



Corso Luigi Einaudi, 55 - Torino

Appunti universitari

Tesi di laurea

Cartoleria e cancelleria

Stampa file e fotocopie

Print on demand

Rilegature

NUMERO: 598

DATA: 23/07/2013

A P P U N T I

STUDENTE: Marsicovetere

MATERIA: Fondamenti di Meccanica Strutturale + Eserc.

Prof. Carrera

Il presente lavoro nasce dall'impegno dell'autore ed è distribuito in accordo con il Centro Appunti.

Tutti i diritti sono riservati. È vietata qualsiasi riproduzione, copia totale o parziale, dei contenuti inseriti nel presente volume, ivi inclusa la memorizzazione, rielaborazione, diffusione o distribuzione dei contenuti stessi mediante qualunque supporto magnetico o cartaceo, piattaforma tecnologica o rete telematica, senza previa autorizzazione scritta dell'autore.

**ATTENZIONE: QUESTI APPUNTI SONO FATTI DA STUDENTIE NON SONO STATI VISIONATI DAL DOCENTE.
IL NOME DEL PROFESSORE, SERVE SOLO PER IDENTIFICARE IL CORSO.**

MATTEO

MARSICOVETERE

FONDAMENTI DI MECCANICA STRUTTURALE

ING. AEROSPAZIALE 2012/2013

PROF. CARRERA

$$\delta_{xy} = 2 \epsilon_{xy} = \frac{1}{G} \sigma_{xy}$$

$$\delta_{xz} = 2 \epsilon_{xz} = \frac{1}{G} \sigma_{xz}$$

$$\delta_{yz} = 2 \epsilon_{yz} = \frac{1}{G} \sigma_{yz}$$

$$\sigma_{xx} = (\lambda + 2G) \epsilon_{xx} + \lambda \epsilon_{yy} + \lambda \epsilon_{zz}$$

$$\sigma_{yy} = \lambda \epsilon_{xx} + (\lambda + 2G) \epsilon_{yy} + \lambda \epsilon_{zz}$$

$$\sigma_{zz} = \lambda \epsilon_{xx} + \lambda \epsilon_{yy} + (\lambda + 2G) \epsilon_{zz}$$

$$\sigma_{xy} = G \delta_{xy}$$

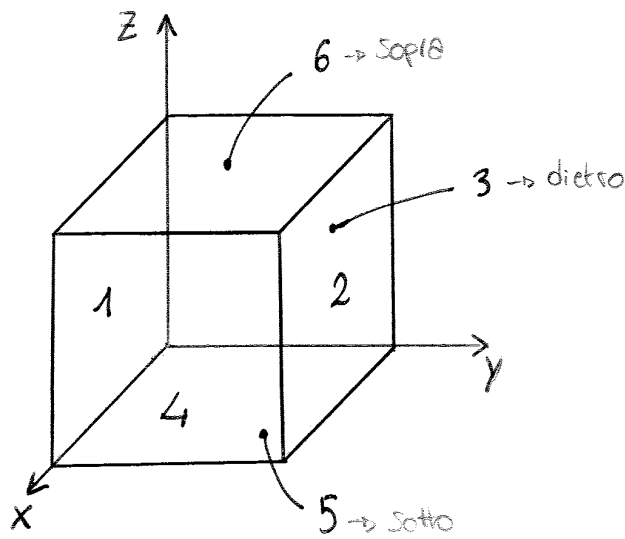
$$\sigma_{xz} = G \delta_{xz}$$

$$\sigma_{yz} = G \delta_{yz}$$

$$\lambda = \frac{E\nu}{(1+\nu)(1-2\nu)}$$

$$G = \frac{E}{2(1+\nu)}$$

me



Trovare le varie I e gli sforzi sulle facce

- ①
- ②
- ③
- ④
- ⑤
- ⑥

②

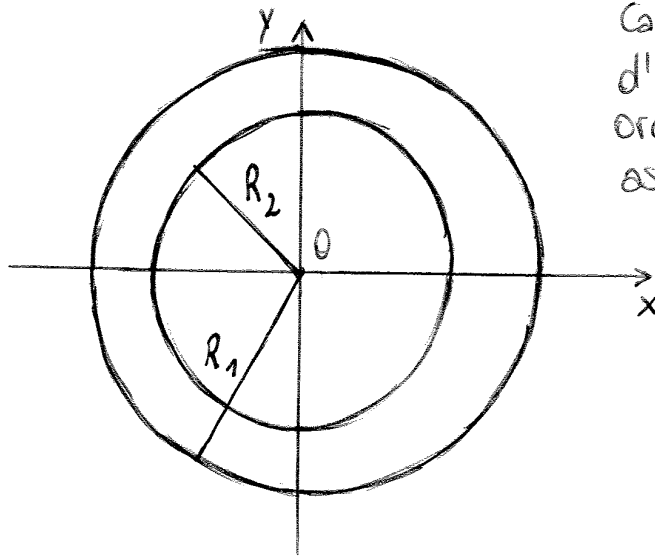
FACCIA		
1	$\sigma_{yx} dx dz$	$dy/2$
2	$\sigma'_{yx} dx dz$	$dy/2$
3	$-\sigma_{xy} dy dz$	$dx/2$
4	$-\sigma'_{xy} dy dz$	$dx/2$
5		0
6		0

supponendo $\bar{I}_x > \bar{I}_y$:

$$I_E = (\bar{I}_x + \bar{I}_y) \frac{1}{2} + \frac{1}{2} \sqrt{(\bar{I}_x - \bar{I}_y)^2 + 4\bar{I}_{xy}^2}$$

$$I_M = \frac{1}{2} (\bar{I}_x + \bar{I}_y) - \frac{1}{2} \sqrt{(\bar{I}_x - \bar{I}_y)^2 + 4\bar{I}_{xy}^2}$$

Al



Calcolare i momenti
d'inerzia del secondo
ordine rispetto agli
assi x y

$$\int_D f \, dx \, dy = \int_{D_1} f \, dx \, dy - \int_{D_2} f \, dx \, dy$$

$$D = D_1 - D_2$$

$$\begin{aligned} x &= r \cos \theta \\ y &= r \sin \theta \\ \det J &= r \end{aligned}$$

$$I_x = \int_A y^2 \, dx \, dy = \int_{A'} (r \sin \theta)^2 r \, dr \, d\theta$$

$$I_x = \left[\frac{1}{2} (\theta - \sin \theta \cos \theta) \right]_0^{2\pi} \cdot \left\{ \left[\frac{r^4}{4} \right]_0^{R_1} - \left[\frac{r^4}{4} \right]_0^{R_2} \right\} =$$

$$= \frac{\pi}{4} (R_1^4 - R_2^4)$$

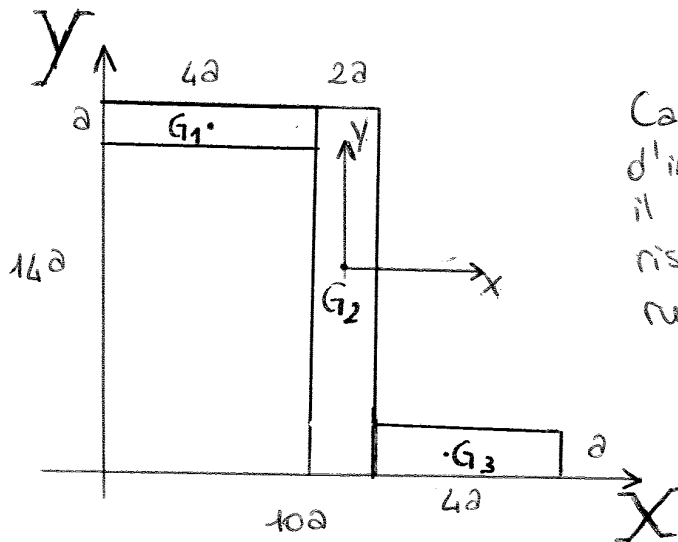
$$I_y = \int_A x^2 \, dx \, dy = \int_{A'} (r \cos \theta)^2 r \, dr \, d\theta$$

$$I_y = \left[\frac{1}{2} (\theta + \sin \theta \cos \theta) \right]_0^{2\pi} \cdot \left\{ \left[\frac{r^4}{4} \right]_0^{R_1} - \left[\frac{r^4}{4} \right]_0^{R_2} \right\} =$$

$$= \frac{\pi}{4} (R_1^4 - R_2^4) = I_x$$

$$I_{xy} = 0$$

②



Calcolare i momenti d'inerzia del secondo ordine, il baricentro e i momenti rispetto ad un riferimento ruotato

Essendo la figura simmetrica rispetto al rettangolo centrale, il baricentro complessivo corrisponde al baricentro del centrale

$$G = \left(4a + \frac{2a}{2}, \frac{14a}{2} \right) = (5a, 7a)$$

$$\begin{aligned} I_x &= \int_A y^2 dx dy = \int_{A_1} y^2 dx dy + \int_{A_2} y^2 dx dy + \int_{A_3} y^2 dx dy = \\ &= 2 \int_{A_1} y^2 dx dy + \int_{A_2} y^2 dx dy \end{aligned}$$

$$I_x = 2 \left(\frac{4a \cdot a^3}{12} + 4a \cdot a \cdot \left(\frac{13}{2} a \right)^2 \right) + \frac{2a \cdot (14a)^3}{12} = 796 a^4$$

$$I_y = 2 \left[\frac{(4a)^3 \cdot a}{12} + 4a^2 (3a)^2 \right] + \frac{(2a)^3 \cdot 14a}{12} = 92 a^4$$

$$I_{xy} = 4a^2 (-3a) \left(\frac{13}{2} a \right) + 4a^2 (3a) \left(-\frac{13}{2} a \right) = -156 a^4$$

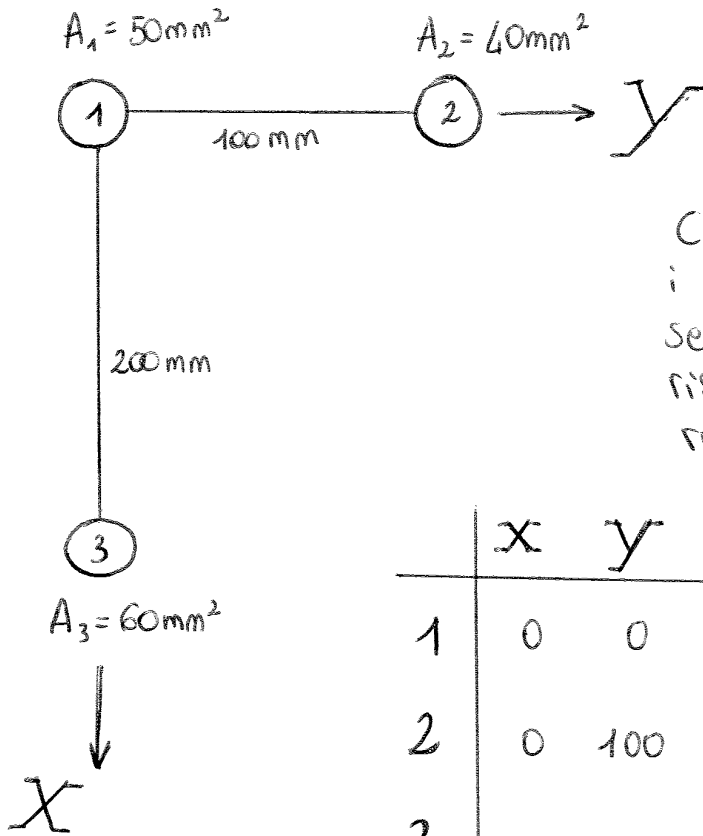
$$S_{x_1} = S_{x_2} \rightarrow \text{si semplificano}$$

$$S_{y_2} = S_{y_1} \rightarrow \text{si semplificano}$$

$$I_{xy} = 0$$

③

Lu



Calcolare il baricentro, i momenti d'inerzia del secondo ordine e i momenti rispetto ad un riferimento ruotato

