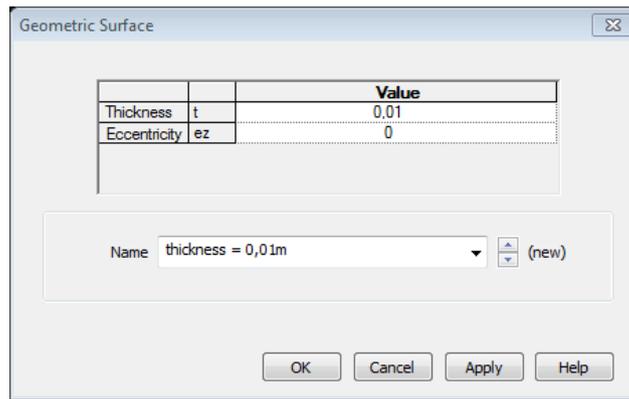


-Geometry ,surface line, (create a triangle on the right and one on the left)

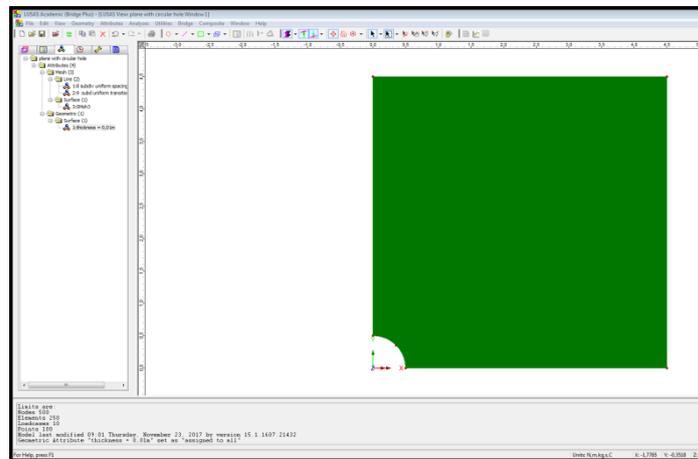
-Attributes, mesh, surface, plane stress :



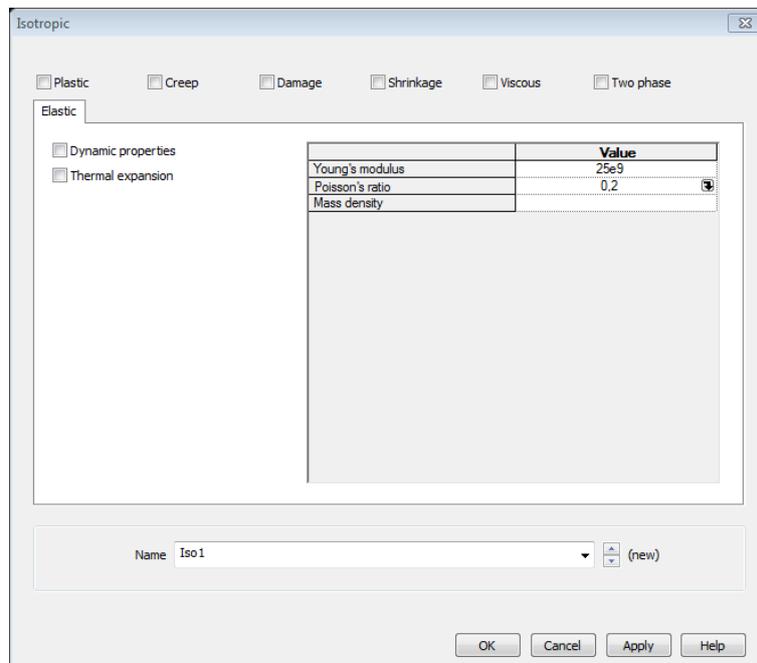
*-Attributes, geometric, surface*



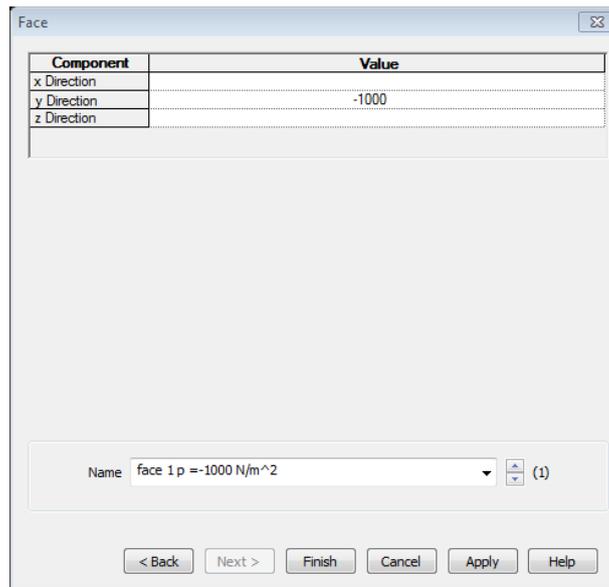
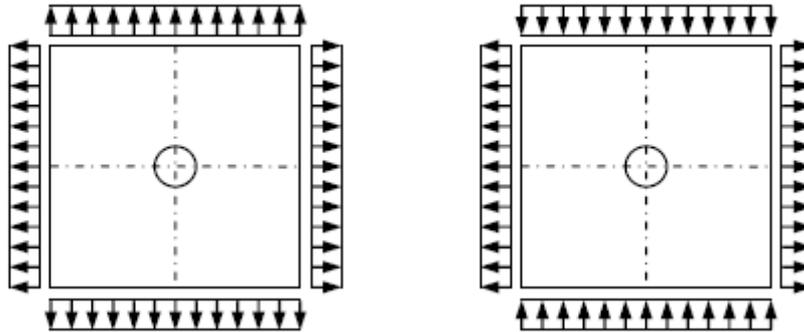
-Assign it:



*-Attributes, materials, isotropic*

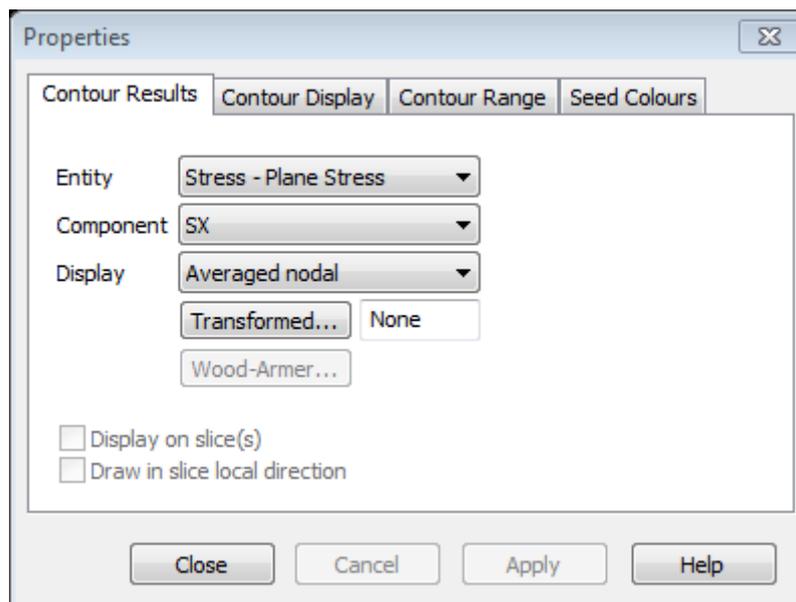


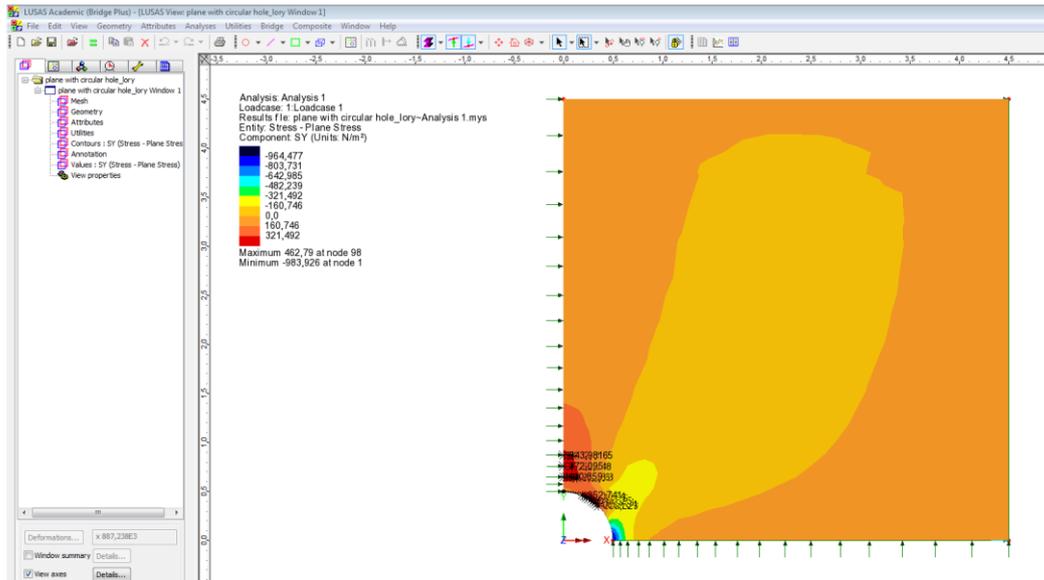
### Different loading conditions:



**RUN!!!**

- Right click, contours, entity plane stress, **SX**

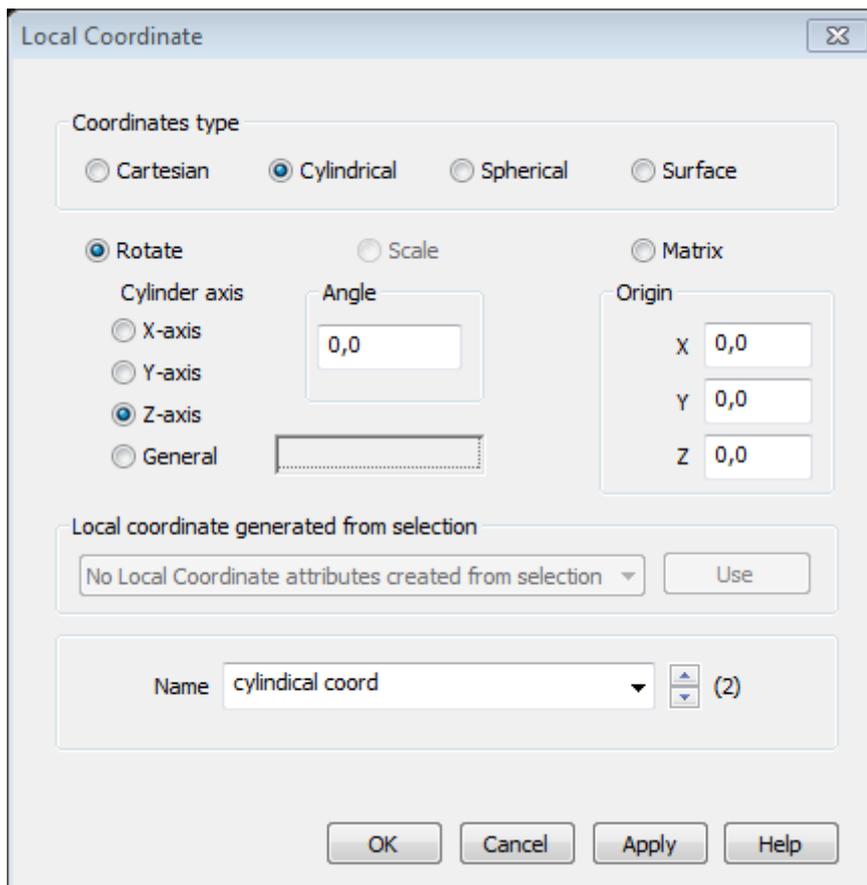




• **How to change RS from Cartesian to cylindrical:**

It changes Radial distances \ Angles \ Z vertical.

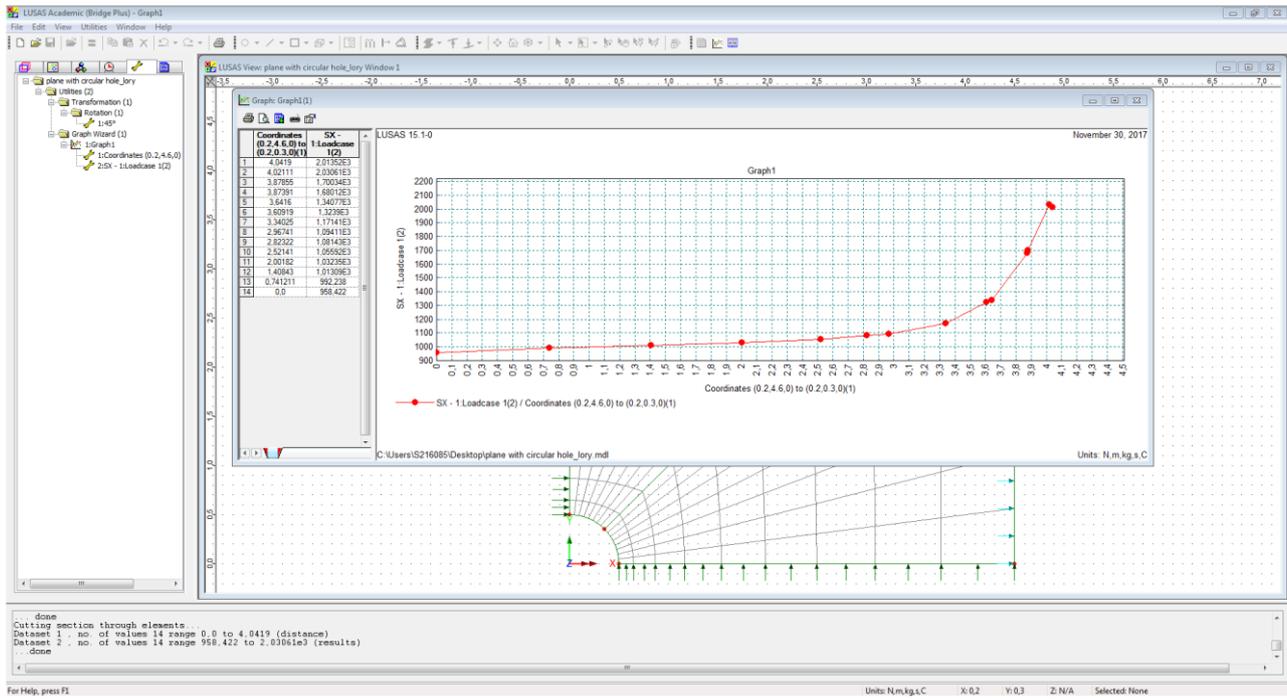
-Attributes, local coordinates, cylindrical,ok (!!!also used for rotations of 45°)



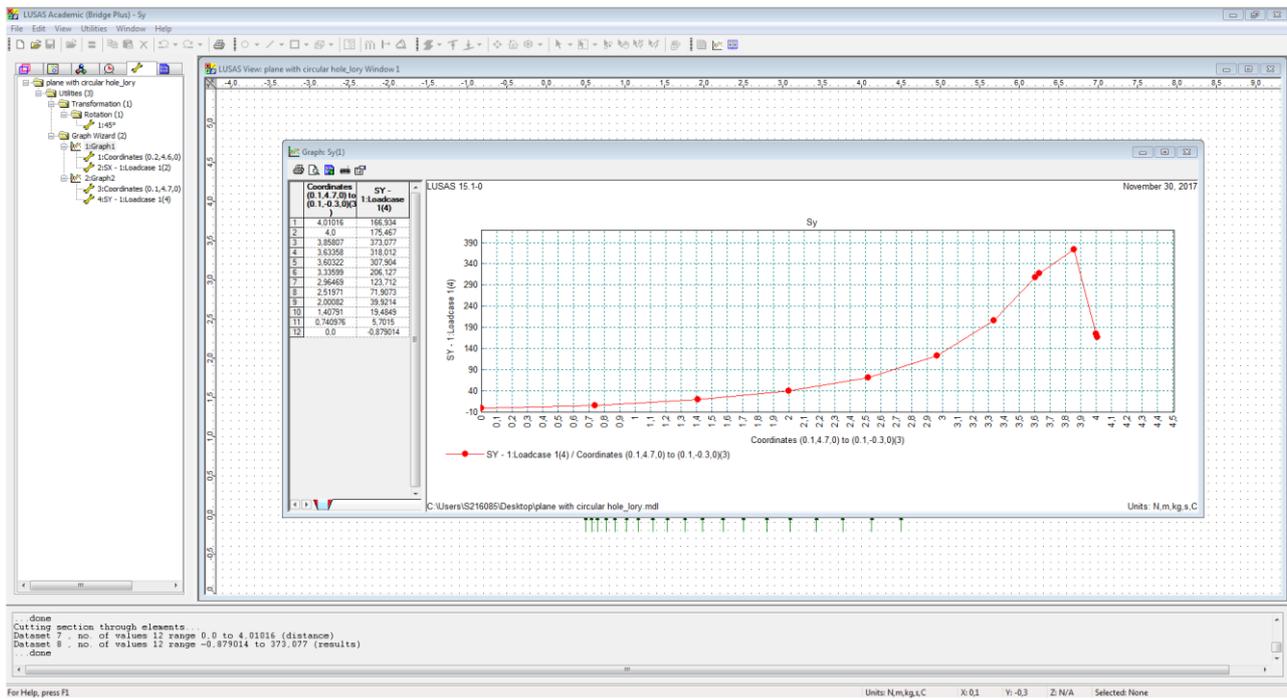
-Assign to all after selecting.

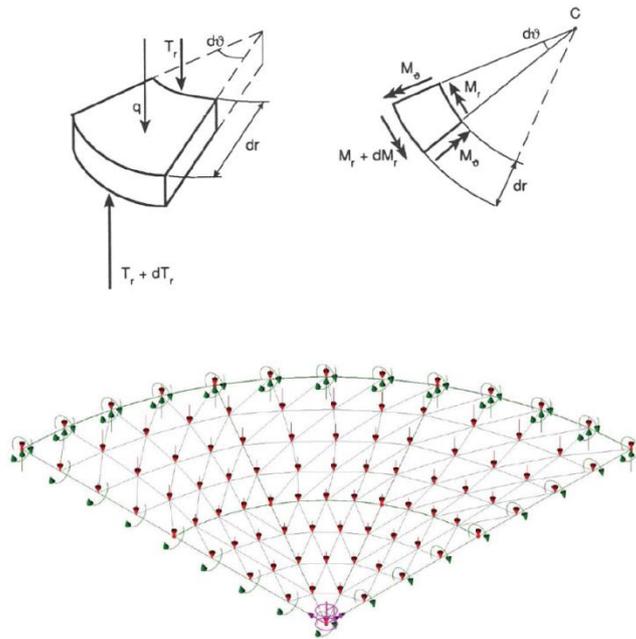
-Contours ,stress-plane, stress

- For SX:



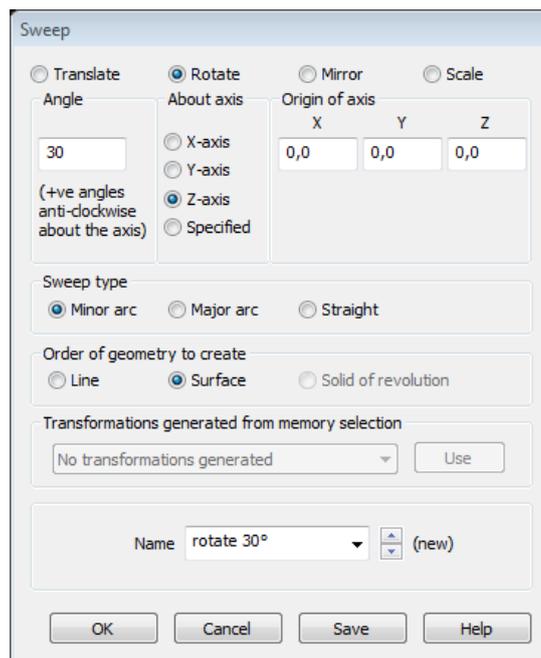
- For SY:





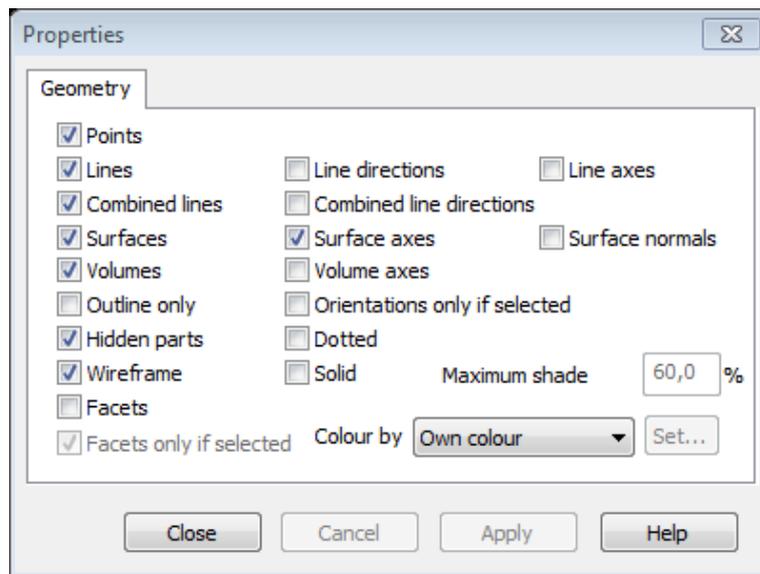
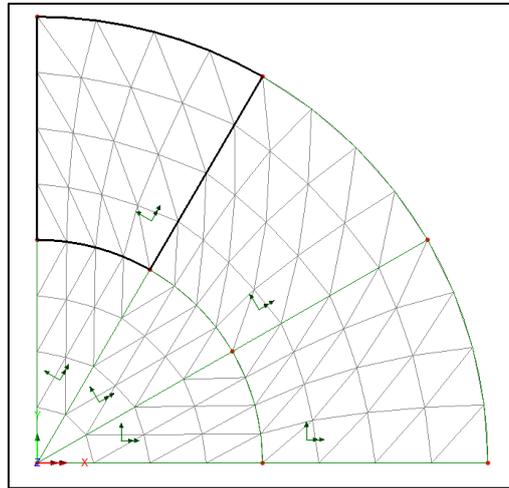
- **How to draw and apply attributes:**

- Do points (0;0), (0.5;0),(1;0) then create 2 line then sweep 3 times the line: by rotating by 30°:
- *Geometry, surface, by sweeping*,



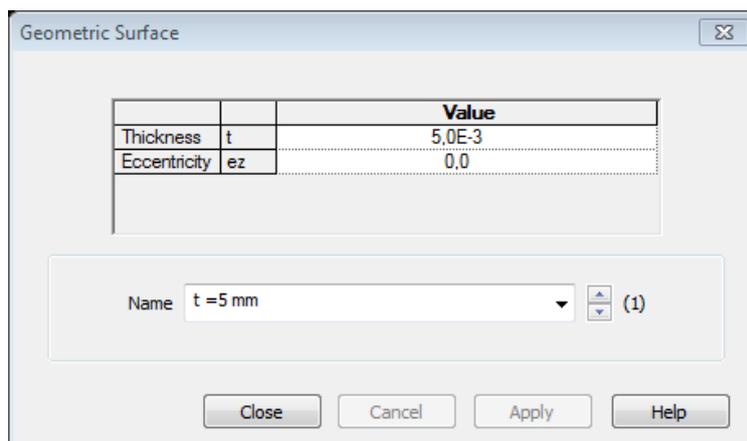
We have created 2 surfaces by sweeping 2 lines.

- Now select the 2 new lines and click : *Geometry, surface, by sweeping* , and select the created “rotate 30°”.



The mesh is not symmetric (no problem).

*-attributes, geometric,surface, assign thickness*



*-attributes, material,isotropic,*

Structural Supports

		Free	Fixed	Spring stiffness
Translation in	X	<input checked="" type="radio"/>	<input type="radio"/>	<input type="text"/>
	Y	<input checked="" type="radio"/>	<input type="radio"/>	<input type="text"/>
	Z	<input checked="" type="radio"/>	<input type="radio"/>	<input type="text"/>
Rotation about	X	<input type="radio"/>	<input checked="" type="radio"/>	<input type="text"/>
	Y	<input checked="" type="radio"/>	<input type="radio"/>	<input type="text"/>
	Z	<input checked="" type="radio"/>	<input type="radio"/>	<input type="text"/>
Hinge rotation		<input checked="" type="radio"/>	<input type="text"/>	
Pore pressure		<input checked="" type="radio"/>	<input type="text"/>	

Spring stiffness distribution

Stiffness

Stiffness/unit length

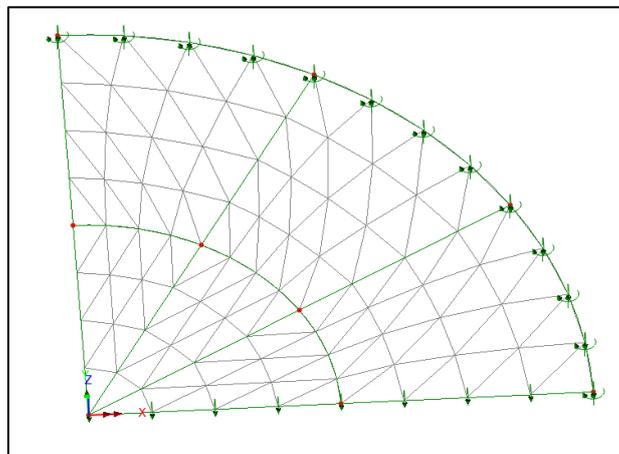
Stiffness/unit area

Lift-off >>

Contact >>

Name:  (new)

OK Cancel Apply Help



Structural Supports

		Free	Fixed	Spring stiffness
Translation in	X	<input checked="" type="radio"/>	<input type="radio"/>	<input type="text"/>
	Y	<input checked="" type="radio"/>	<input type="radio"/>	<input type="text"/>
	Z	<input checked="" type="radio"/>	<input type="radio"/>	<input type="text"/>
Rotation about	X	<input checked="" type="radio"/>	<input type="radio"/>	<input type="text"/>
	Y	<input type="radio"/>	<input checked="" type="radio"/>	<input type="text"/>
	Z	<input checked="" type="radio"/>	<input type="radio"/>	<input type="text"/>
Hinge rotation		<input checked="" type="radio"/>	<input type="text"/>	
Pore pressure		<input checked="" type="radio"/>	<input type="text"/>	

Spring stiffness distribution

Stiffness

Stiffness/unit length

Stiffness/unit area

Lift-off >>

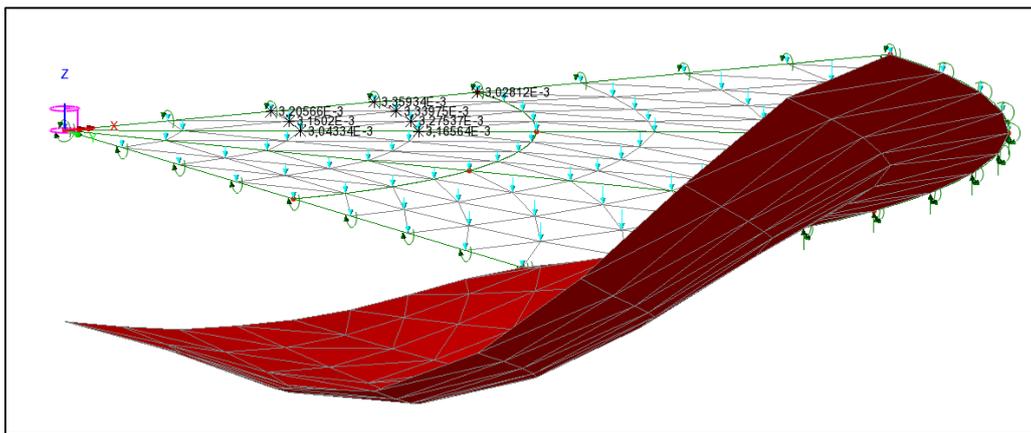
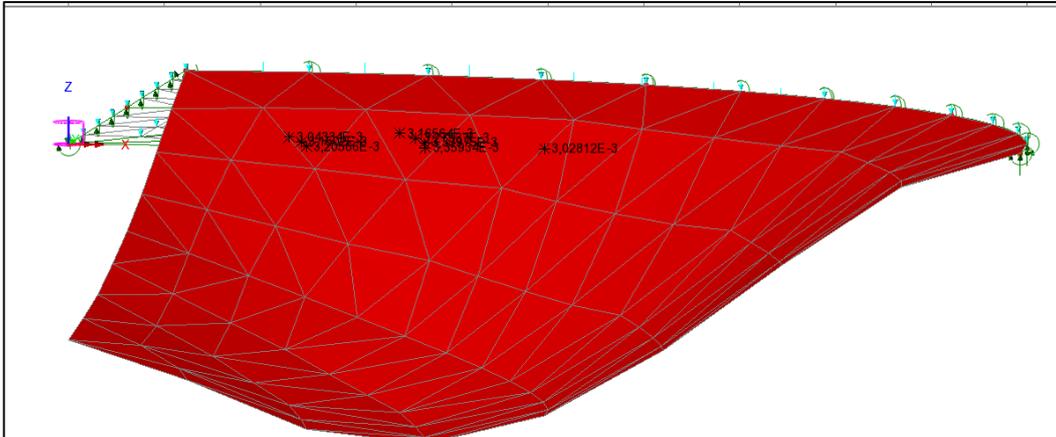
Contact >>

Name:  (new)

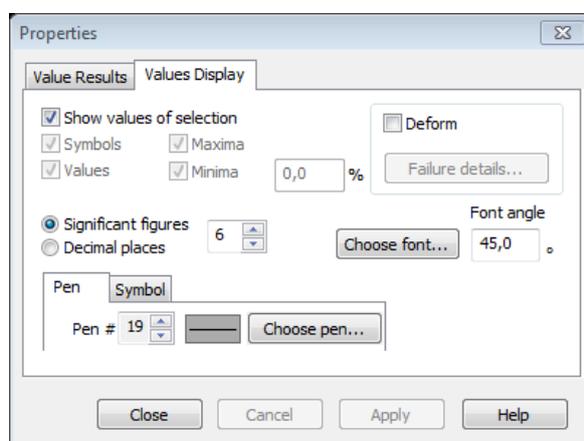
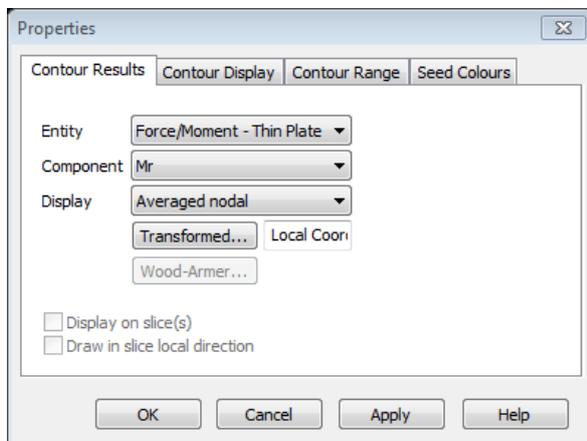
OK Cancel Apply Help