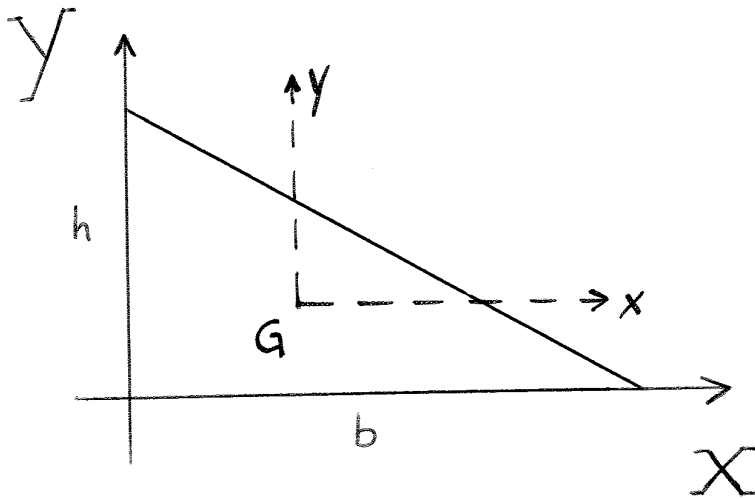
	x	y	A	Ax	Ay
1	0	0	50	0	0
2	0	100	40	0	4000
3	200	0	60	12000	0

$$G_x = \frac{\sum_{i=1}^3 x_i A_i}{\sum_{i=1}^3 A_i} = \frac{12000}{150} = 80 \text{ mm}$$

$$G_y = \frac{\sum_{i=1}^3 y_i A_i}{\sum_{i=1}^3 A_i} = \frac{4000}{150} = 26,6\overline{6} \text{ mm}$$

④

li



$$G_x = \frac{\int_A x \, dx \, dy}{\int_A dx \, dy} = \frac{S_y}{A}$$

$$S_y = \int_0^b x \left[\int_0^{-\frac{h}{b}x+h} dy \right] dx = \int_0^b \left(-\frac{h}{b}x^2 + hx \right) dx =$$

$$= \left[-\frac{1}{3}x^3 \frac{h}{b} + \frac{1}{2}hx^2 \right]_0^b = \frac{hb^2}{6}$$

$$G_x = \frac{hb^2}{6} \cdot \frac{2}{hb} = \frac{b}{3}$$

$$S_x = \int_0^h y \left[\int_0^{-\frac{b}{h}(y-h)} dx \right] dy = \int_0^h \left(-\frac{b}{h}y^2 + by \right) dy =$$

$$= \left[-\frac{1}{3} \frac{b}{h} y^3 + \frac{1}{2} by^2 \right]_0^h = \frac{h^2b}{6}$$

$$G_y = \frac{bh^2}{6} \cdot \frac{2}{bh} = \frac{h}{3}$$

⑤

$$I_x = I_{x_c} - dy^2 A - \underbrace{2 S_x dy}_{\text{dato che sono nel baricentro } \bar{e} = 0}$$

$$I_x = \frac{bh^3}{12} - \left(\frac{h}{3}\right)^2 \cdot \frac{bh}{2} = \frac{bh^3}{36}$$

$$I_y = I_{y_c} - dx^2 A$$

$$I_y = \frac{b^3h}{12} - \left(\frac{b}{3}\right)^2 \cdot \frac{bh}{2} = \frac{b^3h}{36}$$

$$I_{xy} = I_{x_y} - dx dy A$$

$$I_{xy} = \frac{h^2 b^2}{24} - \frac{h}{3} \cdot \frac{b}{3} \cdot \frac{bh}{2} = -\frac{b^2 h^2}{72}$$

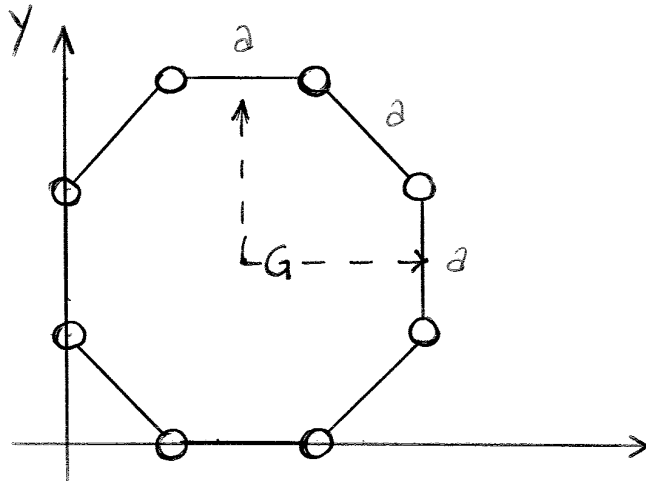
$$\tilde{\theta} = \frac{1}{2} \operatorname{arctg} \left[-2 \frac{-\frac{b^2 h^2}{72}}{\frac{bh^3}{36} - \frac{b^3 h}{36}} \right] = \frac{1}{2} \operatorname{arctg} \left[\frac{bh}{h^2 - b^2} \right]$$

$$I_\varepsilon = \frac{1}{2} (I_x + I_y) + \frac{1}{2} \sqrt{(I_x - I_y)^2 + 4 I_{xy}^2}$$

$$I_m = \frac{1}{2} (I_x + I_y) - \frac{1}{2} \sqrt{(I_x - I_y)^2 + 4 I_{xy}^2}$$

5.2

li



Calcolare il baricentro
dando una misura
ai lati

Prendo un ottagono regolare \rightarrow tutti i lati uguali

$$G_x = G_y = \frac{a}{2} + \frac{a}{\sqrt{2}} = \frac{a + a\sqrt{2}}{2} = \frac{a}{2}(1 + \sqrt{2})$$

7

$$I_x = \int_A y^2 dx dy = \frac{bh^3}{12}$$

$$I_y = \int_A x^2 dx dy = \frac{b^3h}{12}$$

$$EI_x = E_1 \frac{h_1^3}{3} b + E_2 \frac{h_2^3}{3} b$$

Determinare la matrice $[?]$ sapendo che

$$[?] \begin{Bmatrix} \sigma \end{Bmatrix} = \begin{Bmatrix} 0 \end{Bmatrix}$$

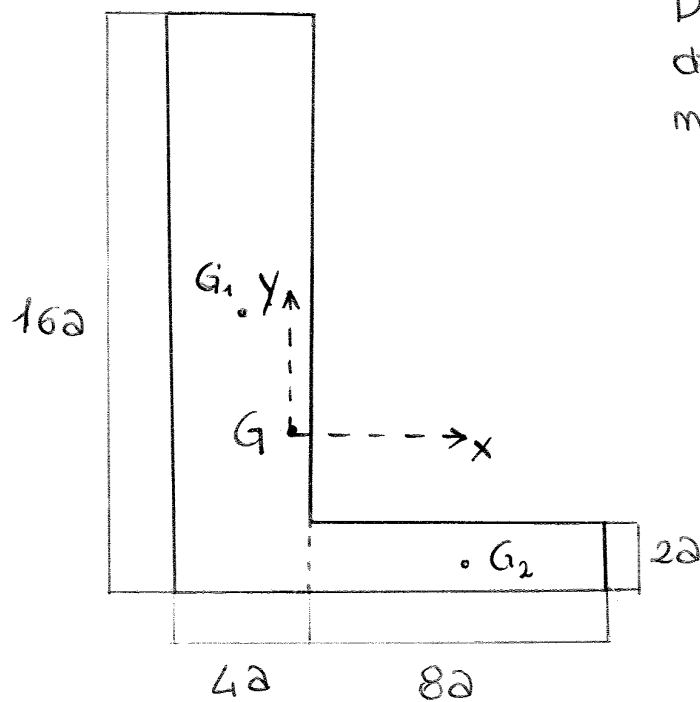
$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{xy} \\ \sigma_{xz} \\ \sigma_{yz} \end{bmatrix} \begin{bmatrix} \frac{\partial \sigma}{\partial x} & 0 & 0 & \frac{\partial \sigma}{\partial y} & \frac{\partial \sigma}{\partial z} & 0 \\ \vdots & \vdots & \vdots & \vdots & \vdots & \vdots \\ 0 & \frac{\partial \sigma}{\partial y} & 0 & \frac{\partial \sigma}{\partial x} & 0 & \frac{\partial \sigma}{\partial z} \\ 0 & 0 & \frac{\partial \sigma}{\partial z} & 0 & \frac{\partial \sigma}{\partial x} & \frac{\partial \sigma}{\partial y} \end{bmatrix}$$

$$[?]^* = ([?^{**}])^T \quad \text{PERCHÉ?}$$

$[?^{**}]^T = [?^*]$ in quanto, moltiplicando le matrici $[?^{**}]$ e $[\sigma]$ ottengo le equazioni indefinite di equilibrio che so essere = 0 per ipotesi.

(2)

ESERCITAZIONE 2

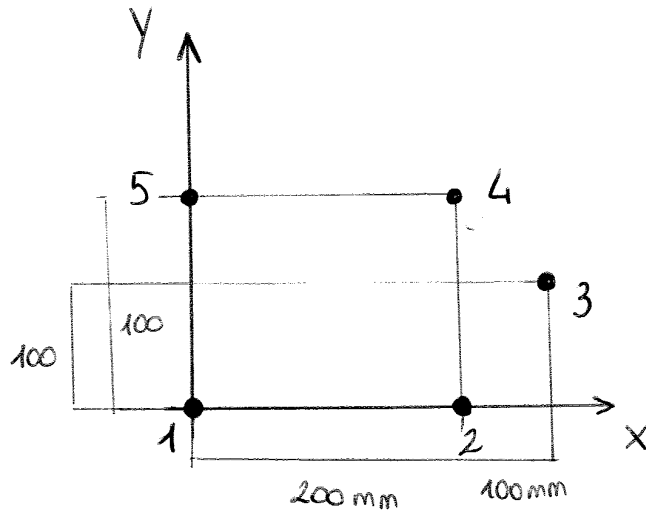


Determinare il baricentro, le direzioni principali e il momento centrifugo

$$G = \left(\frac{2a \cdot 16 \cdot 4a^2 + 8a \cdot 16a^2}{16 \cdot 4a^2 + 16a^2}, \frac{8a \cdot 16 \cdot 4a^2 + a \cdot 16a^2}{16 \cdot 4a^2 + 16a^2} \right) =$$

$$= (3, 2a; 6, 6a)$$

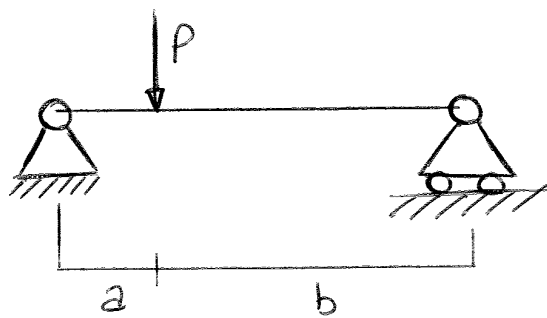
①



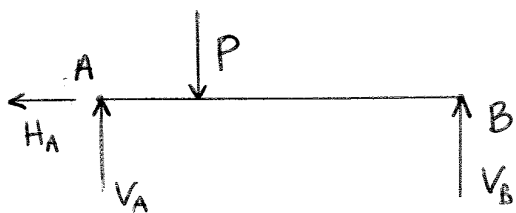
	X_i	Y_i	A_i	$X_i A_i$	$Y_i A_i$
1	0	0	100	0	0
2	200	0	50	10'000	0
3	300	100	20	6'000	2'000
4	200	200	50	10'000	10'000
5	0	200	100	0	20'000

$$G = \left(\frac{26'000}{320} ; \frac{32'000}{320} \right) = (81,25 \text{ mm} ; 100 \text{ mm})$$