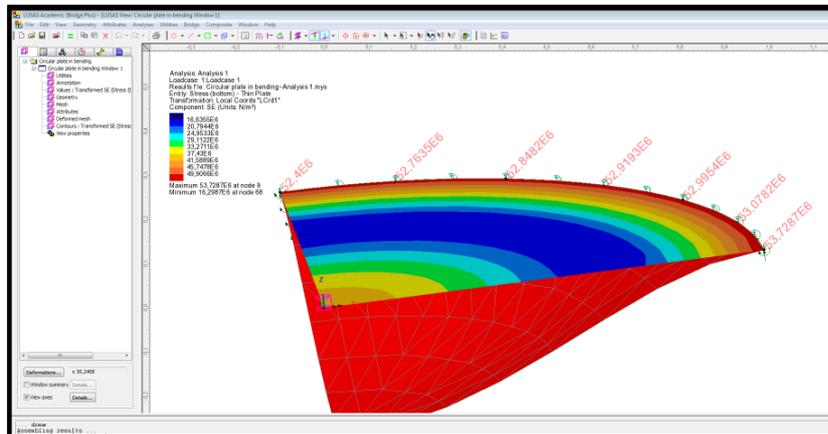
**RUN!!!**

- Deformed mesh:



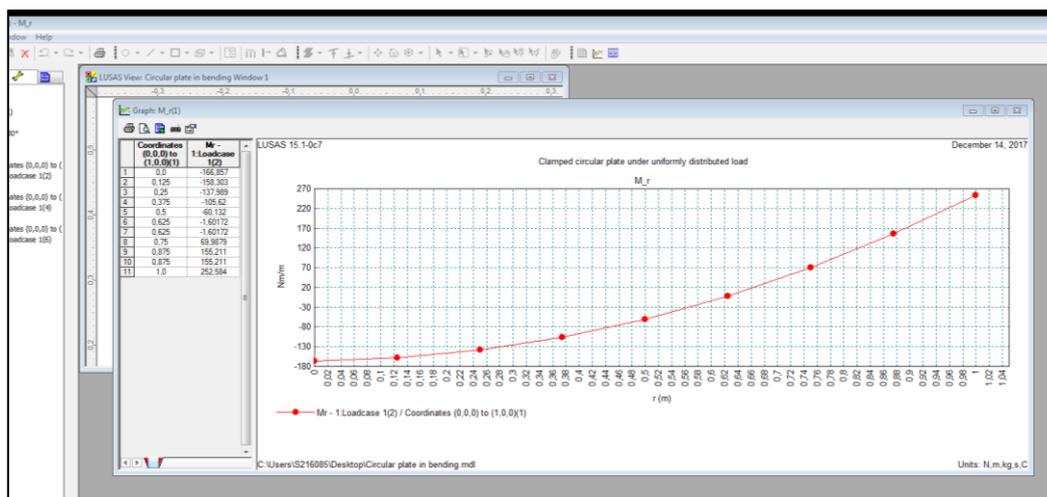
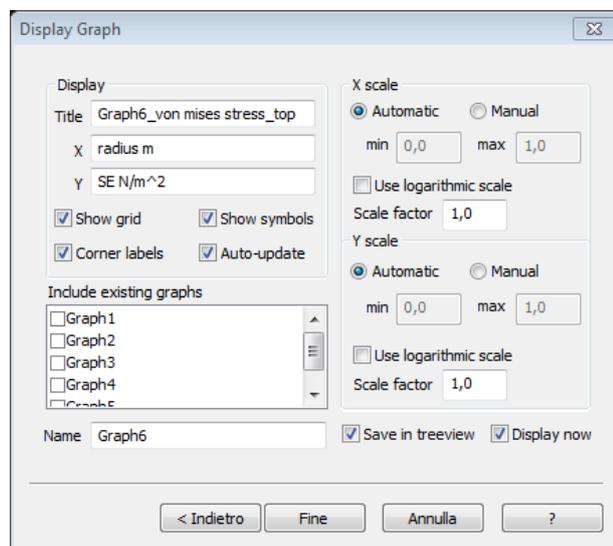
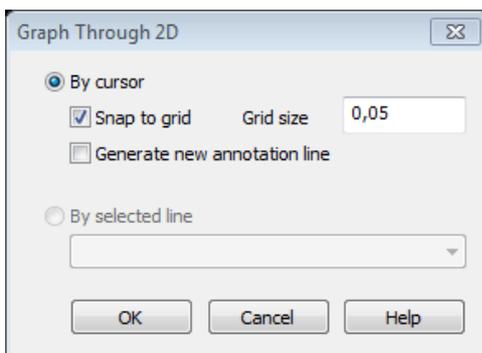
- Mr component:





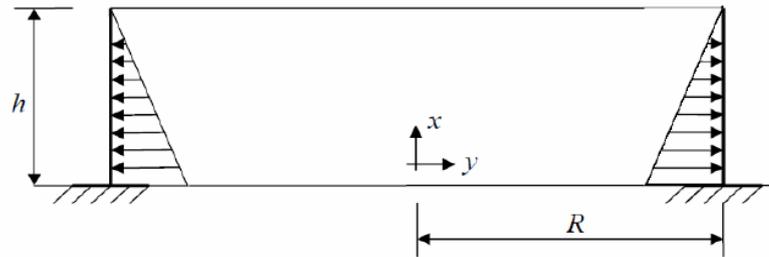
• **How to draw diagrams:**

- Utilities, graph through 2D, grid = 0.05



- to see the diagrams, click on utilities (in the right panel-yellow key).

# Cylindrical tank under hydrostatic pressure



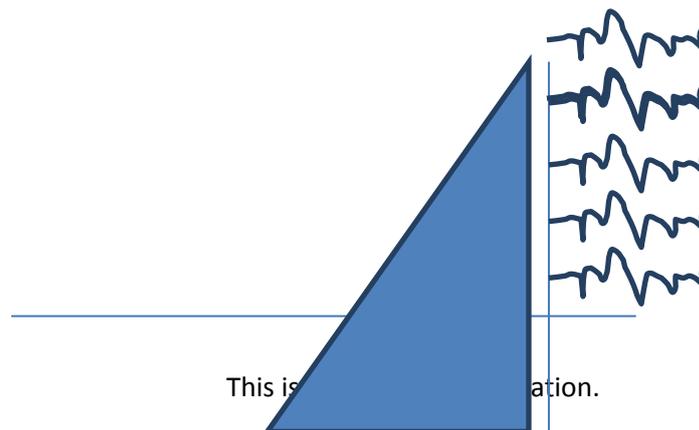
Vertical section

$R = 3 \text{ m}$ ,  $h = 1 \text{ m}$ , thickness  $t = 4\text{E-}4 \text{ m}$

Steel with:  $E = 2.1\text{E}11 \text{ Pa}$ ,  $\nu = 0.3$

Water:  $\gamma = 9.81\text{E}3 \text{ N/m}^3$

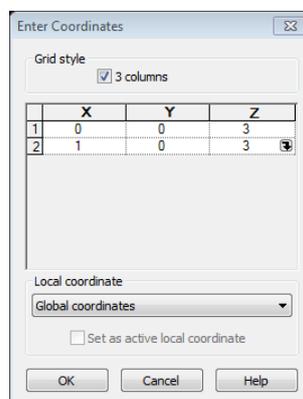
1. Model one fourth of tank using “Thin Shell” elements (2D).
2. Model the tank by a line using “Axisymmetric Thin Shell” elements.

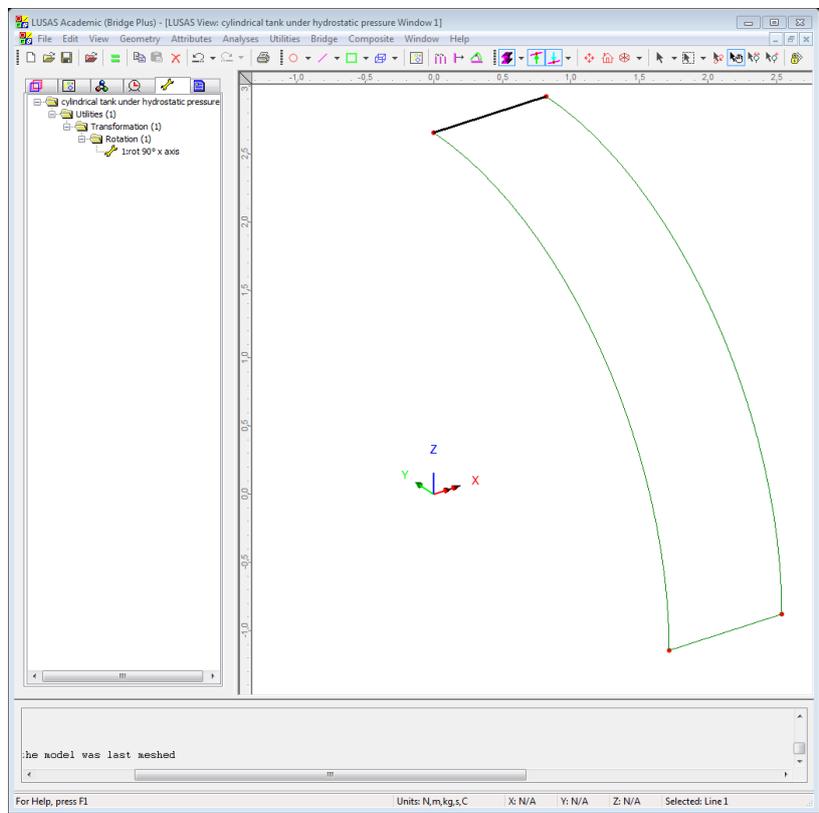
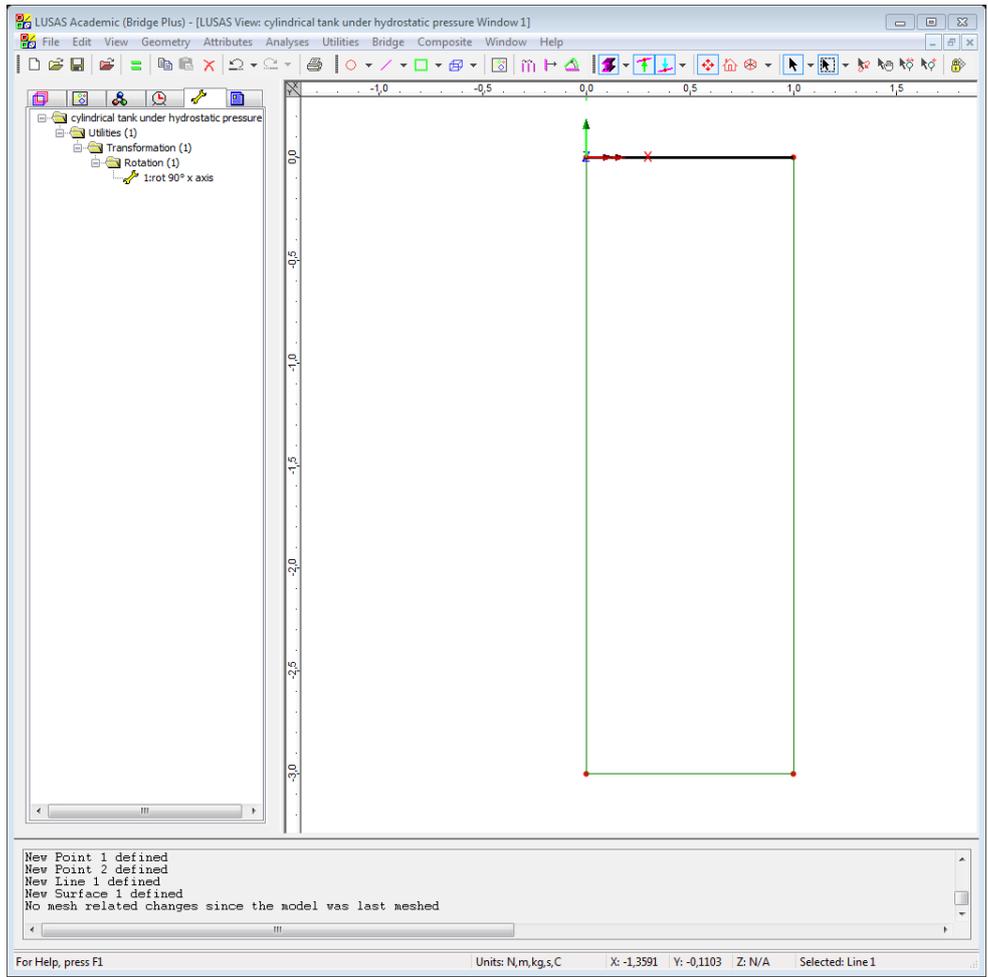


Then we can use the thin shell elements or the axis-symmetric(symmetry around vertical axis)

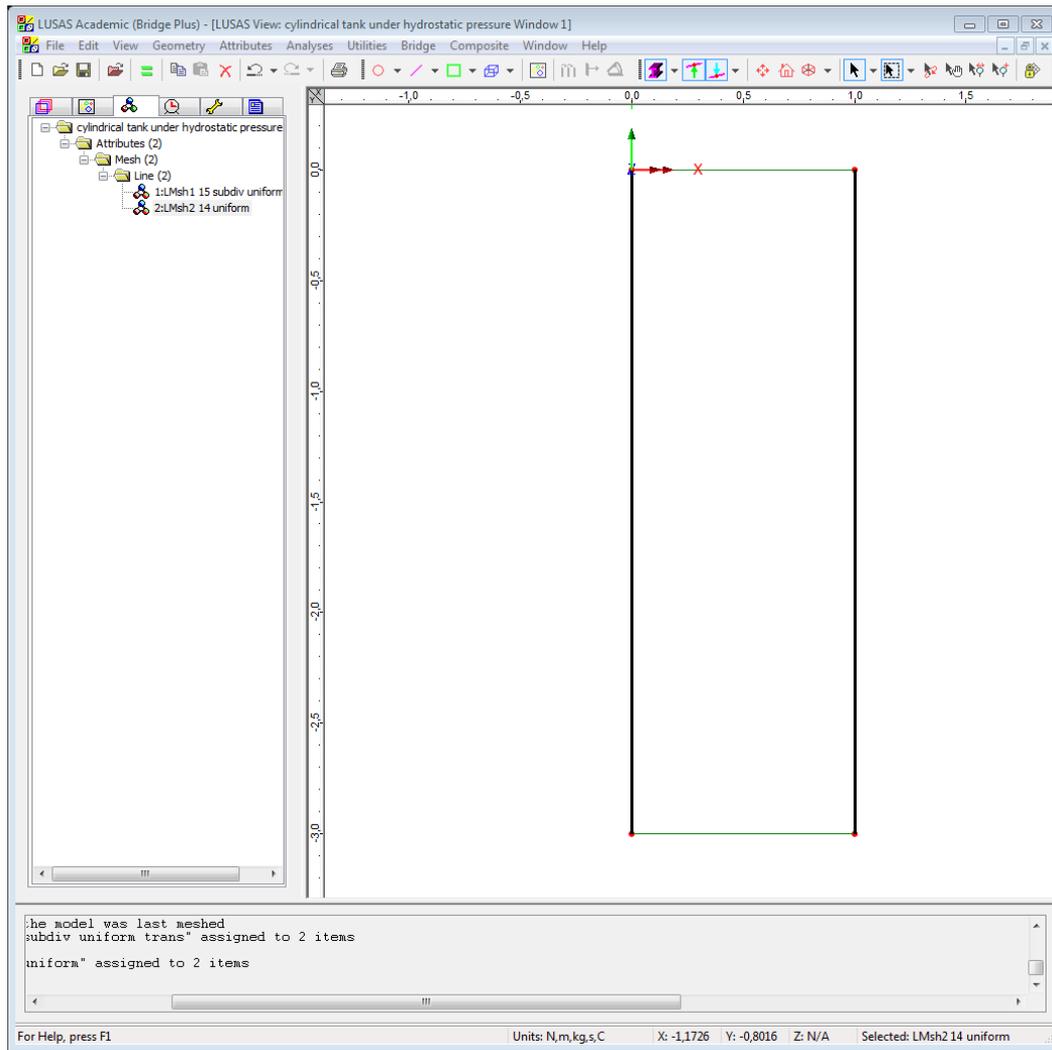
- **How to draw the tank:**

- New point: (0,0,3);(1,0,3) and connect by a line.

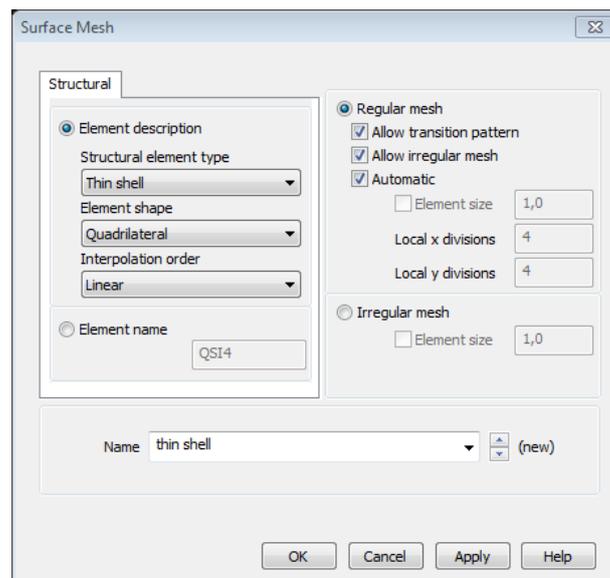


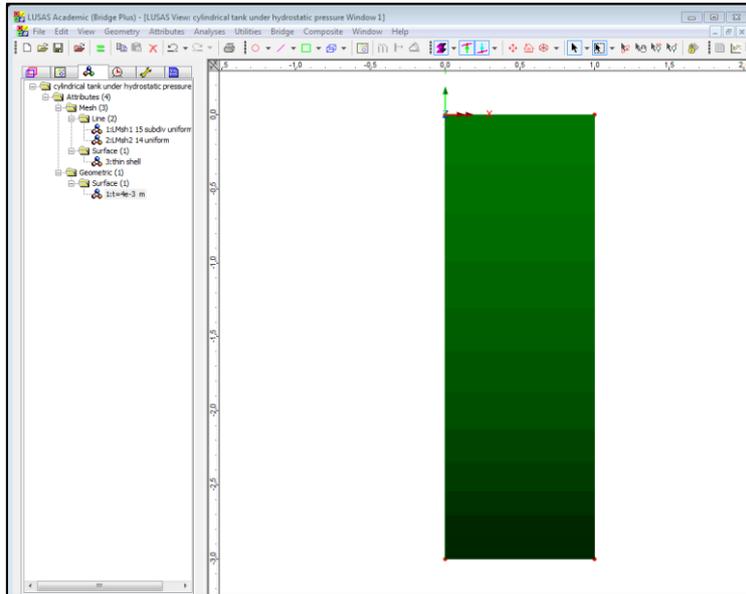


- Assign to the short side the uniform transition and to the circular (long) side the uniform spacing:

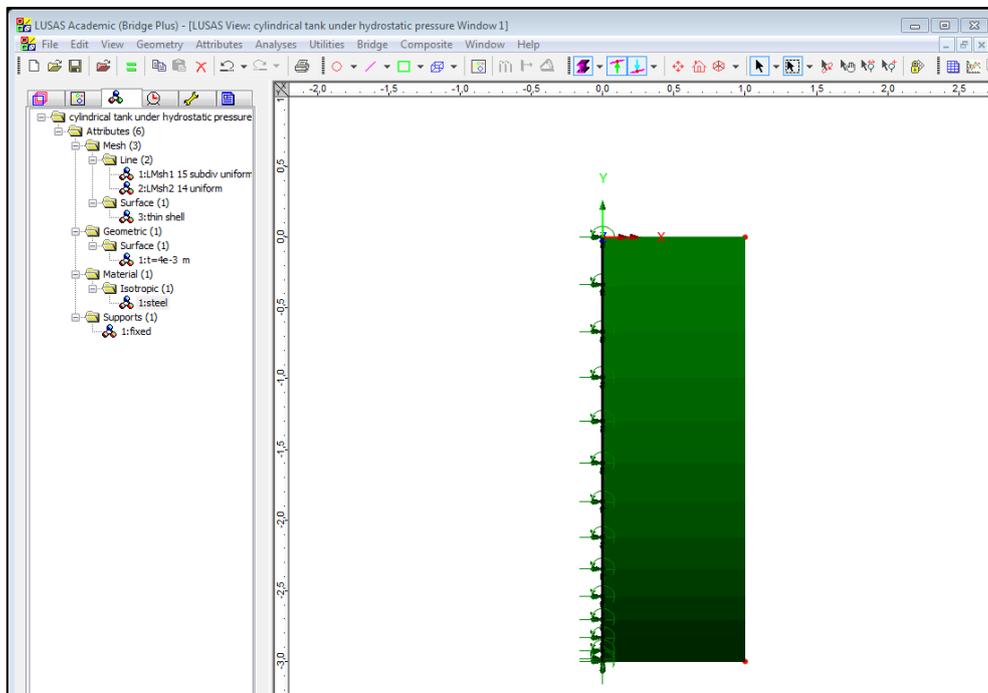


- *Attrib,mesh, surface,thin shell* (Kirchoff theory) = no shear deformation:

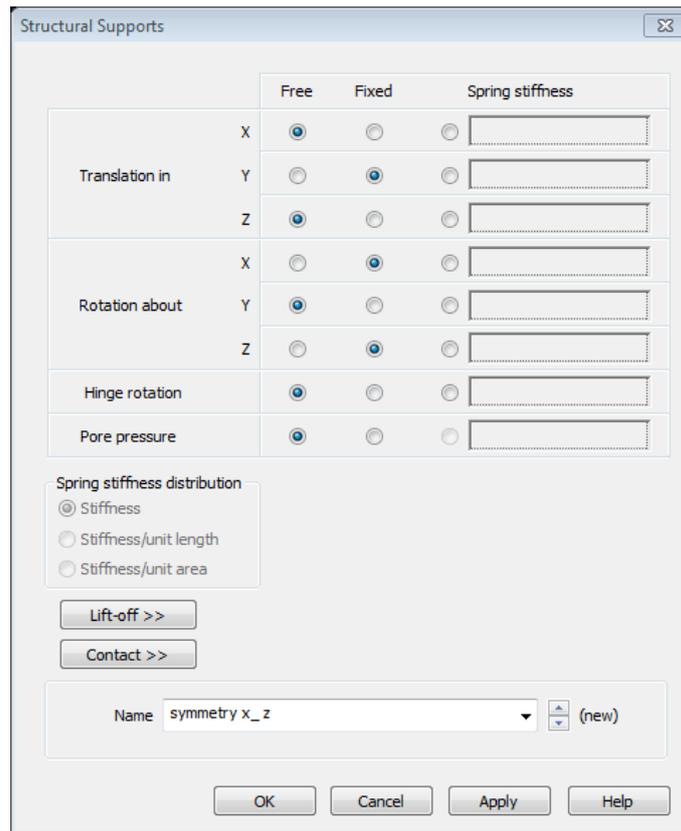




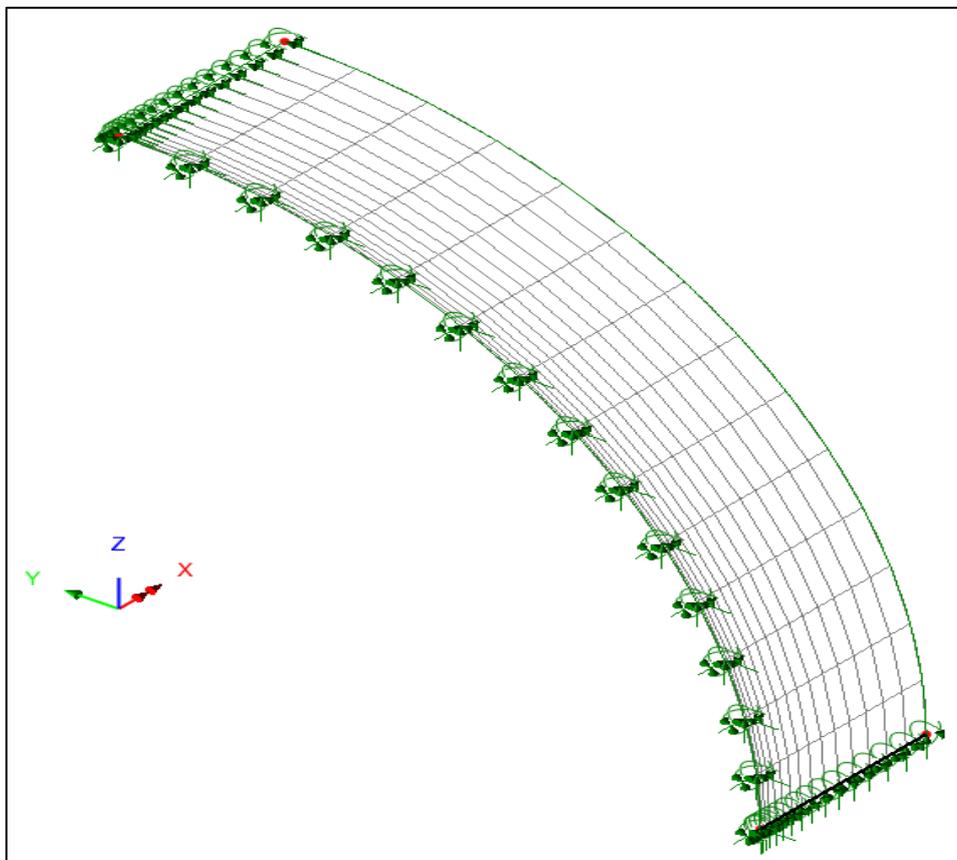
- attributes, supports,...



- Symmetry x-z: attributes, supports,..



- Assign the loads: !!!pay attention to which line:



Local Distributed ✕

Distribution type

Line  Area

Component	Value
x Direction	
y Direction	1,0*triangular load
z Direction	

Name  (new)

< Back   Next >   Finish   Cancel   Apply   Help

Local Distributed ✕

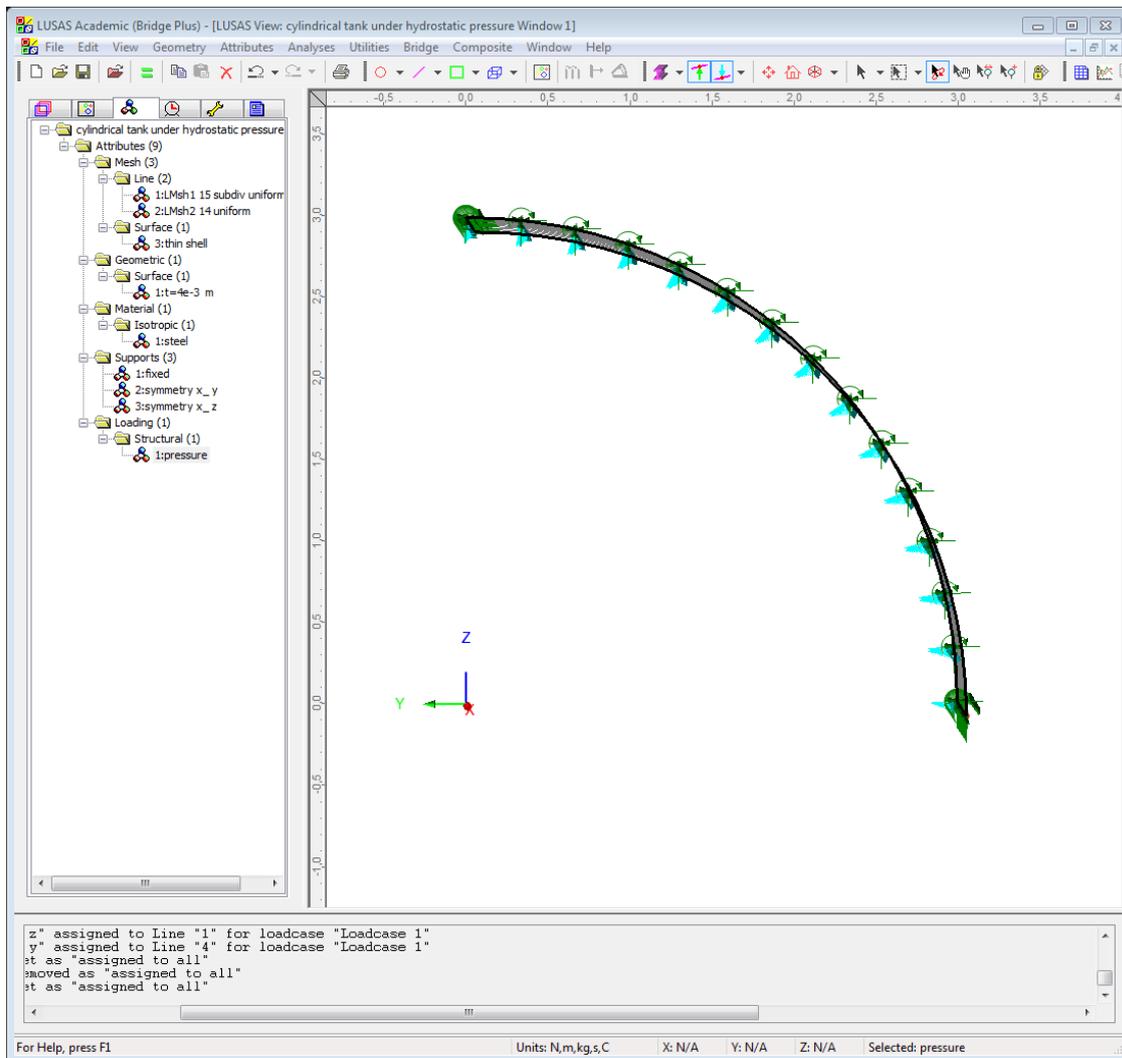
Distribution type

Line  Area

Component	Value
x Direction	
y Direction	1,0*triangular load
z Direction	

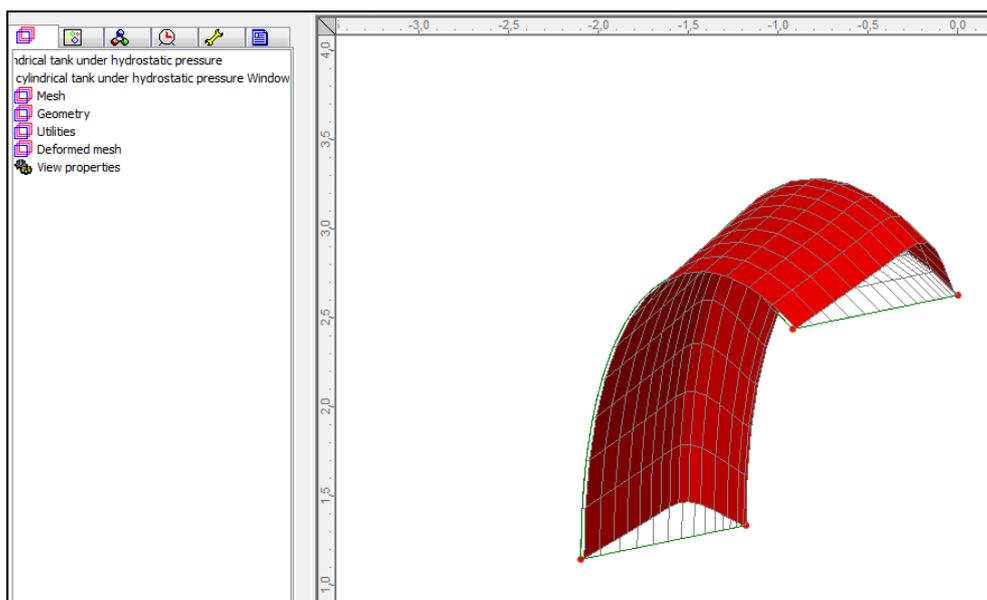
Name  (new)

< Back   Next >   Finish   Cancel   Apply   Help



**RUN!!!**

Deformed mesh:



**Enter Coordinates**

Grid style  
 3 columns

	X	Y	Z
1	1	0	0
2	2	0	0

Local coordinate  
 Global coordinates

Set as active local coordinate

OK Cancel Help

**Line Mesh**

Structural

Element description

Structural element type  
 Thin beam

Number of dimensions  
 2 dimensional

Interpolation order  
 Quadratic

Use default spacing

Number of divisions  
 20

Spacing...