②



Calcolare le reazioni vincolari



$$H_A = 0$$

$$V_A + V_B = P$$

$$\sum \curvearrowright V_B(a+b) = Pa$$

3

TEORIA 3

$$\left\{ \begin{array}{l} \frac{\sigma_{xx}}{\partial x} + \frac{\sigma_{xy}}{\partial y} + \frac{\sigma_{xz}}{\partial z} = t_x \\ \frac{\sigma_{yx}}{\partial x} + \frac{\sigma_{yy}}{\partial y} + \frac{\sigma_{yz}}{\partial z} = t_y \\ \frac{\sigma_{zx}}{\partial x} + \frac{\sigma_{zy}}{\partial y} + \frac{\sigma_{zz}}{\partial z} = t_z \end{array} \right.$$

$$\begin{aligned} \sigma_{xx,x} &= (\lambda + 2G) \varepsilon_{xx,x} + \lambda \varepsilon_{yy,x} + \lambda \varepsilon_{zz,x} = \\ &= (\lambda + 2G) u_{,x,x} + \lambda v_{,y,x} + \lambda w_{,z,x} \end{aligned}$$

$$\sigma_{xy,y} = G \gamma_{xz,z} = G(u_{,y} + v_{,x})_{,y}$$

$$\sigma_{xz,z} = G \gamma_{xz,z} = G(u_{,z} + w_{,x})_{,z}$$

$$\sigma_{yx,x} = G \gamma_{xy,x} = G(u_{,y} + v_{,x})_{,x}$$

①

MATERIALE	E [GPa]	λ [GPa]	G [GPa]	ν	C_{11}^* [GPa]	C_{11}^{**} [GPa]	diff %
OTONE	96	76	86	0,34	148	96	54,1
NIKEL	240	130,7	80	0,31	290,7	240	38,43
FERRO	83	23	35	6,2	93	83	12,05
ACCIAIO	180	55	75	0,27	205	180	7,9
TUNGSTENO	340	94	142	0,2	378	340	11,2
BRONTO	96	76	36	0,36	148	96	54,2
LEGA Mg	41	35	15	0,35	65	41	58,5
LEGA Ti	100	73	38	0,33	149	100	49
LEGA AL (2014-T6)	73	53	27	0,33	107	73	46,6
VETRO	48	10,6	20,5	0,17	51,6	48	7,5

(2)

Dimostrare che le equazioni di equilibrio per l'asta sono uguali a zero:

$$R_{xx} = \frac{\partial^2 E_{yy}}{\partial z^2} + \frac{\partial^2 E_{zz}}{\partial y^2} - \frac{\partial^2 \sigma_{yz}}{\partial y \partial z} = 0$$

$$R_{yy} = \frac{\partial^2 E_{xx}}{\partial z^2} + \frac{\partial^2 E_{zz}}{\partial x^2} - \frac{\partial^2 \sigma_{xz}}{\partial x \partial z} = 0$$

$$R_{zz} = \frac{\partial^2 E_{xx}}{\partial y^2} + \frac{\partial^2 E_{yy}}{\partial x^2} - \frac{\partial^2 \sigma_{xy}}{\partial x \partial y} = 0$$

$$R_{yz} = \frac{\partial}{\partial x} \left[\frac{\partial \sigma_{xz}}{\partial y} + \frac{\partial \sigma_{xy}}{\partial z} - \frac{\partial \sigma_{yz}}{\partial x} \right] - \frac{2 \partial^2 E_{xx}}{\partial y \partial z} = 0$$

$$R_{xz} = \frac{\partial}{\partial y} \left[\frac{\partial \sigma_{xy}}{\partial z} + \frac{\partial \sigma_{yz}}{\partial x} - \frac{\partial \sigma_{xz}}{\partial y} \right] - \frac{2 \partial^2 E_{yy}}{\partial x \partial z} = 0$$

$$R_{yt} = \frac{\partial}{\partial z} \left[\frac{\partial \sigma_{yz}}{\partial x} + \frac{\partial \sigma_{xz}}{\partial y} - \frac{\partial \sigma_{xy}}{\partial z} \right] - \frac{2 \partial^2 E_{zz}}{\partial y \partial x} = 0$$

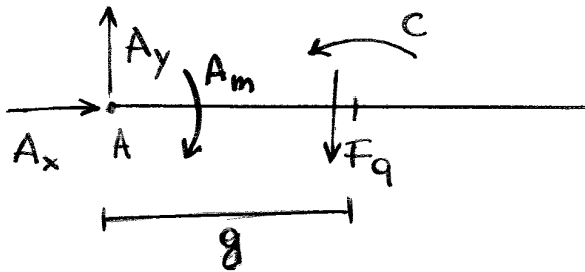
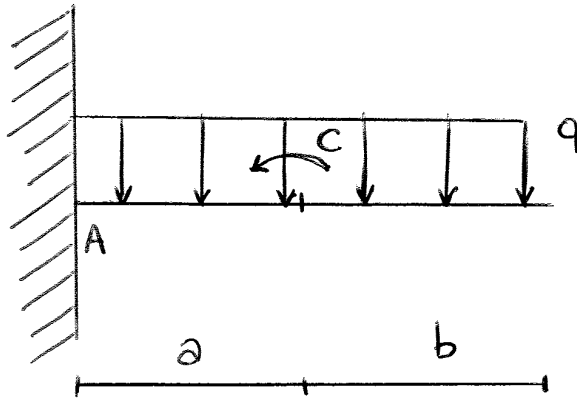
per ipotesi: nell'asta

$E_{xx} = 0$	$\sigma_{xy} = 0$
$E_{yy} = 0$	$\sigma_{xz} = 0$
$E_{zz} = 0$	$\sigma_{yz} = 0$

ne consegue che:

$R_{xx} = 0$	$R_{xy} = 0$
$R_{yy} = 0$	$R_{xz} = 0$
$R_{zz} = 0$	$R_{yz} = 0$

③



$$\left\{ \begin{array}{l} \sum F_x = 0 \\ \sum F_y = 0 \\ \sum M_A = 0 \end{array} \right. \quad \left\{ \begin{array}{l} A_x = 0 \\ A_y + F_y = 0 \\ F_q \cdot g - C + A_m = 0 \end{array} \right.$$

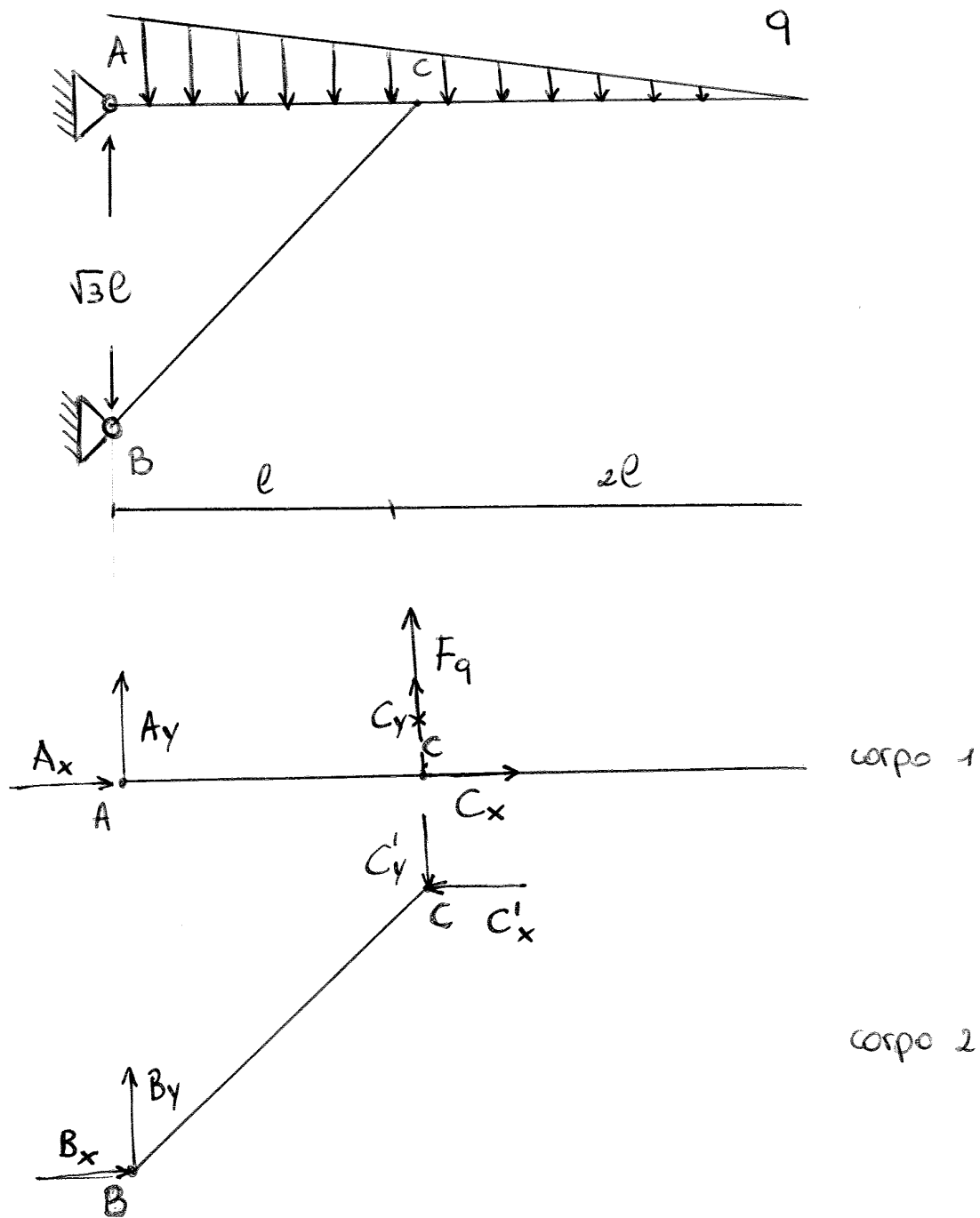
$$F_q = q \cdot (a+b)$$

$$g = \frac{a+b}{2}$$

$$q \cdot (a+b)^2 \cdot \frac{1}{2} - C + A_m = 0$$

$$A_m = C - \frac{q}{2} (a+b)^2$$

2



4

Il lavoro esterno è pari a:

$$L_e^{ab} = \int_S \vec{P}^a \cdot \vec{S}^b dS = \int_S \hat{\sigma}^a \vec{n} \vec{S}^b dS =$$

$$= \int_S u^b (\sigma_{xx}^a n_x + \sigma_{xy}^a n_y + \sigma_{xz}^a n_z) + v^b (\sigma_{yx}^a n_x + \sigma_{yy}^a n_y + \sigma_{yz}^a n_z) + w^b (\sigma_{zx}^a n_x + \sigma_{zy}^a n_y + \sigma_{zz}^a n_z) dS$$

Imponendo l'uguaglianza tra la somma di tutti i lavori interni e quello esterno, e applicando le dovute semplificazioni, si ottiene:

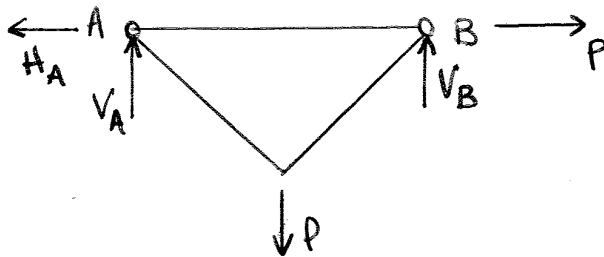
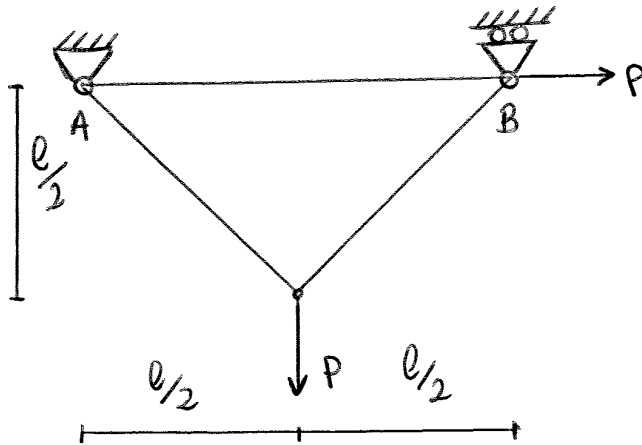
$$\int_V u^b \frac{\partial \sigma_{xx}^a}{\partial x} + u^b \frac{\partial \sigma_{xy}^a}{\partial y} + v^b \frac{\partial \sigma_{xy}^a}{\partial x} + v^b \frac{\partial \sigma_{yy}^a}{\partial y} +$$

$$+ v^b \frac{\partial \sigma_{yz}^a}{\partial z} + w^b \frac{\partial \sigma_{yz}^a}{\partial y} + w^b \frac{\partial \sigma_{zz}^a}{\partial z} + w^b \frac{\partial \sigma_{xz}^a}{\partial x} + u^b \frac{\partial \sigma_{xz}^a}{\partial z} dV = 0$$

Dal principio dei lavori virtuali è quindi possibile ricavare le equazioni indefinite d'equilibrio.

$$\begin{pmatrix} \frac{\partial \sigma_{xx}^a}{\partial x} + \frac{\partial \sigma_{xy}^a}{\partial y} + \frac{\partial \sigma_{xz}^a}{\partial z} \\ \frac{\partial \sigma_{yx}^a}{\partial x} + \frac{\partial \sigma_{yy}^a}{\partial y} + \frac{\partial \sigma_{yz}^a}{\partial z} \\ \frac{\partial \sigma_{zx}^a}{\partial x} + \frac{\partial \sigma_{zy}^a}{\partial y} + \frac{\partial \sigma_{zz}^a}{\partial z} \end{pmatrix} \begin{pmatrix} u^b \\ v^b \\ w^b \end{pmatrix} = 0$$

ESERCITAZIONE 4



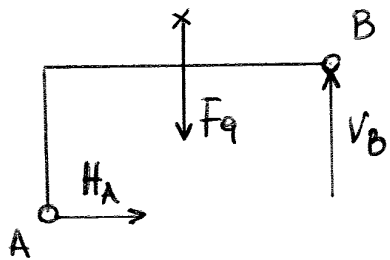
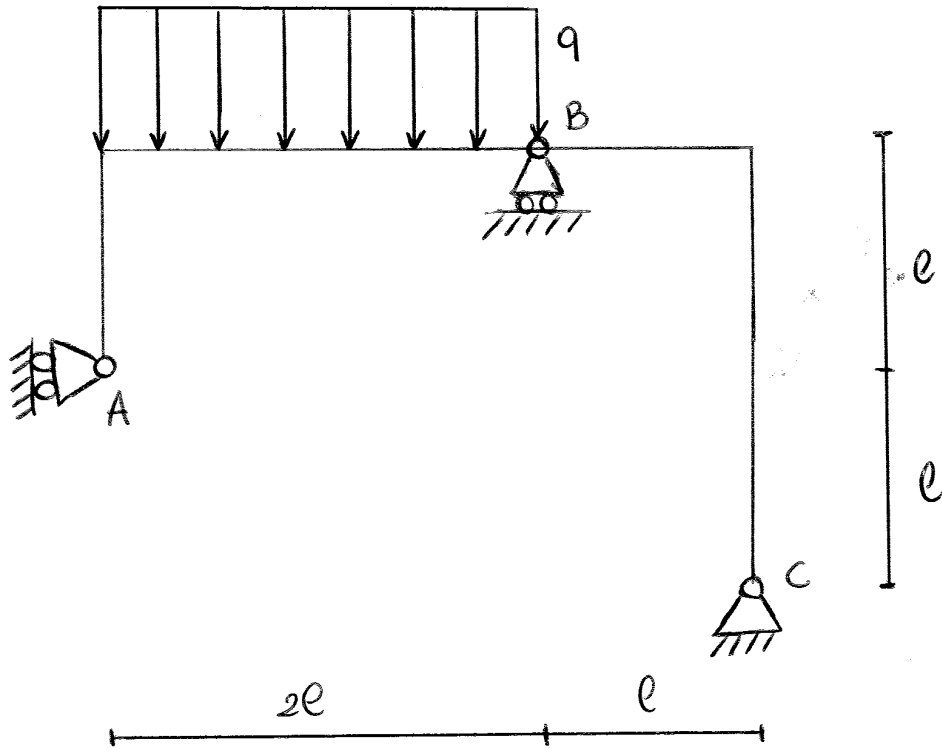
$$\left\{ \begin{array}{l} \rightarrow H_A - P = 0 \\ \uparrow V_A + V_B - P = 0 \\ \curvearrowright P \frac{l}{2} - V_B l = 0 \end{array} \right.$$

$$H_A = P$$

$$V_B = \frac{P}{2}$$

$$V_A = \frac{P}{2}$$

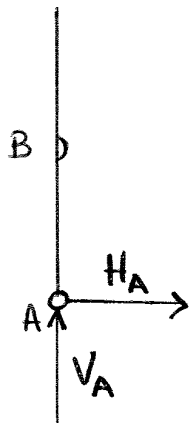
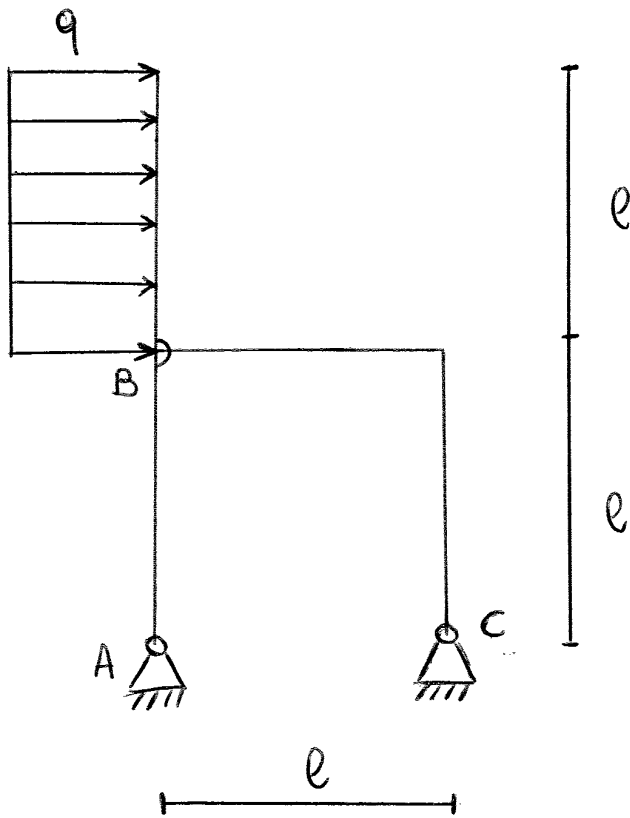
①



$$c) - F_q \cdot e - H_A \cdot e = 0$$

$$H_A = -2qe$$

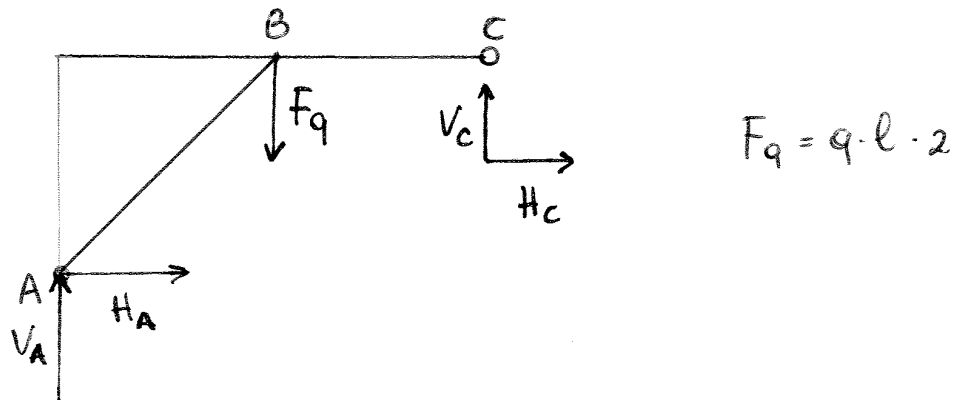
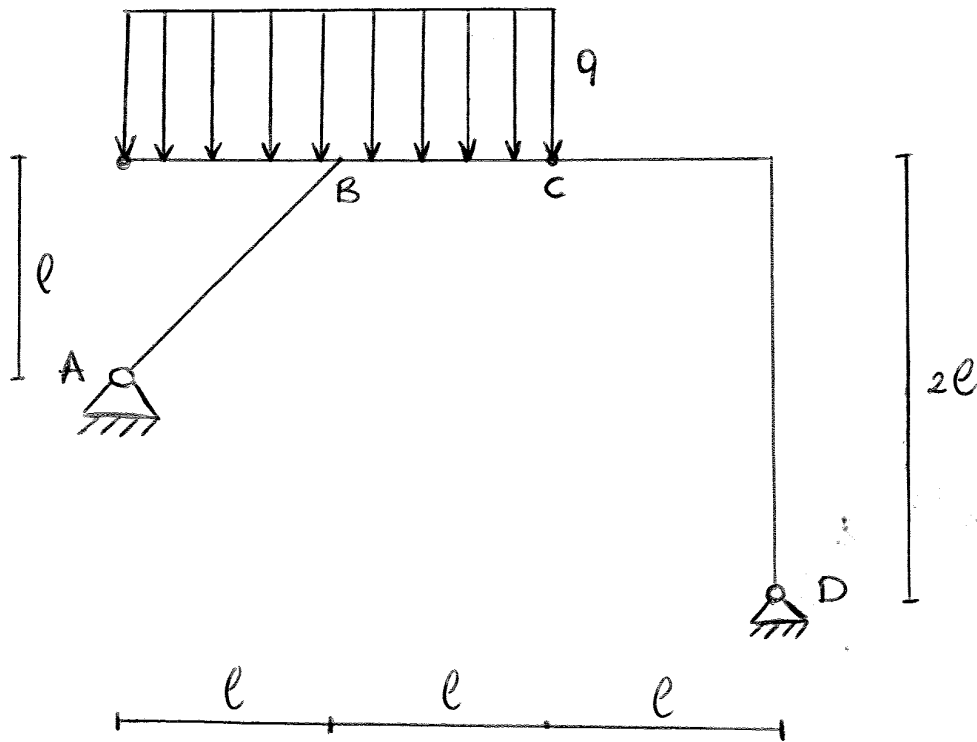
③