Element length  
 0,0

End conditions...

Name 20 subdiv (new)

OK Cancel Apply Help

**Geometric Line**

Usage  
 2D Thin Beam

Definition  
 From Library  
 Enter Properties

Rotation about centroid 0

100%

	Value
Cross sectional area (A)	1e-4
Second moment of area about z axis (Izz)	8,333e-10
Eccentricity in y direction (ey)	0,0

Visualise... Tapering >> Section details...

Name 10 mm x 10 mm (new)

OK Cancel Apply Help

**Isotropic**

Plastic  Creep  Damage  Shrinkage  Viscous  Two phase

Elastic

Dynamic properties  
 Thermal expansion

	Value
Young's modulus	2,1e11
Poisson's ratio	0,3
Mass density	7850

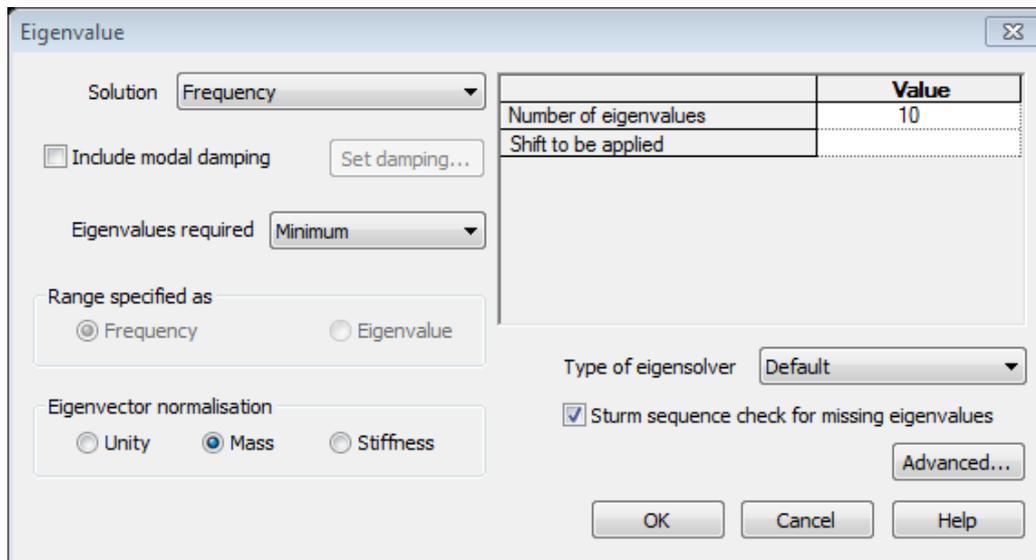
Name Iso1 (new)

OK Cancel Apply Help

!!!  $N$  = nr of elements (coming from the beam subdivisions). We can ask up to  $(N-1)$  eigenvalues .

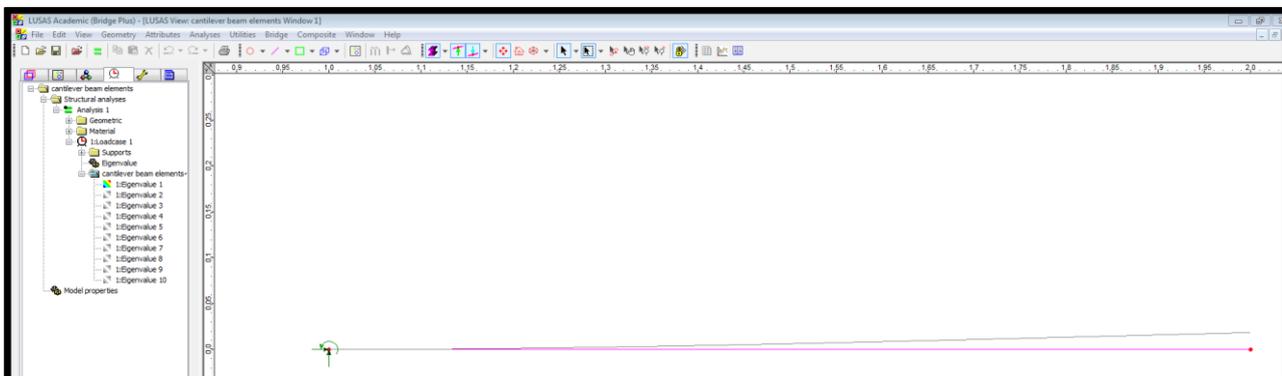
We divided into 20 but we ask 10 eigenvalues (only the first ones will be well described very well)

“Shift” means that the start frequency is shifted: ex. if “shift to be applied=30 Hz”, then it starts from 30 Hz.

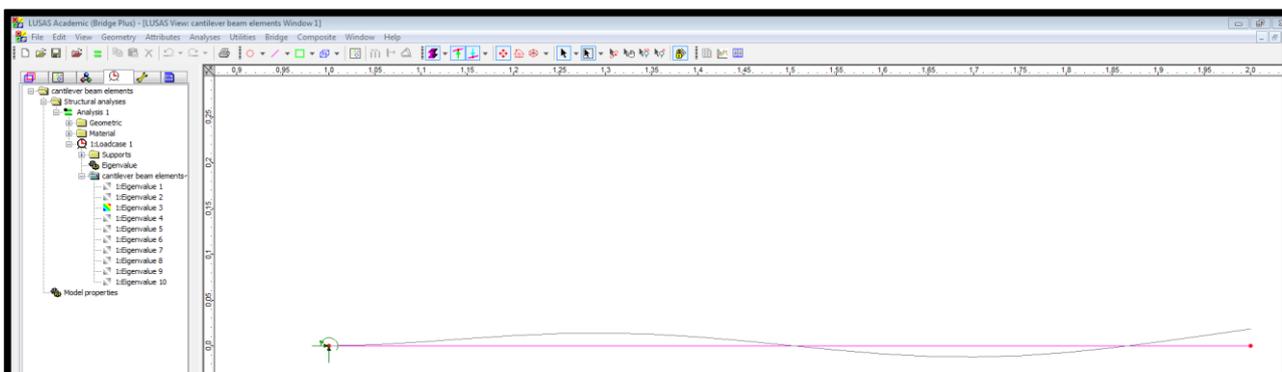


**RUN!!!**

- Eigenvalue 1:

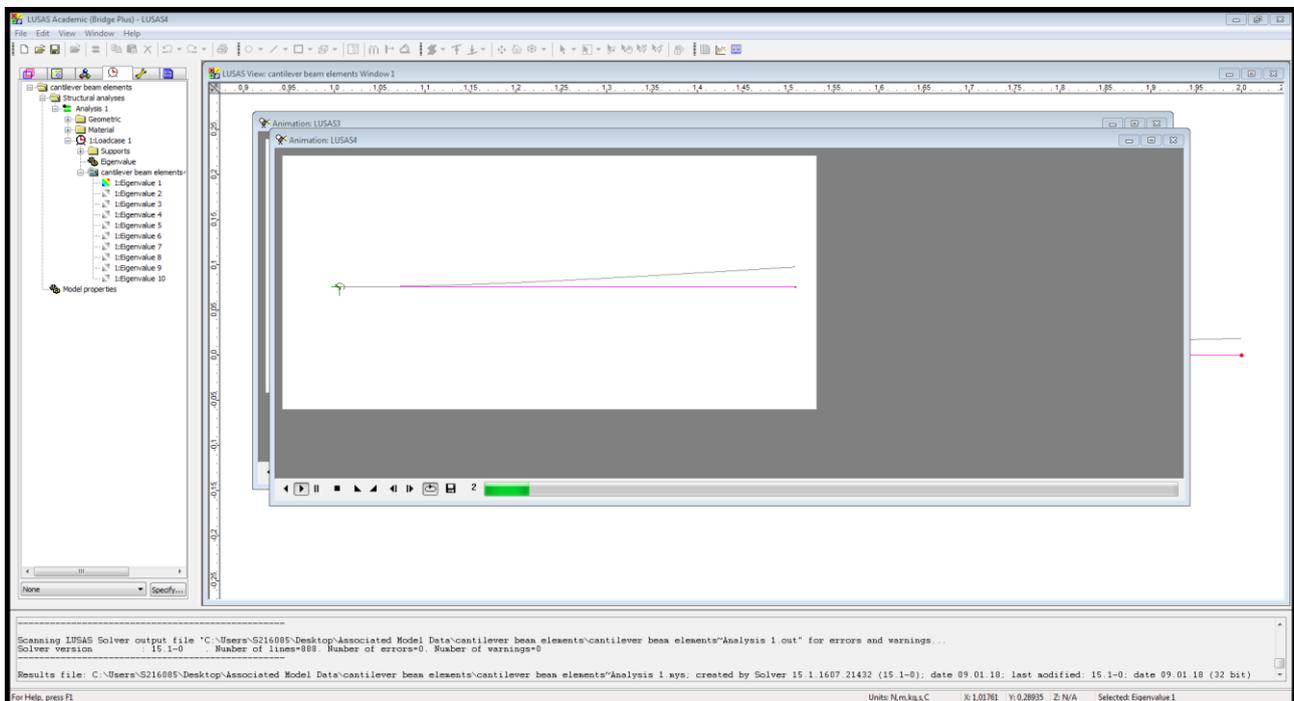
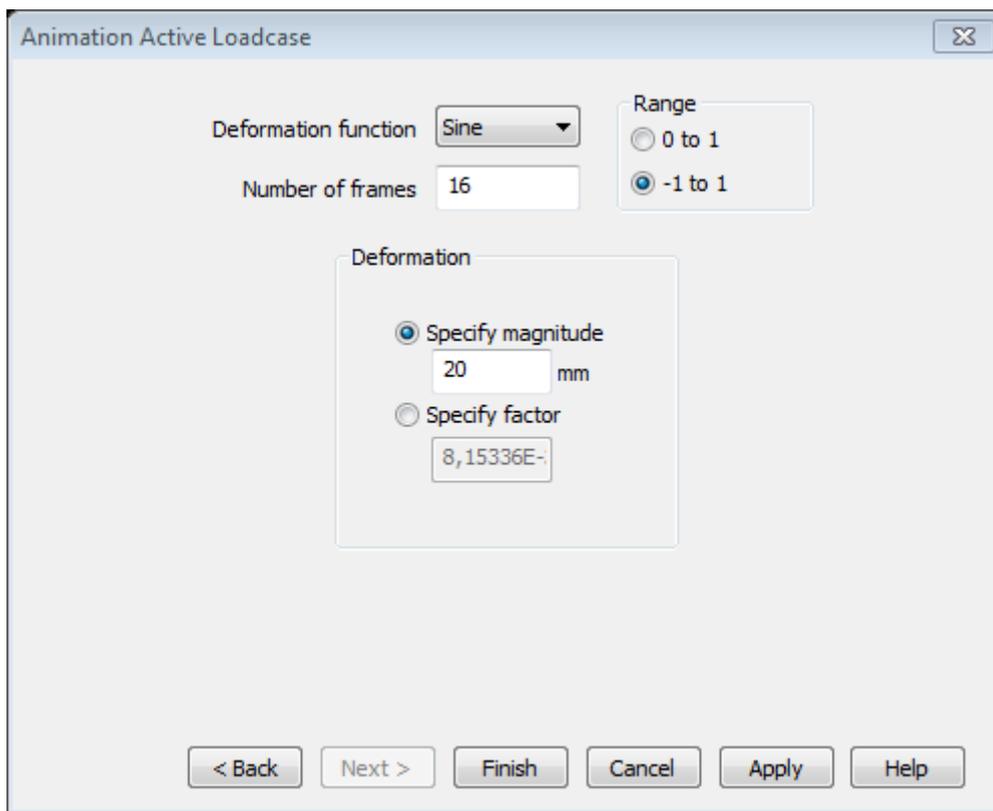


- Eigenvalue 3:



- **How to see the video of the movement:**

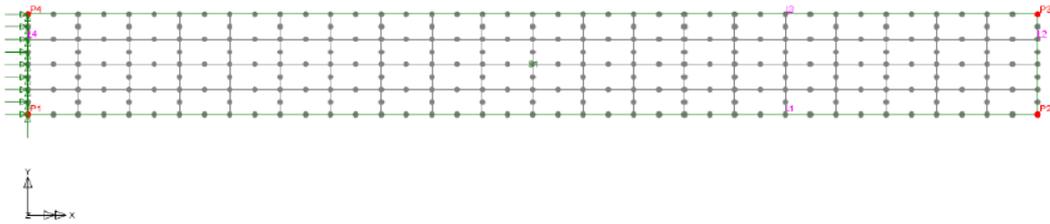
- Utility, animation wizard, next, nr of frames 16, from -1 to 1 (complete cycle)



# Dynamics: Plane stress elements 2D

Same beam as in the previous example with cross section 10 cm x 10 cm.

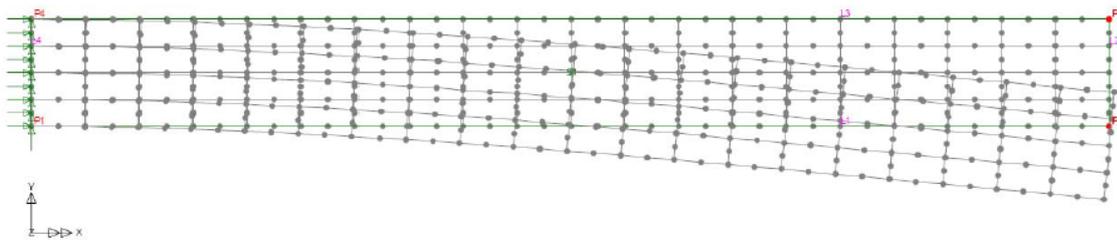
For the solution, use quadrilateral *plane stress* elements, with *quadratic interpolation* and a subdivision in 20 x 4 elements.