$$B) F_q \cdot \frac{e}{2} - H_A \cdot e$$

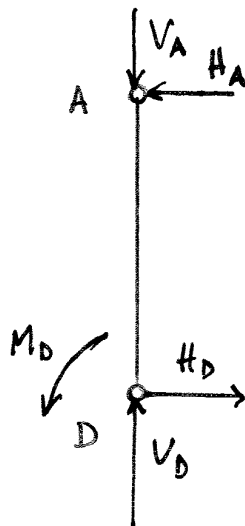
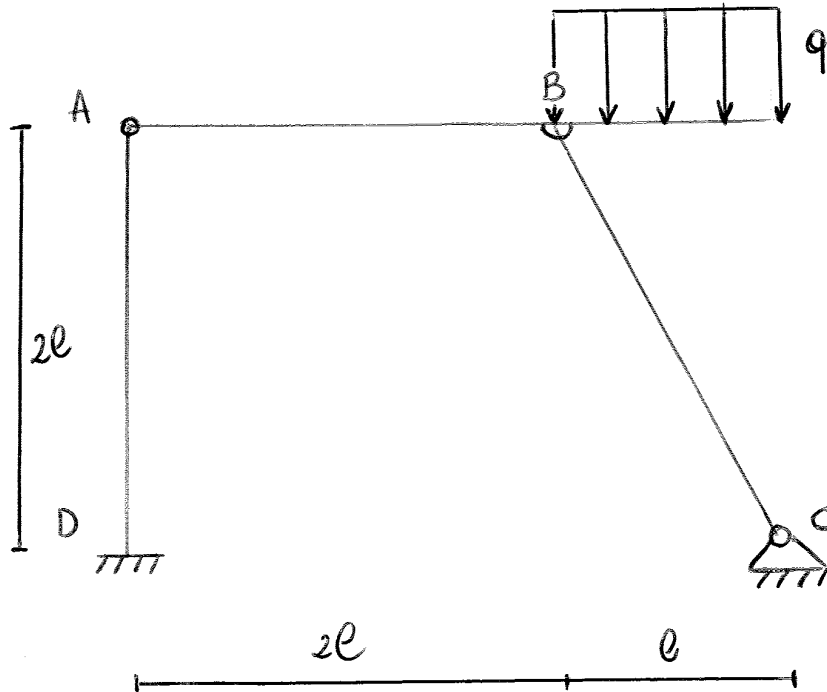
$$H_A = \frac{F_q}{2} = \frac{qe}{2}$$

4



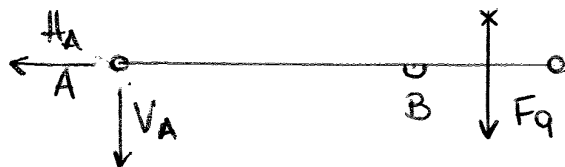
$$\begin{aligned}
 c) \quad & V_A \cdot 2l - F_q \cdot l - H_A \cdot l = 0 \\
 & 2V_A = q \cdot l + H_A \\
 & V_A = \frac{2ql + H_A}{2}
 \end{aligned}$$

5



$$B) \quad 2lV_D - 2lH_D - M_D = 0$$

$$M_D = 2lV_D - 2lH_D$$



$$V_A = V_D$$

$$B) \quad V_D \cdot 2l + F_q \cdot \frac{l}{2} = 0$$

$$V_D = - \frac{F_q}{4}$$

⑥

TEORIA 5

Verificare che da $N_z = -P(z)$ si ricava w :

$$N = \sigma_{zz} A = E \epsilon_{zz} A = EA \frac{\partial w}{\partial z}$$

$$N = \left(EA \frac{\partial w}{\partial z} \right)_{,z} = EA \frac{\partial^2 w}{\partial z^2}$$

↓
= costante

$$EA \frac{\partial^2 w}{\partial z^2} = -P(z)$$

Pongo $P(z) = P \operatorname{sen} \left(\frac{n\pi z}{L} \right)$ quindi:

$$EA \frac{\partial^2 w}{\partial z^2} = -P \operatorname{sen} \left(\frac{n\pi z}{L} \right)$$

$$\int EA \frac{\partial^2 w}{\partial z^2} dz = \int -P \operatorname{sen} \left(\frac{n\pi z}{L} \right) dz$$

$$\int EA \frac{\partial w}{\partial z} = \frac{LP}{n\pi} \int \cos \left(\frac{n\pi z}{L} \right) dz$$

$$EA w = \frac{L^2}{n^2 \pi^2} \operatorname{sen} \left(\frac{n\pi z}{L} \right) \dots$$

$$w(z) = \frac{L^2}{n^2 \pi^2 EA} \operatorname{sen} \left(\frac{n\pi z}{L} \right)$$

$$w(0) = 0$$

$$w(L) = 0$$

①

Ricaviamo $\bar{\Phi}_x$ e $\bar{\Phi}_y$ partendo dalle formule di Timoshenko:

$$u(x, y, z) = u^0(z)$$

$$v(x, y, z) = v^0(z)$$

$$w(x, y, z) = w^0(z) + x \bar{\Phi}_x(z) + y \bar{\Phi}_y(z)$$

$$\sigma_{xz} = \frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} = u'_{,z}(z) + \bar{\Phi}_x(z) = 0$$

$$\bar{\Phi}_x(z) = -u'_{,z}(z)$$

$$\sigma_{yz} = \frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} = v'_{,z}(z) + \bar{\Phi}_y(z) = 0$$

$$\bar{\Phi}_y = -v'_{,z}(z)$$

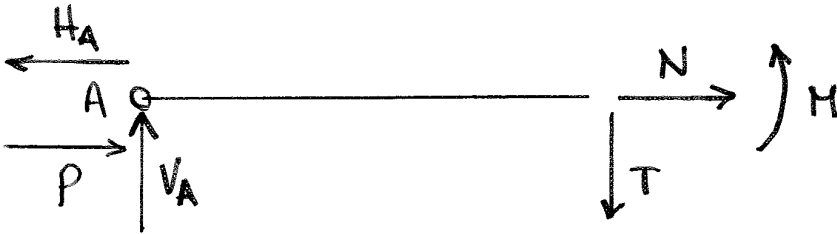
Quindi ricavo:

$$u(x, y, z) = u^0(z)$$

$$v(x, y, z) = v^0(z)$$

$$w(x, y, z) = w^0(z) - x u'_{,z}(z) - y v'_{,z}(z)$$

tratto \overline{AC} : $0 < z < a$

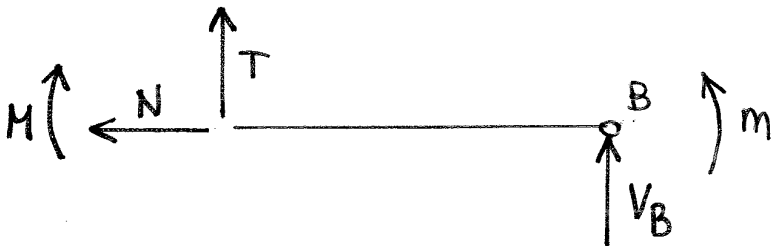


$$N = H_A - P = 0$$

$$T = V_A = \frac{P \cdot b + m}{a + b}$$

$$M = V_A \cdot z = \begin{cases} M_A = 0 \\ M_C = V_A \cdot a = \frac{P \cdot b + m}{a + b} \cdot a \end{cases}$$

tratto \overline{BC} : $0 < z < b$



$$N = 0$$

$$T = -V_B = -\frac{P \cdot a - m}{a + b}$$

$$M = m + V_B \cdot z = \begin{cases} M_C = m + V_B \cdot b = \frac{P \cdot b + m}{a + b} \cdot a \\ M_B = m \end{cases}$$

$$P = 10\text{N}$$

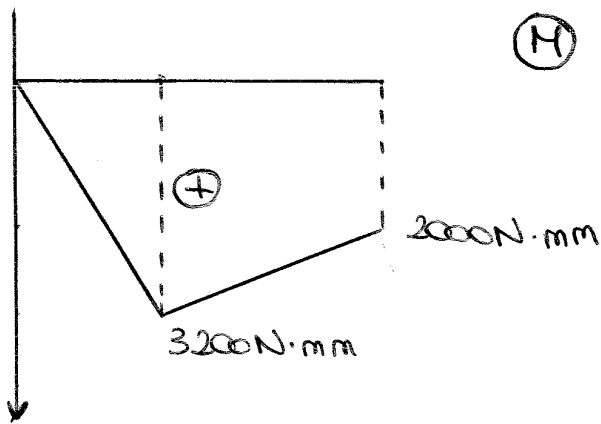
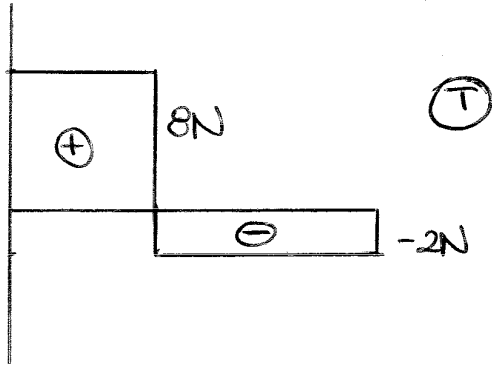
$$m = 2000\text{N}\cdot\text{mm}$$

$$a = 400\text{mm}$$

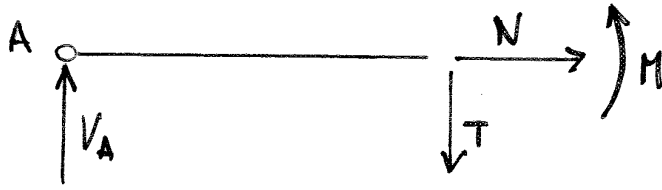
$$b = 600\text{mm}$$

$$\text{tratto } AC: \begin{cases} T = 8\text{N} \\ M_A = 0 \\ M_C = 3200\text{N}\cdot\text{mm} \end{cases}$$

$$\text{tratto } CB: \begin{cases} T = -2\text{N} \\ M_C = 3200\text{N}\cdot\text{mm} \\ M_B = 2000\text{N}\cdot\text{mm} \end{cases}$$



tratto \overline{AB} : $0 < z < a$

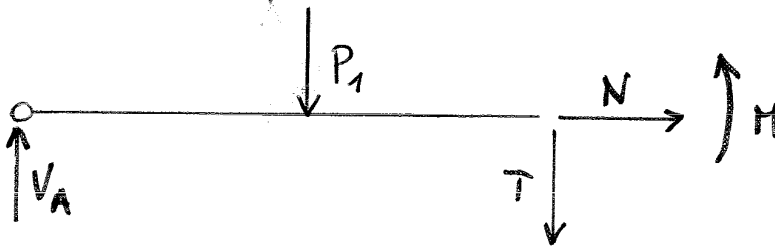


$$N = 0$$

$$T = V_A = 20 \text{ N}$$

$$M = V_A \cdot z = \begin{cases} M_A = 0 \\ M_B = V_A \cdot a = 20 \cdot 1000 \text{ N} \cdot \text{mm} \end{cases}$$

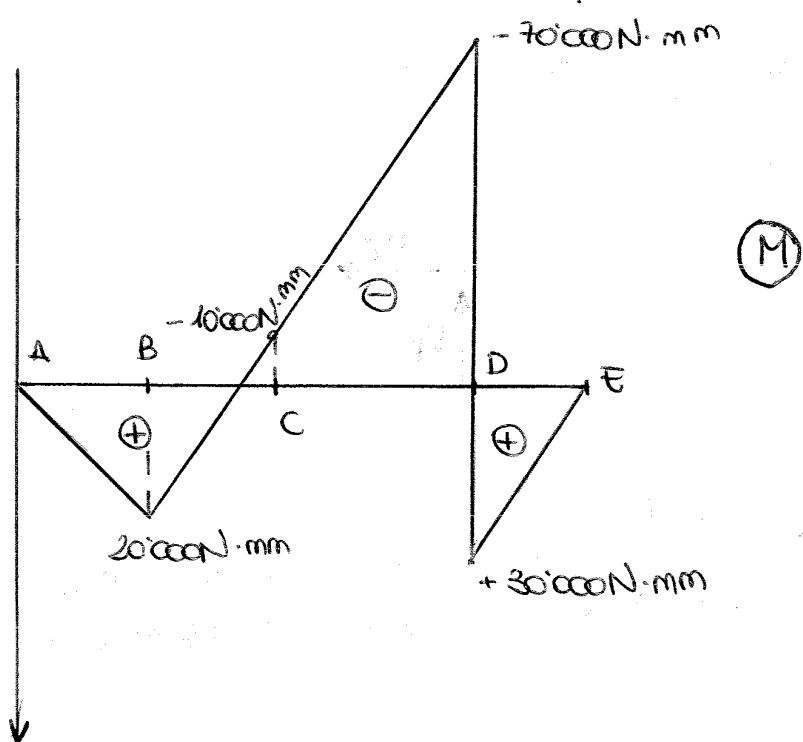
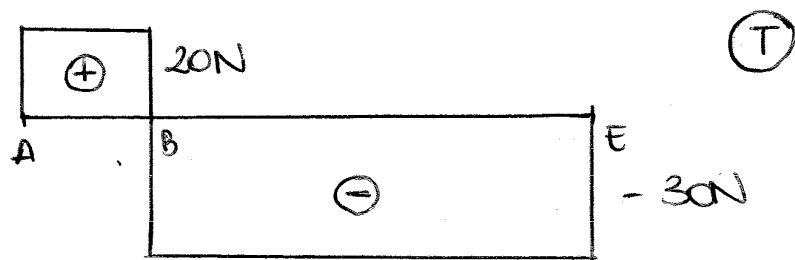
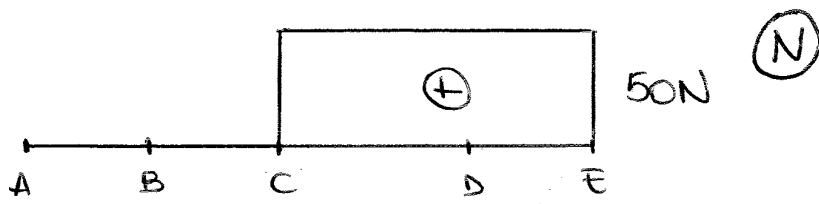
tratto \overline{AC} : $a < z < 2a$



$$N = 0$$

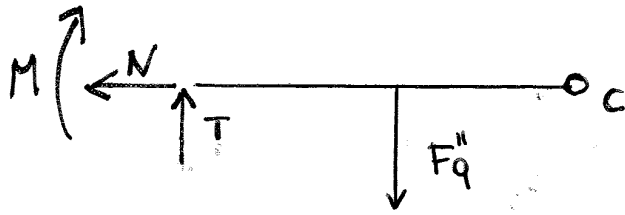
$$T = V_A - P_1 = -30 \text{ N}$$

$$M = V_A \cdot z - P_1 \cdot (z - a) = \begin{cases} M_B = 20 \cdot 1000 \text{ N} \cdot \text{mm} \\ M_C = -10 \cdot 1000 \text{ N} \cdot \text{mm} \end{cases}$$



tratto \overline{CB} : $0 < z < b$

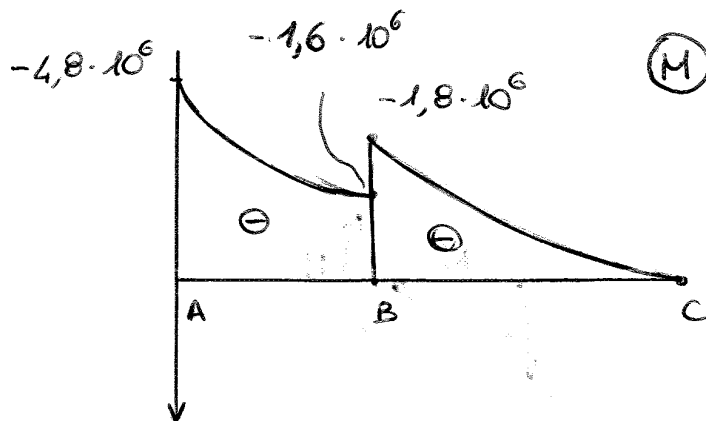
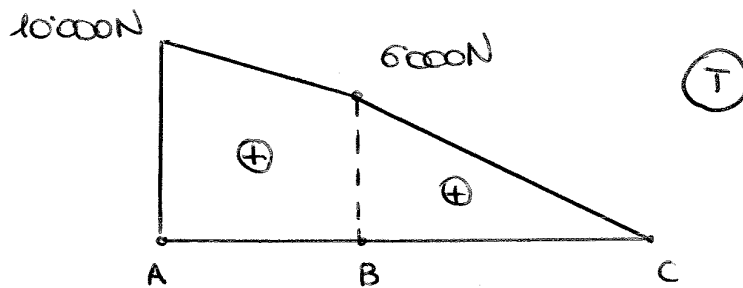
$$F_q'' = q \cdot z$$



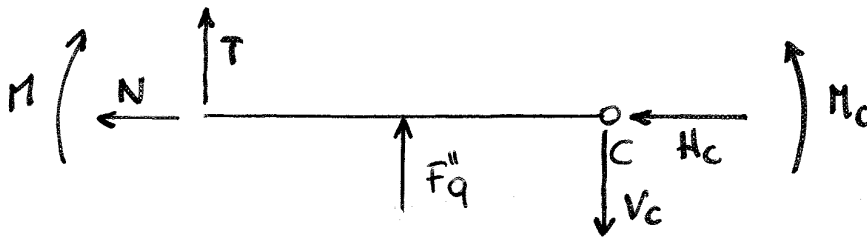
$$N = 0 \text{ N}$$

$$T = F_q'' = \begin{cases} T_c = 0 \text{ N} \\ T_B = qb = 6'000 \text{ N} \end{cases}$$

$$M = -F_q'' \cdot \frac{z}{2} = \begin{cases} M_c = 0 \text{ N} \cdot \text{mm} \\ M_B = -1'800'000 \text{ N} \cdot \text{mm} \end{cases}$$



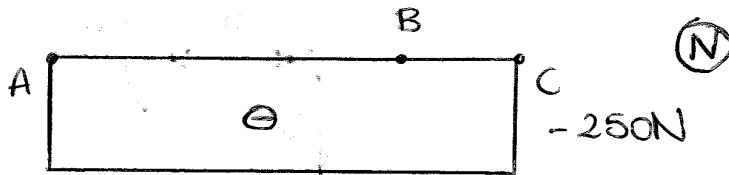
tratto CB: $0 < z < a$



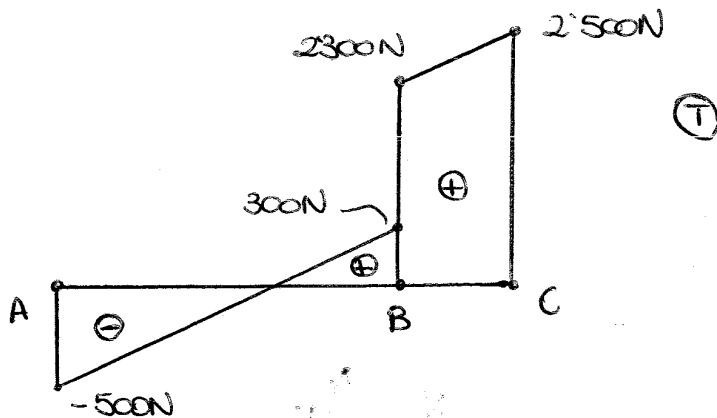
$$N = -H_c = -250\text{N}$$

$$T = V_c - F_q'' = \begin{cases} T_c = 2500\text{N} \\ T_B = 2300\text{N} \end{cases}$$

$$M = M_c - V_c \cdot z + F_q'' \cdot \frac{z}{2} = \begin{cases} M_c = M_c = 400000\text{N}\cdot\text{mm} \\ M_B = -80000\text{N}\cdot\text{mm} \end{cases}$$



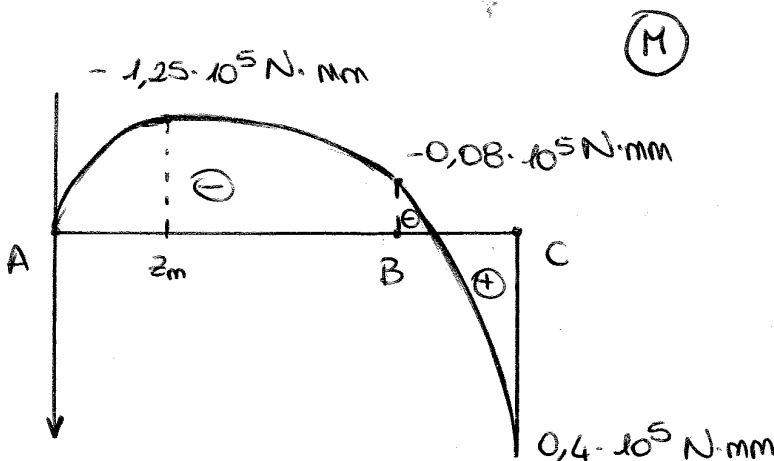
deb trovare il punto di max del momento:



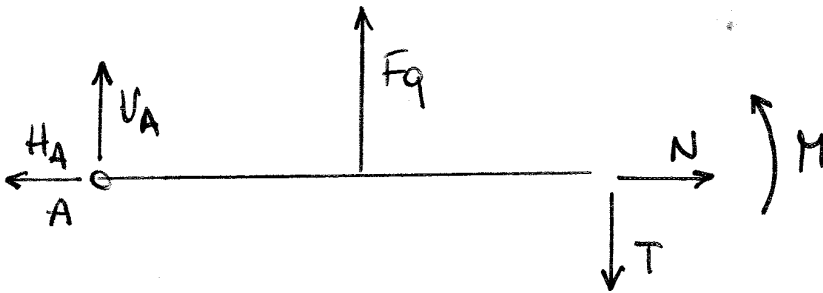
$$\frac{dM_{AB}}{dz} = 0 \rightarrow T_{AB} = 0$$

$$qz_m = P_2 \rightarrow z_m = 500\text{mm}$$

$$M = -P_2 \cdot 500 + F_q'' \cdot 250 = -1,25 \cdot 10^5 \text{N}\cdot\text{mm}$$



tratto \overline{AB} : $0 < z < \ell$

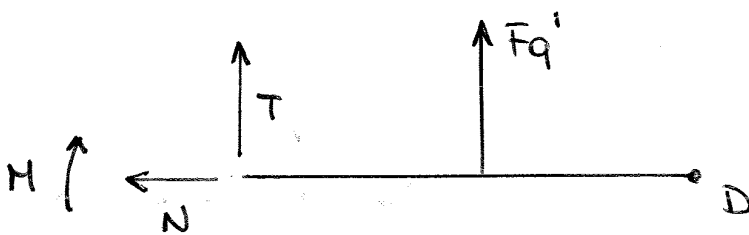


$$N = H_A = -3000 \text{ N}$$

$$T = V_A + \frac{F_q}{3} = \begin{cases} T_A = 1000 \text{ N} \\ T_B = -1666 \text{ N} \end{cases}$$

$$M = V_A \cdot z + \frac{F_q}{3} \cdot \frac{z}{2} = \begin{cases} M_A = 0 \text{ N} \cdot \text{mm} \\ M_B = 2666 \cdot 10^3 \text{ N} \cdot \text{mm} \end{cases}$$

tratto \overline{DB} : $0 < z < 2\ell$



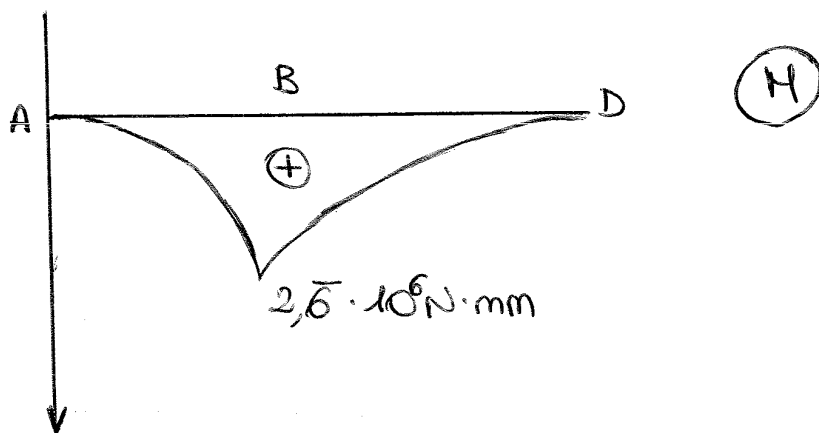
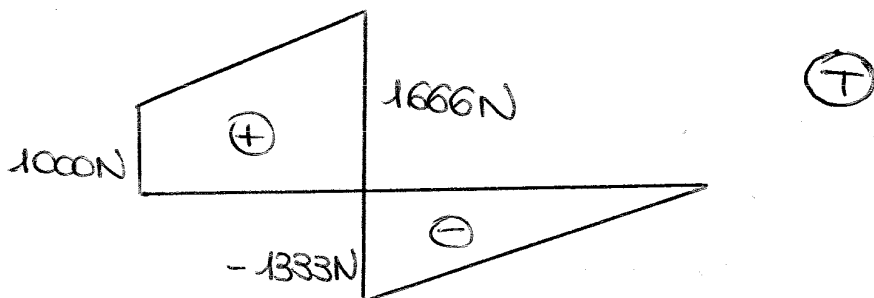
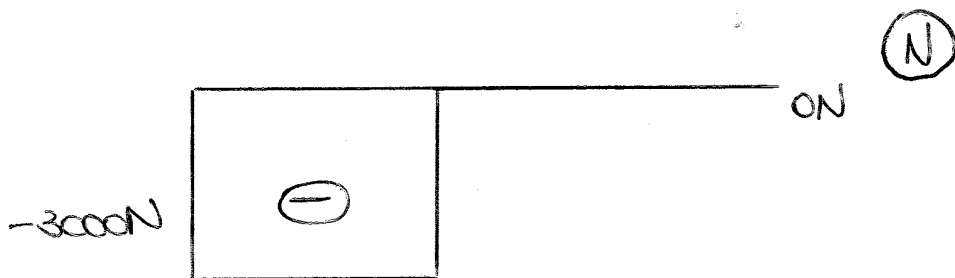
$$F_q' = \frac{2}{3} F_q$$

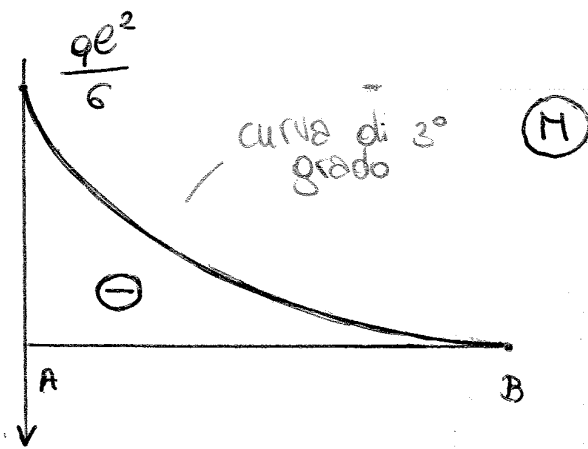
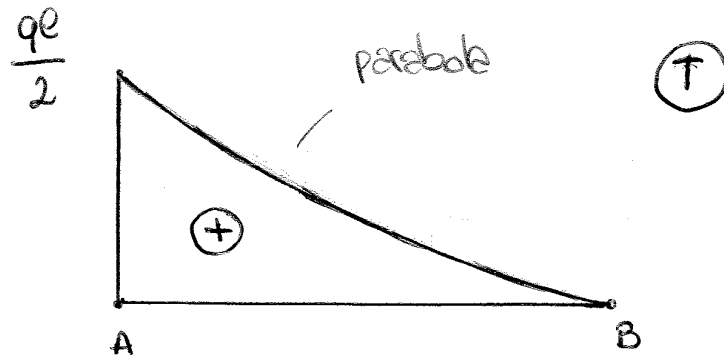
$$N = 0$$

$$T = -F_q' = \begin{cases} T_D = 0 \text{ N} \\ T_B = -1333 \text{ N} \end{cases}$$

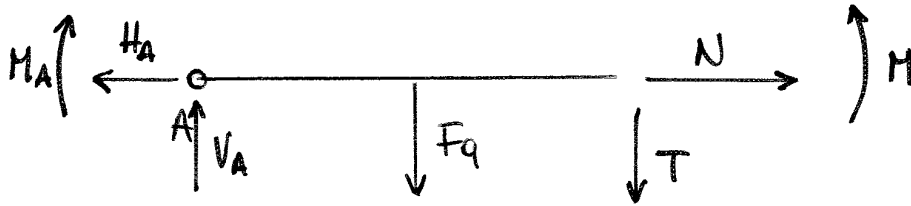
$$M = F_q' \cdot \frac{z}{2} = \begin{cases} M_D = 0 \text{ N} \cdot \text{mm} \\ M_B = 2666 \cdot 10^3 \text{ N} \cdot \text{mm} \end{cases}$$

CORPO 1



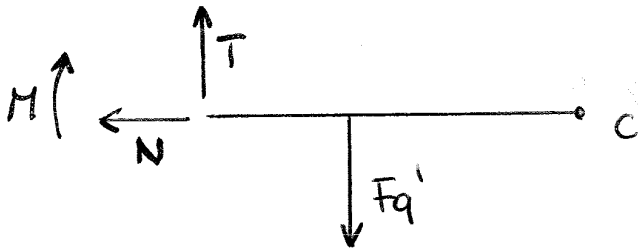


tratto \overline{AB} : $0 < z < e$

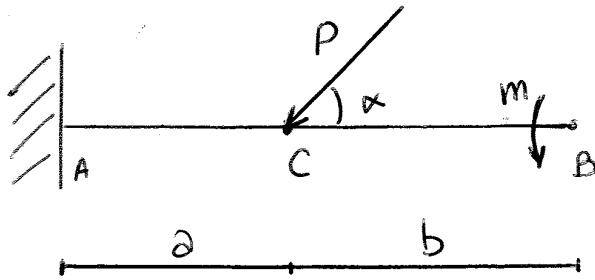


$$\begin{aligned}
 N = H_A = 0 & \quad T_A = V_A = \frac{3}{2} qe \\
 T = V_A - F_q \cdot z = \frac{3}{2} qe - qz & \quad T_B = V_A - F_q \cdot e = \frac{qe}{2} \\
 M = \underbrace{M_A}_{V_A \cdot z} - qz \cdot \frac{z}{2} = \frac{3}{2} qez - \frac{qz^2}{2} & \quad M_A = H_A \\
 & \quad M_B = M_A - \frac{qe^2}{2} + \frac{3}{2} qe^2 = -\frac{1}{6} qe^2
 \end{aligned}$$

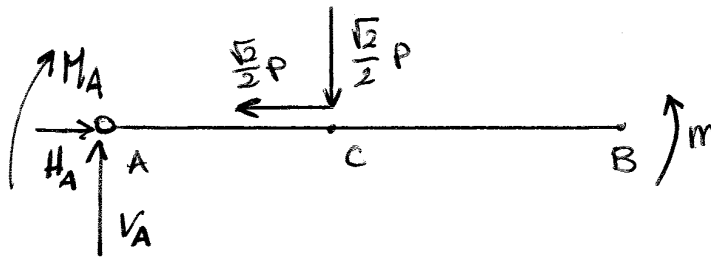
tratto \overline{CB} : $0 < z < e$



$$\begin{aligned}
 N = 0 & \quad T_C = 0 \\
 T = F_q' \cdot z = \frac{3}{2} qe - qz & \quad T_B = \frac{qe}{2} \\
 M = F_q' \cdot \frac{z}{3} \cdot z = \frac{3}{2} qez - \frac{qz^2}{2} & \quad M_C = 0 \\
 & \quad M_B = -\frac{qe^2}{6}
 \end{aligned}$$



$\alpha = 45^\circ$
 $P = 10 \text{ N}$
 $m = 2000 \text{ N}\cdot\text{mm}$
 $a = 400 \text{ mm}$
 $b = 600 \text{ mm}$



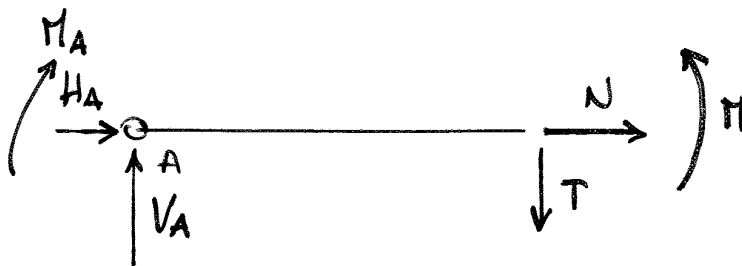
$\frac{P\sqrt{2}}{2} = 7 \text{ N}$

$\rightarrow H_A = \frac{\sqrt{2}}{2}P = 7 \text{ N}$

$\uparrow V_A = \frac{\sqrt{2}}{2}P = 7 \text{ N}$

$\uparrow M_A = m - \frac{\sqrt{2}}{2}P \cdot a = -800 \text{ N}\cdot\text{mm}$

tratto AC: $0 < z < a$

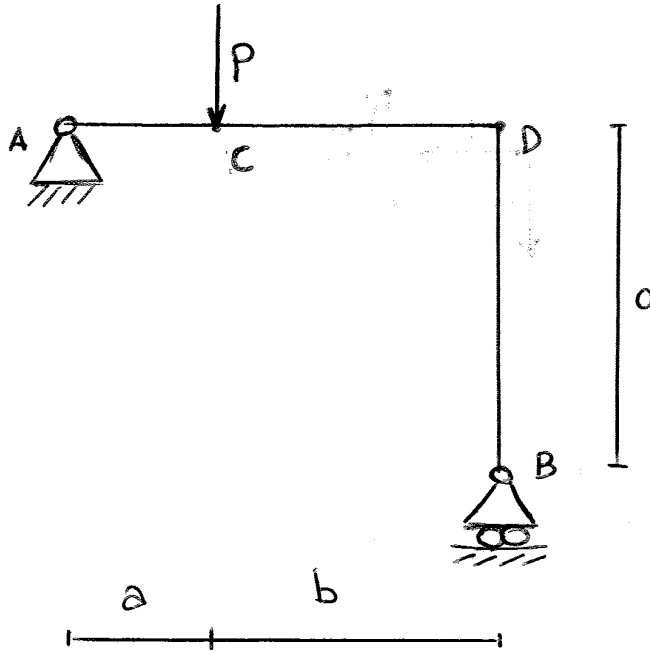


$N = -H_A = -7 \text{ N}$

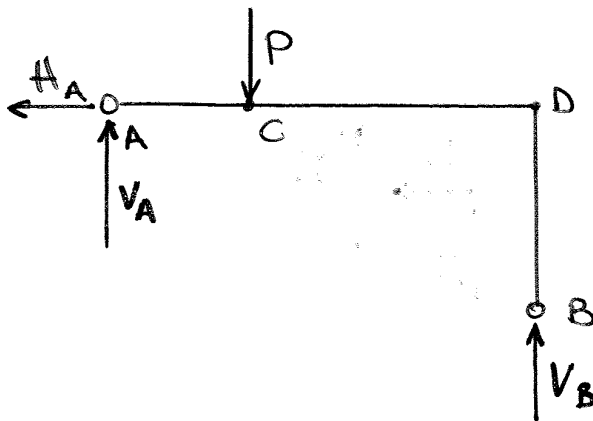
$T = V_A = 7 \text{ N}$

$M = M_A + V_A \cdot z = \begin{cases} M_A = M_A = -800 \text{ N}\cdot\text{mm} \\ M_C = M_A + V_A \cdot 0 = 2000 \text{ N}\cdot\text{mm} \end{cases}$