## NUMERICAL RESULTS:

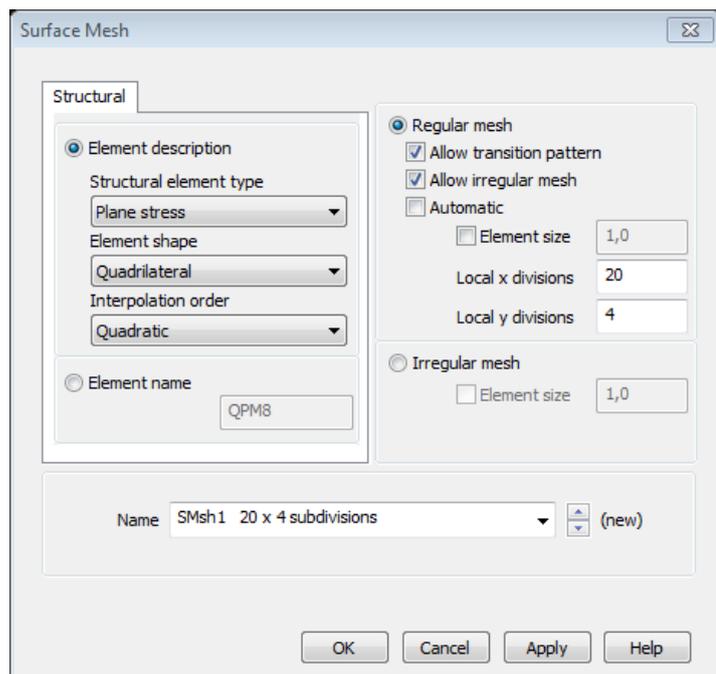
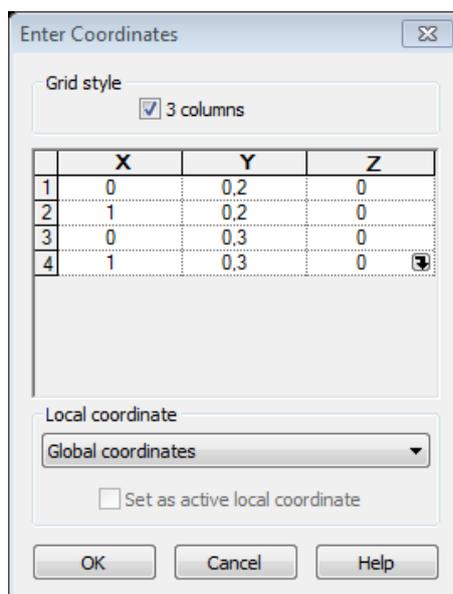
- Mode shapes (and corresponding stress contours):

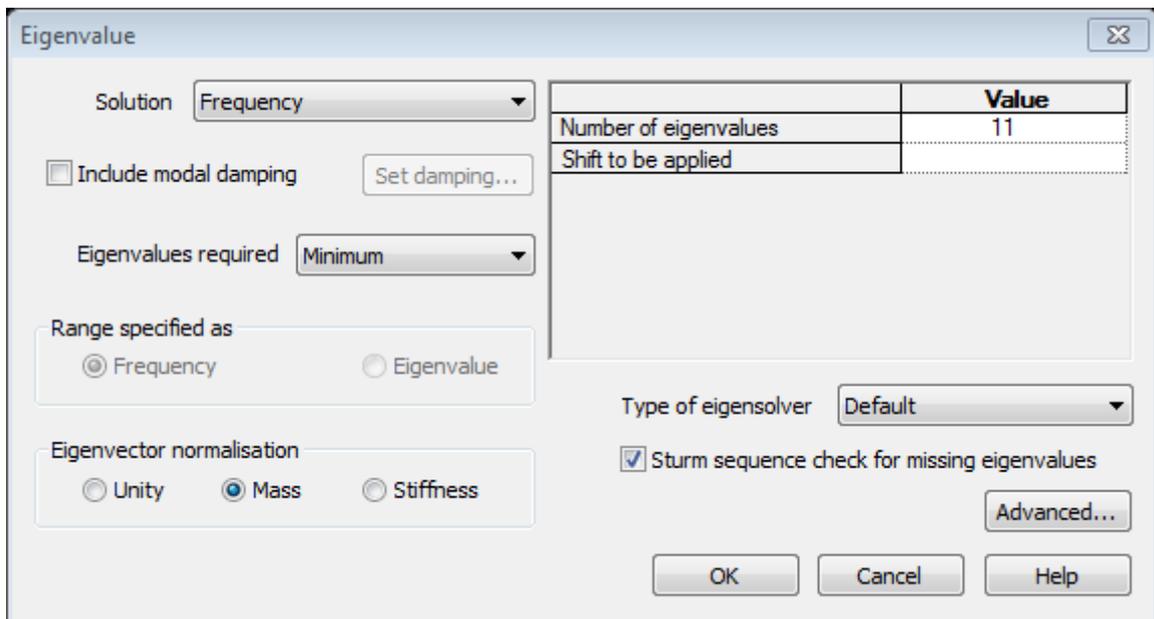
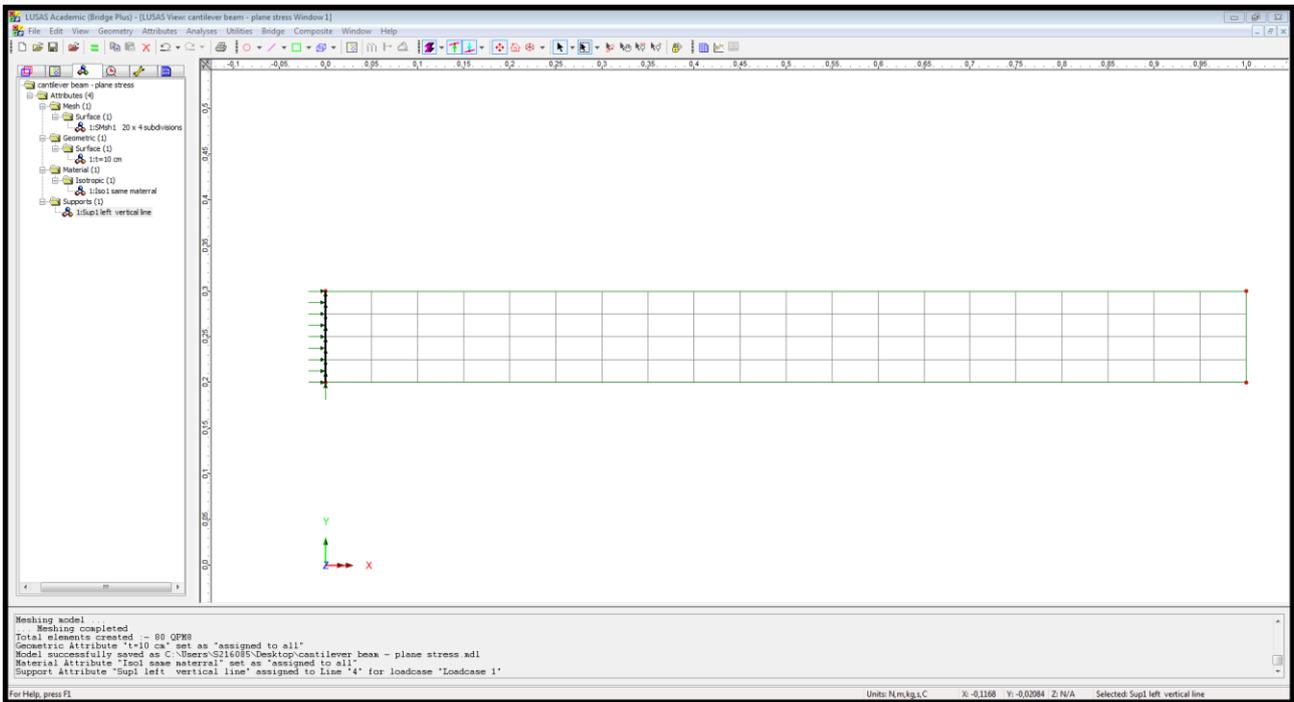
Mode 1



- **Solution:**

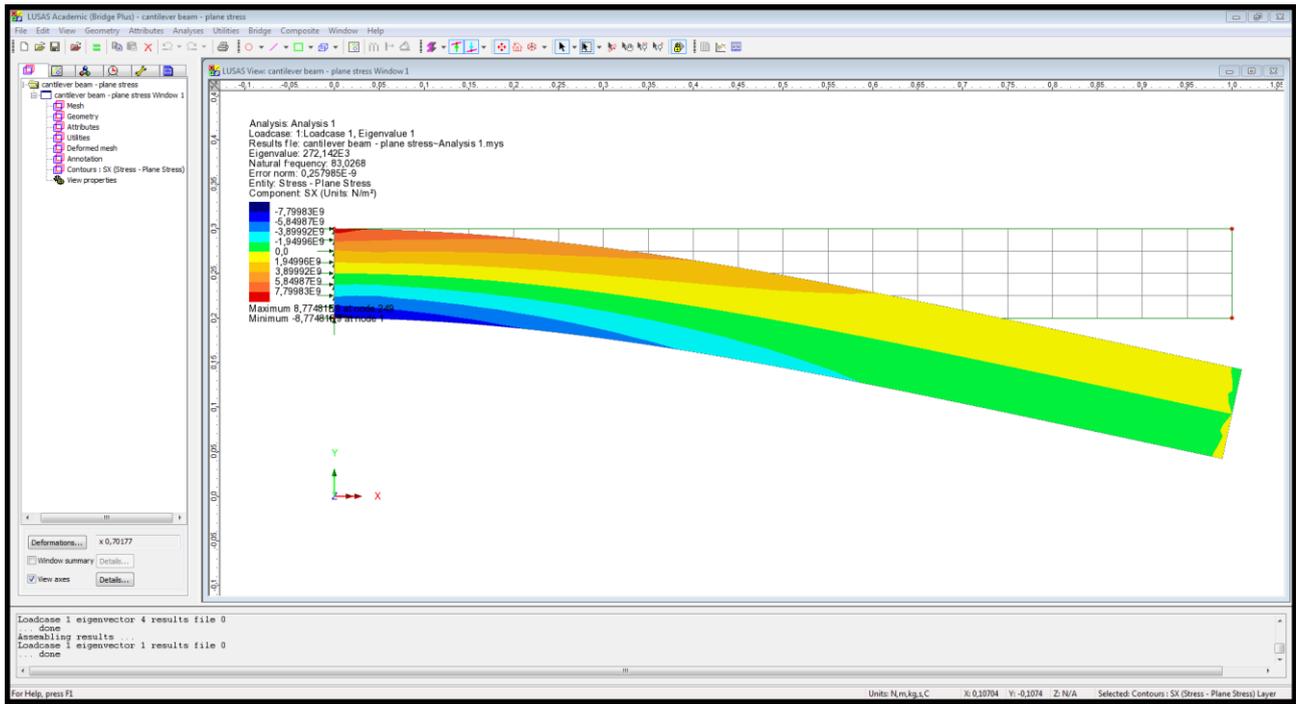
-New points and surface



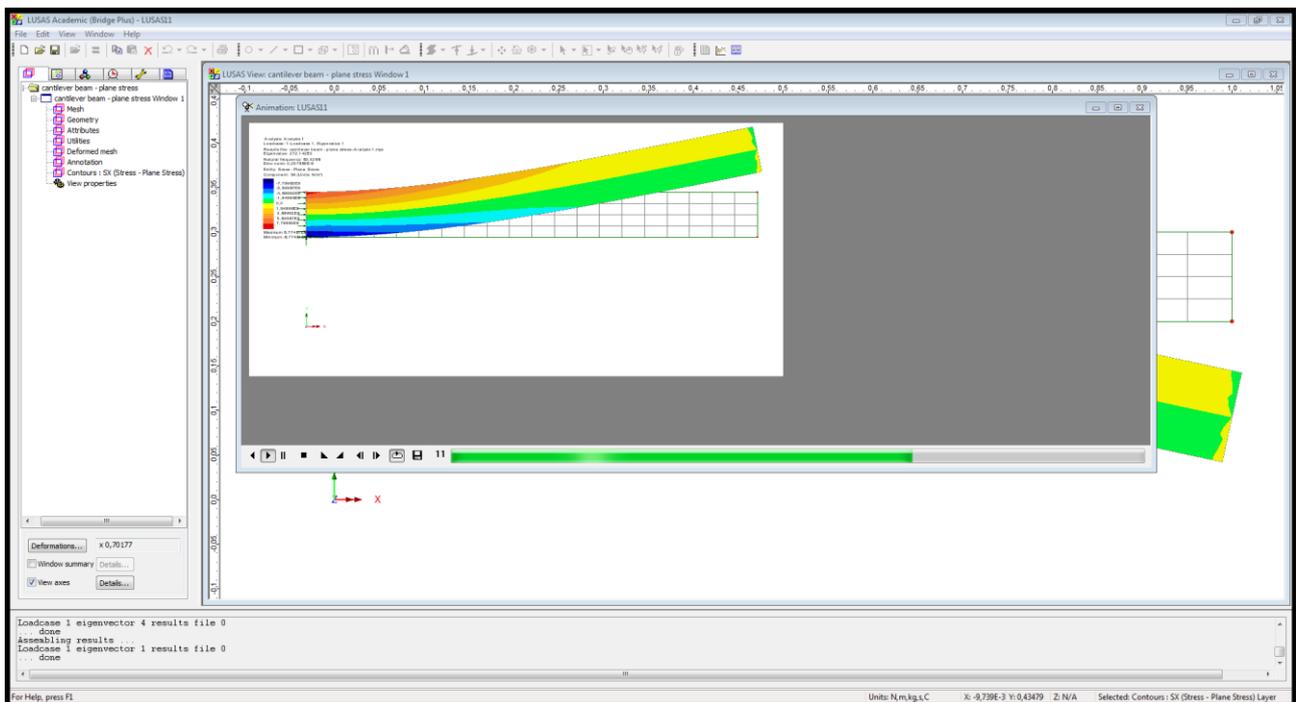


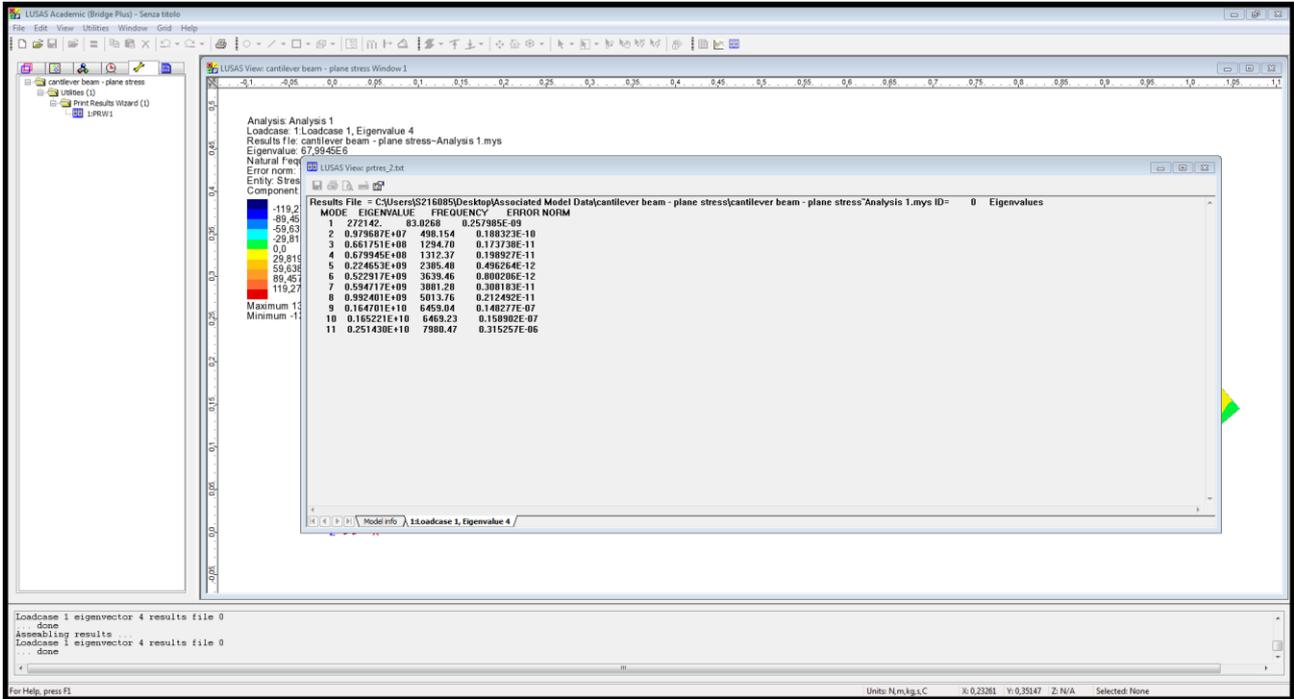
**RUN!!!**

Eigen 1

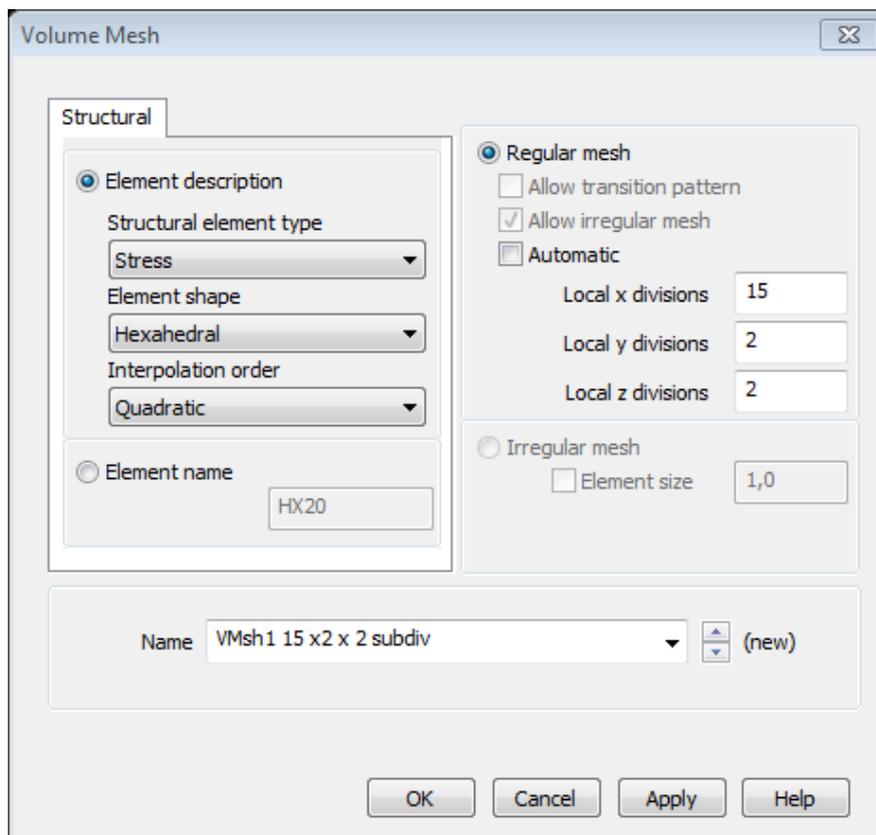
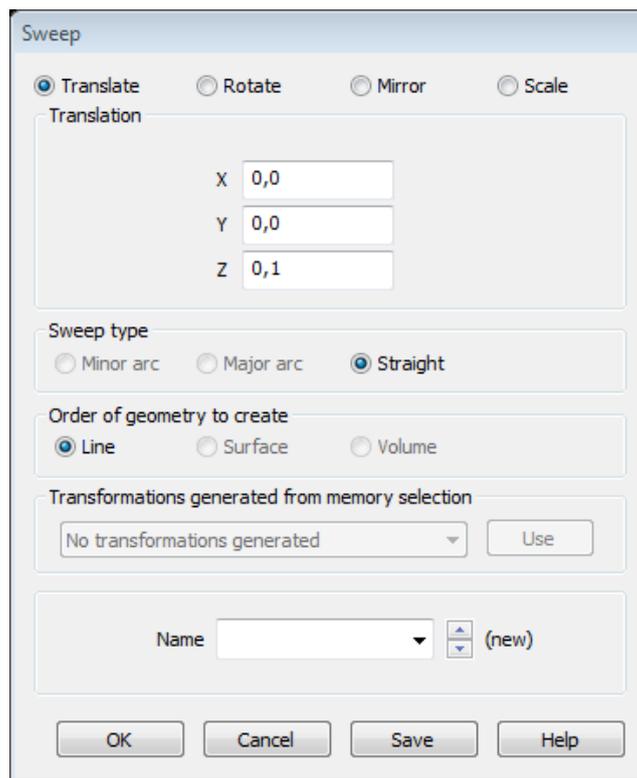


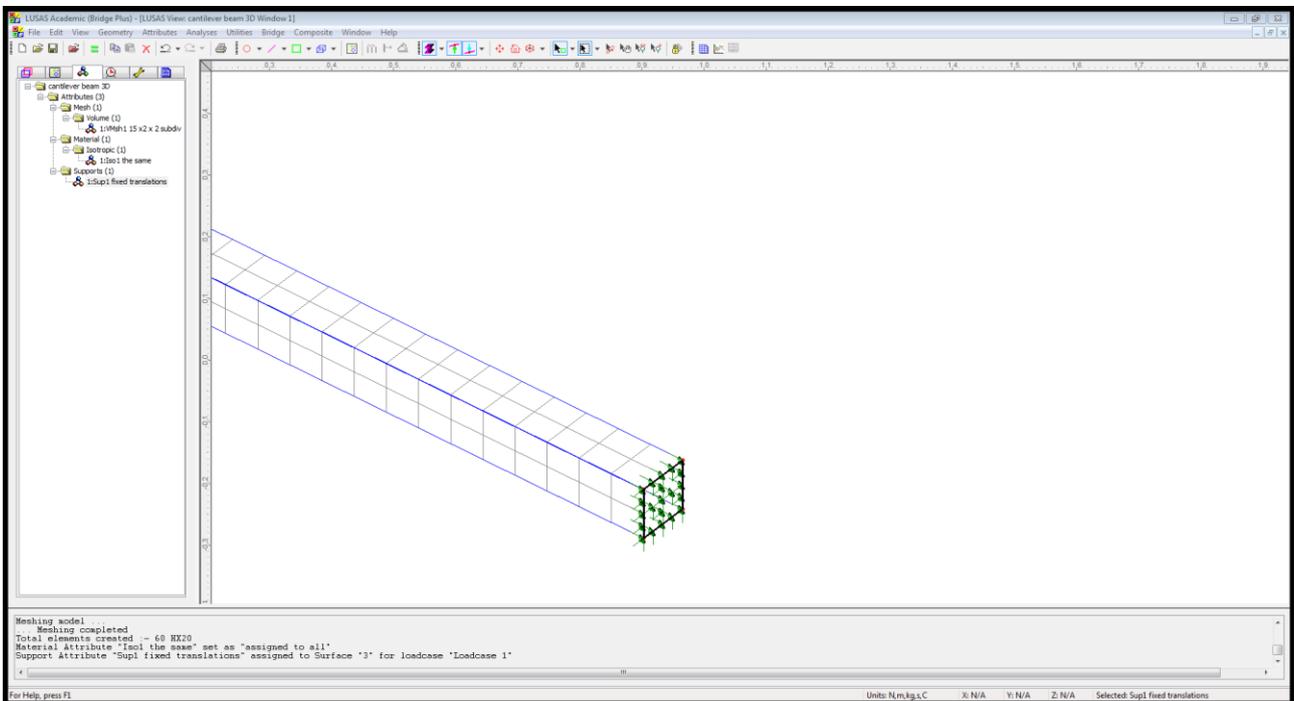
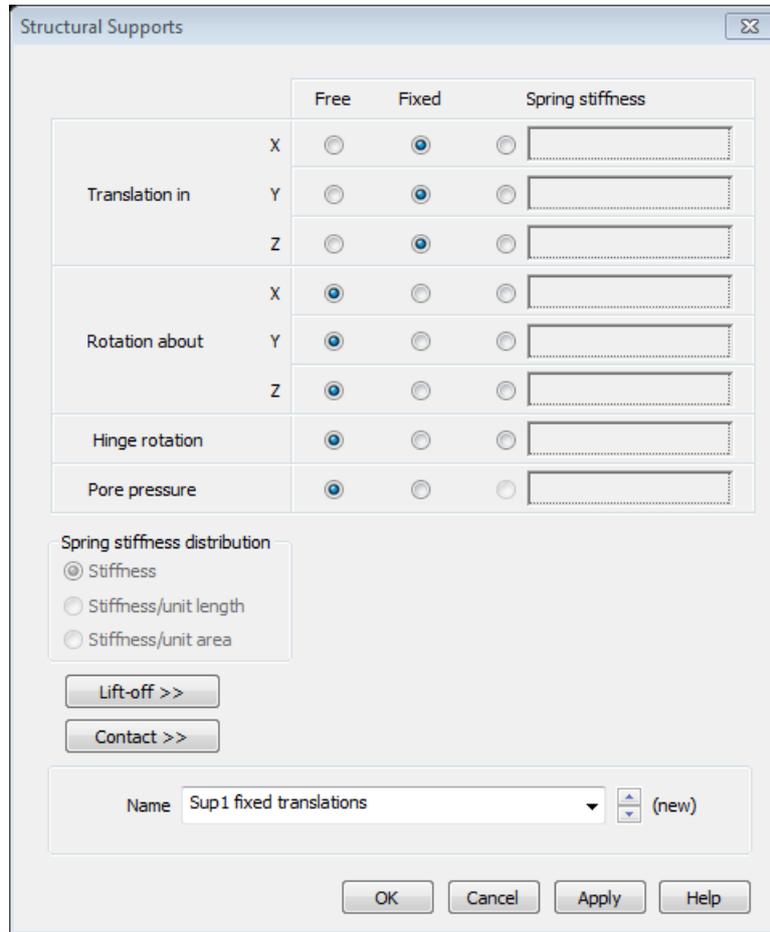
Utilities, animation wizard,





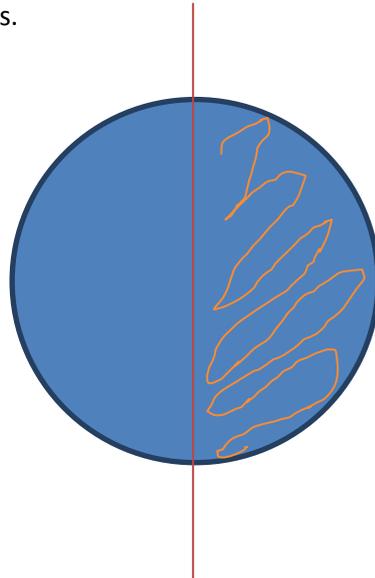
- **How to draw the solid:**
  - Start from the previous cantilever beam and sweep it:



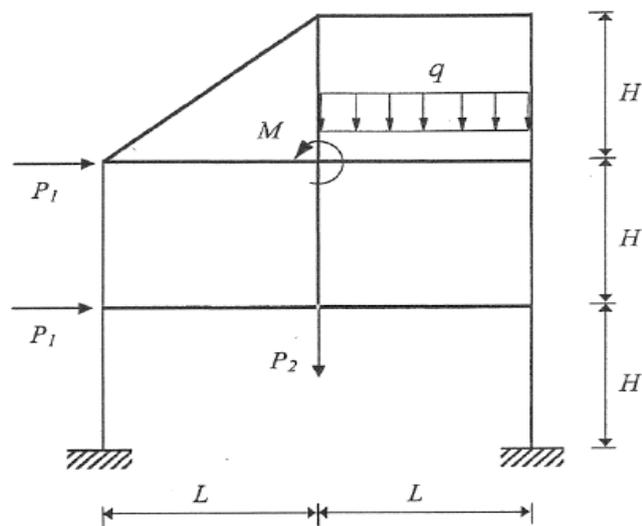


## Dynamics: Free vibrations of frames

!!! For vibration (in dynamics) we need at least half plate (not only one quarter as we did before) so we don't lose the anti-symmetric modes.



(Same structure proposed for running a static analysis in LAIB 2017\_10\_26)



$$L = 5 \text{ m} \quad H = 3 \text{ m}$$

$$P_1 = 10 \text{ kN} \quad P_2 = 20 \text{ kN} \quad M = 10 \text{ kNm} \quad q = 10 \text{ kN/m}$$

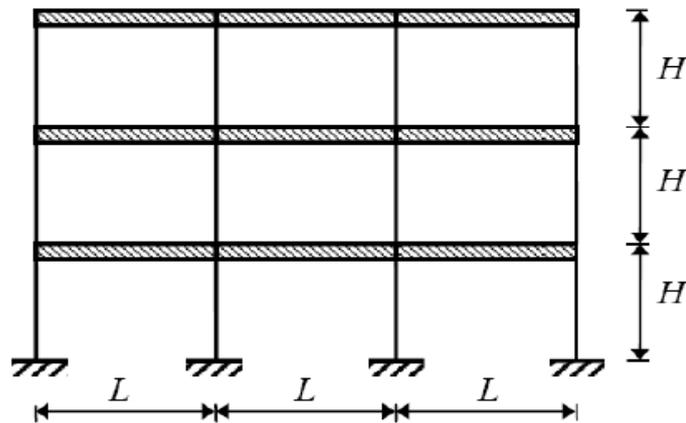
$$\text{Concrete: } E = 2.9 \cdot 10^{10} \text{ N/m}^2 \quad \nu = 0.2 \quad \rho = 2500 \text{ kg/m}^3$$

$$\text{First floor columns: } b \times h = 50 \times 50 \text{ cm}^2 \quad (A = 0.25 \text{ m}^2; I_{zz} = 5.20833 \cdot 10^{-3} \text{ m}^4)$$

$$\text{Second floor columns: } b \times h = 40 \times 40 \text{ cm}^2 \quad (A = 0.16 \text{ m}^2; I_{zz} = 2.1333 \cdot 10^{-3} \text{ m}^4)$$

$$\text{Third floor columns: } b \times h = 30 \times 30 \text{ cm}^2 \quad (A = 0.09 \text{ m}^2; I_{zz} = 0.6750 \cdot 10^{-3} \text{ m}^4)$$

## Dynamics: Plane frame shear type



Columns and beams have rectangular cross-section (width  $b$ , depth  $h$ ).

$$L = 5 \text{ m} \quad H = 3 \text{ m}$$

$$E = 2.9 \cdot 10^{10} \text{ N/m}^2 \quad \nu = 0.2 \quad \rho = 2500 \text{ kg/m}^3$$

$$\text{First floor columns: } b \times h = 50 \times 50 \text{ cm}^2 \quad (A = 0.25 \text{ m}^2; I_{zz} = 5.20833 \cdot 10^{-3} \text{ m}^4)$$

$$\text{Second floor columns: } b \times h = 40 \times 40 \text{ cm}^2 \quad (A = 0.16 \text{ m}^2; I_{zz} = 2.1333 \cdot 10^{-3} \text{ m}^4)$$

$$\text{Third floor columns: } b \times h = 30 \times 30 \text{ cm}^2 \quad (A = 0.09 \text{ m}^2; I_{zz} = 0.6750 \cdot 10^{-3} \text{ m}^4)$$

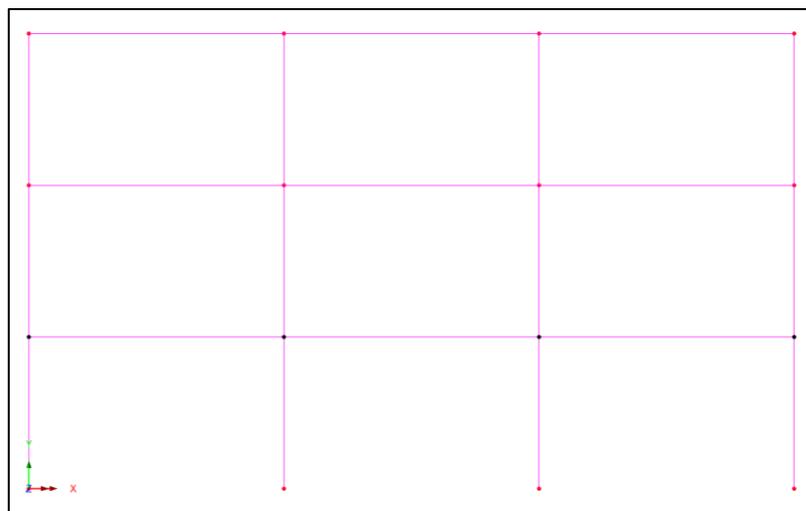
$$\text{Beams: } b \times h = 30 \times 70 \text{ cm}^2 \quad (A = 0.21 \text{ m}^2; I_{zz} = 8.5750 \cdot 10^{-3} \text{ m}^4)$$

Beams are much stiffer than the columns:  $EI(\text{beam}) \gg EI(\text{column})$ .

In an ideal shear type frame, the  $EI(\text{beam})$  goes to infinity.

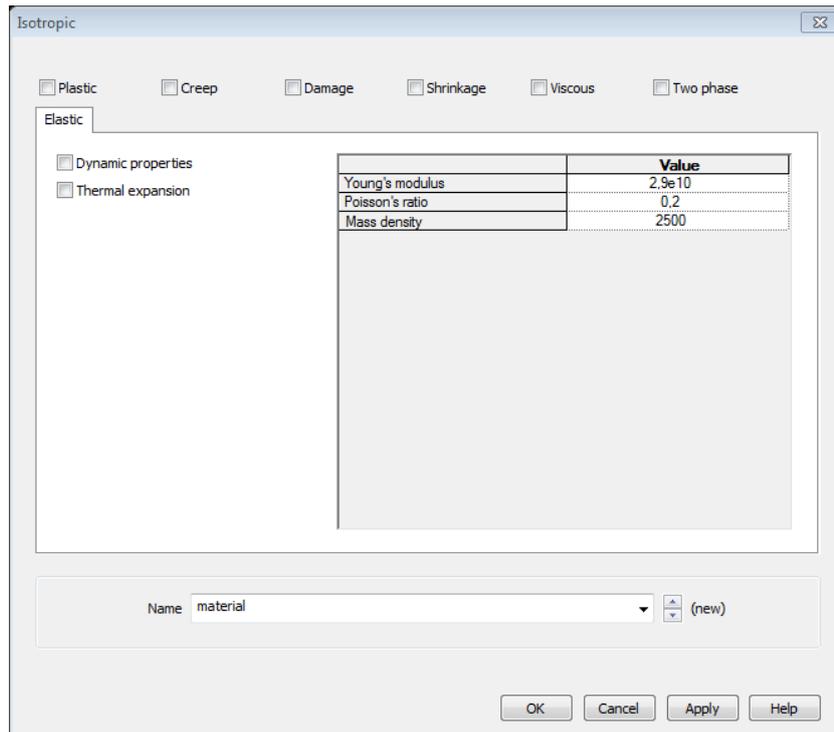
- **How to draw the frame:**

- Insert the points and connect by lines:

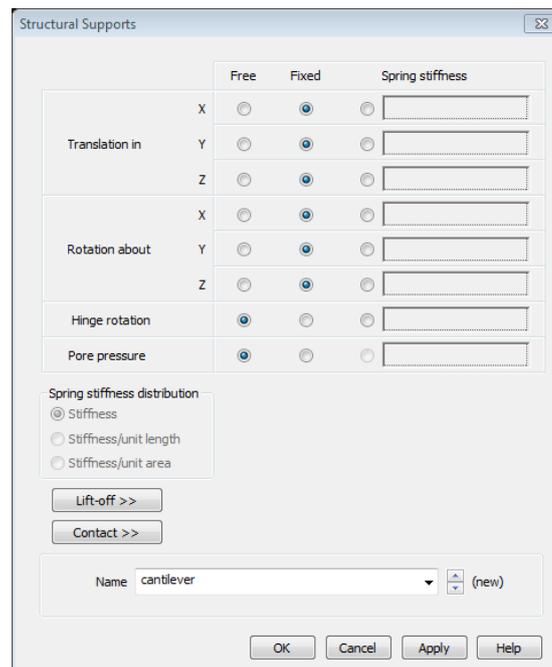


- Assign mesh : 10 subdivisions to each line

-Insert material:



- Assign cantilever support:

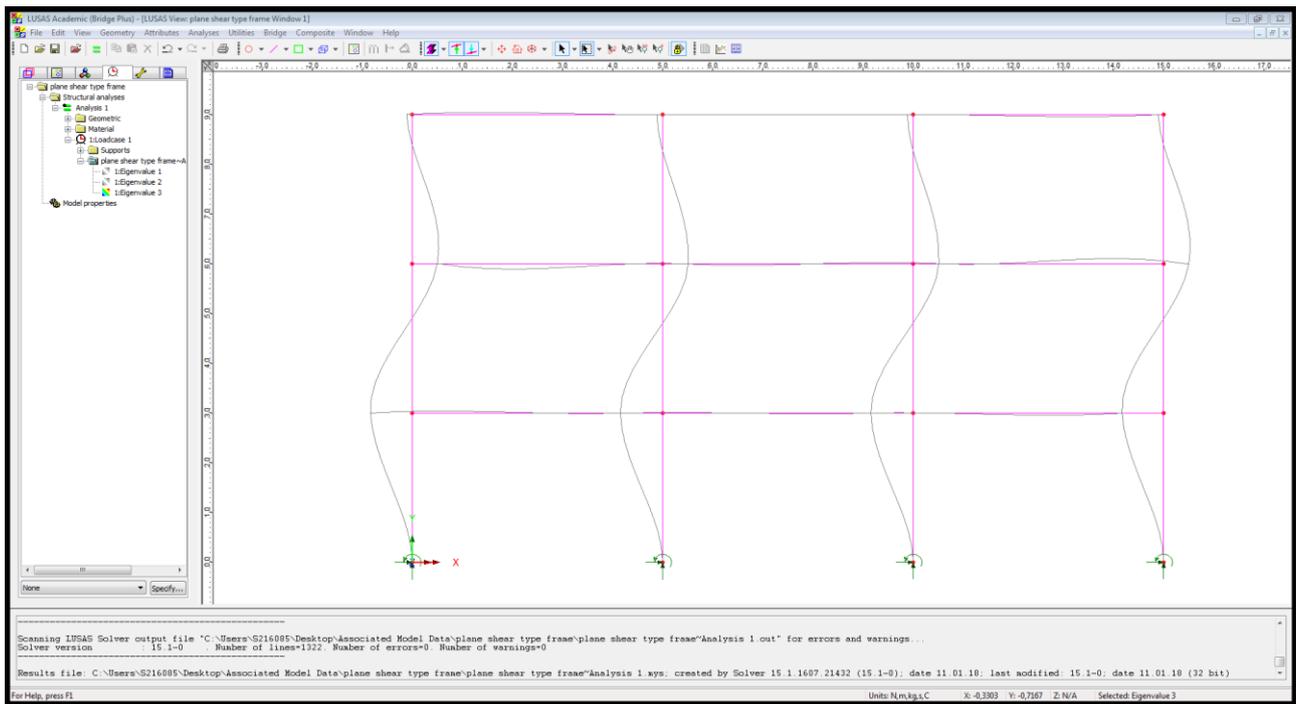


- No load is applied, but only the loadcase1 with 3 eigenvalues: **N = 3 eigenvalues = nr of floors.**

[if nr of floors is 2 then N=2]

**RUN!!!**

- **Mode 3:**



- See the values: *utilities*, *print result wizard*, *next*, *none*, *ok*

N	Eigenvalue	Natural frequency	Error
1	1597.25	6.36073	0.129864E-10
2	8719.54	14.8616	0.447221E-11
3	33523.0	29.1401	0.143019E-05

Displacement unit of measure? [m]

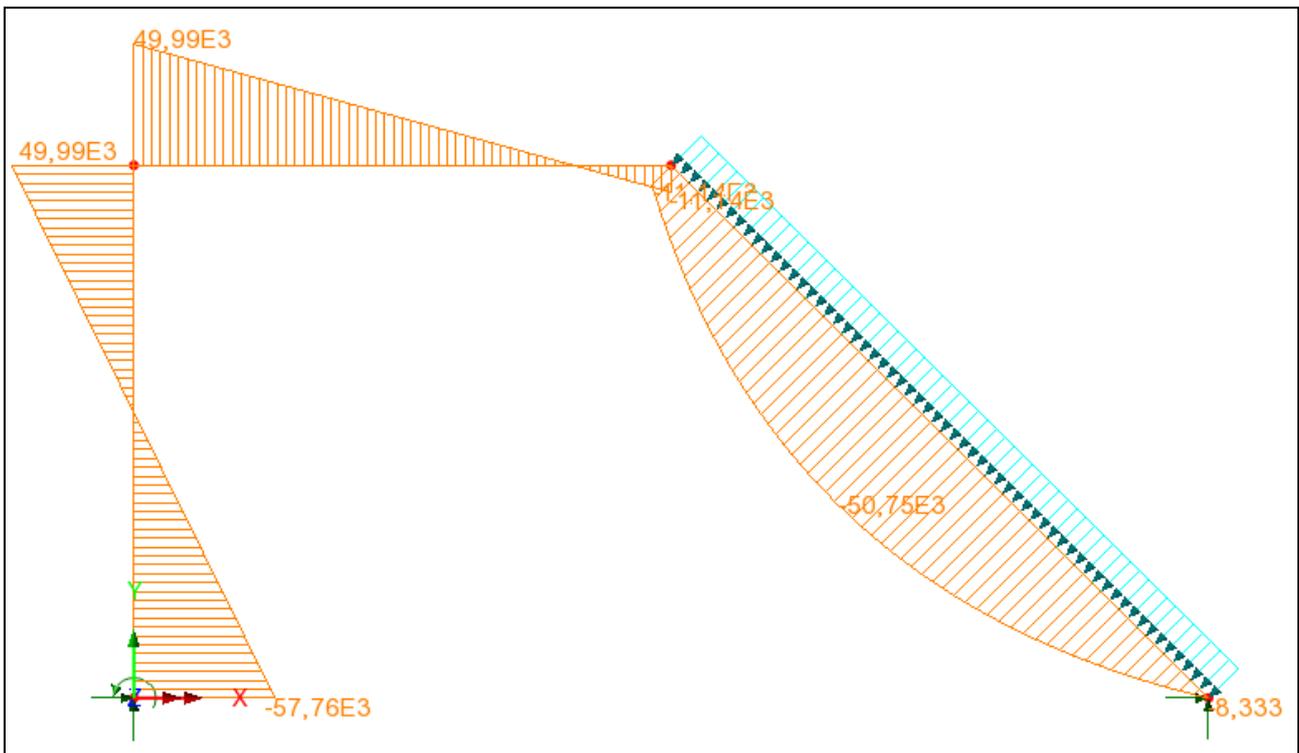
THIN BEAM:

Max displacement (resultant) = 88,0899

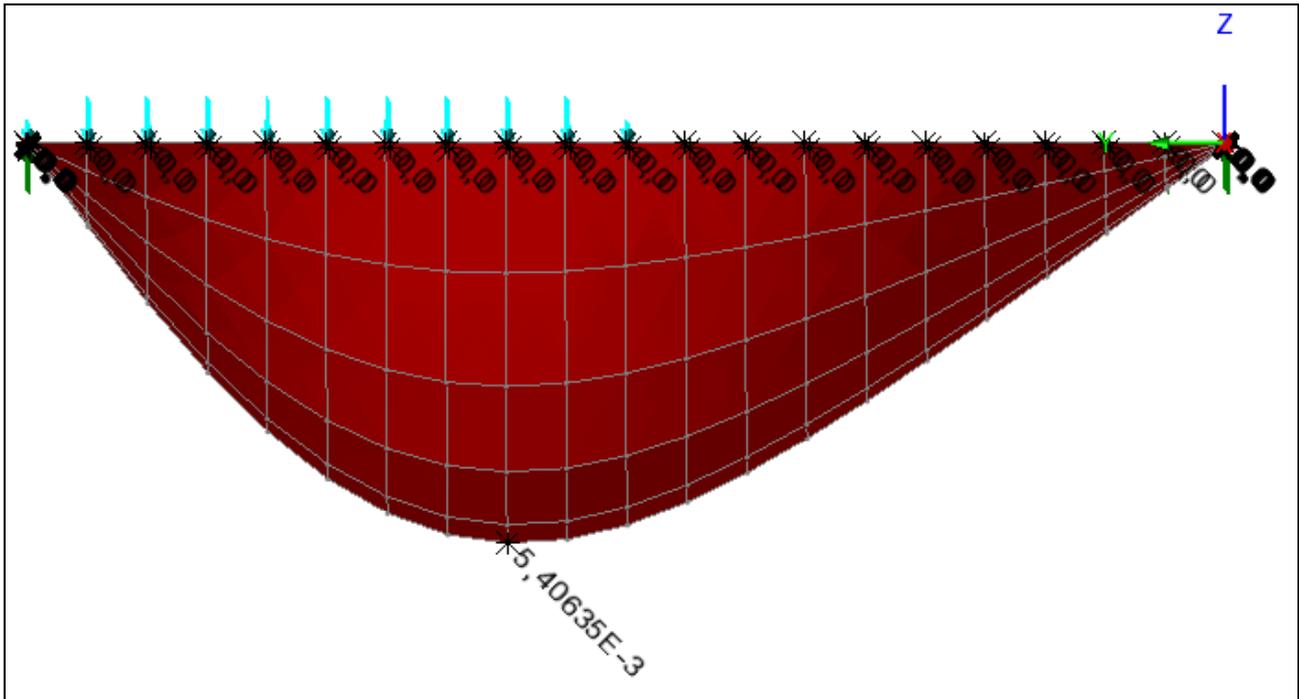
Max displ (Dx) = -62,2881

Max displ (Dy) = -62,2898

Maximum moment (Mz) in abs value =  $57,76 \cdot 10^3$  in the fixed extreme of the vertical beam:



- Deformed mesh:



- Diagram of moments  $M_y$ :