8



$P = 20\text{ N}$
 $a = 400\text{ mm}$
 $b = 600\text{ mm}$
 $d = 500\text{ mm}$



$$\leftarrow H_A = 0$$

$$\uparrow V_A + V_B = P$$

$$\uparrow \sum M_A: P \cdot a - V_B \cdot (a + b) = 0$$

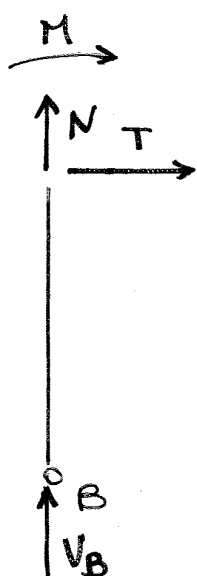
$$H_A = 0\text{ N}$$

$$V_A = 12\text{ N}$$

$$V_B = 8\text{ N}$$

9.1

tratto \overline{BD} : $0 < z < d$



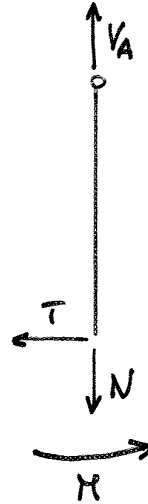
$$N = -V_B = -EN$$

$$T = 0$$

$$M = 0$$

9.2

TEORIA 6



$$\begin{aligned} N &= V_A = mg \\ T &= 0 \\ M &= 0 \end{aligned}$$

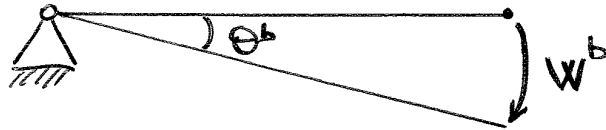
$$\sigma_R = \frac{N}{A} \quad \rightarrow \quad N = \sigma_R A$$

$$mg = \underbrace{\rho L_R A}_{m} g = \sigma_R A$$

$$\sigma_R = \rho L_R g \quad L_R = \frac{\sigma_R}{\rho g}$$

①

sistema "b" → congruente (virtuale)



$$W_B^b = (a+b)\theta^b$$

$$W_P^b = a\theta^b$$

$$\delta W_B^b = (a+b)\delta\theta^b$$

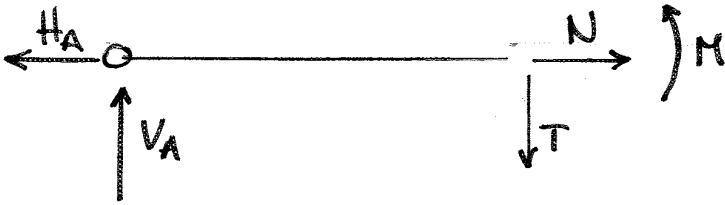
$$\delta W_P^b = a\delta\theta^b$$

$$\delta L_i^{ab} = 0 \quad \delta L_e^{ab} = 0 = P\delta W_P^b - V_B^a\delta W_B^b$$

$$P \cdot a\delta\theta^b = V_B^a \cdot (a+b)\delta\theta^b$$

sistema "b"

tratto \overline{AC} : $0 < z < e$

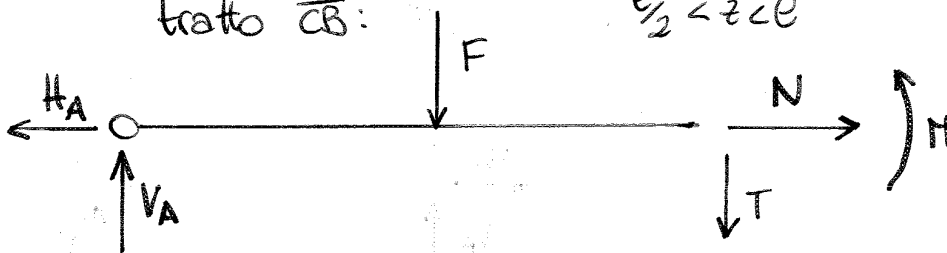


$$N = H_A = 0$$

$$T = V_A = \frac{1}{2}$$

$$M^b = V_A z = \begin{cases} H_A = 0 \\ M_C = \frac{e}{4} \end{cases}$$

tratto \overline{CB} :



$$N = H_A = 0$$

$$T = V_A = \frac{1}{2}$$

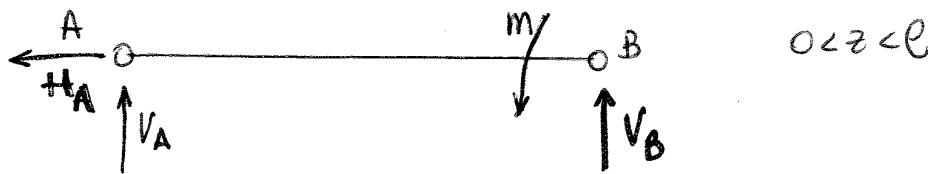
$$M^b = V_A \cdot z + 1 \cdot \left(\frac{e}{2} - z\right) = \begin{cases} M_C = \frac{e}{4} \\ M_B = \frac{3}{4}e \end{cases}$$

$$M^a = \frac{m}{e} \cdot z \quad 0 < z < e$$

$$M^b = \frac{z}{2} \quad 0 < z < \frac{e}{2}$$

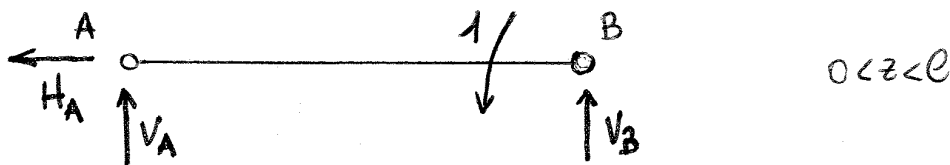
$$M^b = \frac{e-z}{2} \quad \frac{e}{2} < z < e$$

sistema "a": θ_B



$$M^a = \frac{m}{e} \cdot z$$

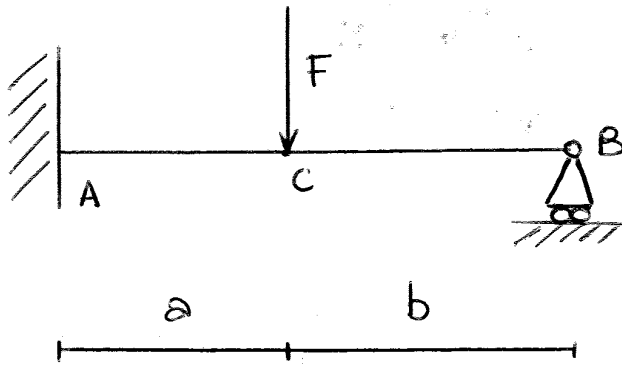
sistema "b": \rightarrow prendo il momento = 1



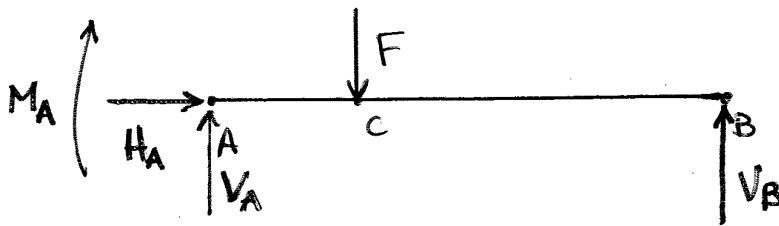
$$M^b = \frac{1}{e} \cdot z = \frac{z}{e}$$

$$L_i^{ab} = L_e^{ab} \quad L_e^{ab} = \theta_B^a$$

$$\begin{aligned} \theta_B &= \int_0^e \frac{M^a M^b}{EI} dz = \frac{m}{e^2 EI} \int_0^e z^2 dz = \\ &= \frac{me}{3EI} \end{aligned}$$

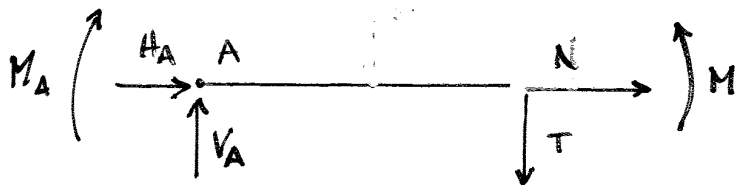


Determinare V_B



$$\begin{aligned} \rightarrow H_A &= 0 \\ \uparrow V_A + V_B &= F \\ \uparrow M_A &= V_B \cdot (a+b) - F \cdot a \end{aligned}$$

sistema "0": \rightarrow sistema in cui togli il vincolo di cui vuoi calcolare le reazioni;
 Etatto \bar{AC} : $0 < z < a$ valido per le iperstatiche



$$\begin{aligned} M_A^0 &= -Fz \\ H_A^0 &= 0 \\ V_A^0 &= F \end{aligned}$$

$$\uparrow V_A^0 = T$$

$$\rightarrow H_A^0 = N = 0$$

$$\uparrow M^0 = M_A^0 + V_A^0 \cdot z = \begin{cases} M_b^0 = M_A^0 \\ M_c^0 = M_A^0 + V_A^0 \cdot a \end{cases}$$

3.1

$$\int_0^{a+b} \frac{M^a (\pi^0 + x M^a)}{EI} dz = \int_0^{a+b} \frac{M^a \pi^0}{EI} dz + x \int_0^{a+b} \frac{(M^a)^2}{EI} dz$$

$$x = - \frac{\int_0^{a+b} M^a \pi^0 dz}{\int_0^{a+b} (M^a)^2 dz}$$

$$\int_0^{a+b} M^a \pi^0 dz = \int_0^a M^a \pi^0 dz + \int_a^{a+b} M^a \pi^0 dz = 0$$

$$= F \int_0^a (a+b-z)(z-a) dz = -F a^2 \left(\frac{b}{2} + \frac{a}{3} \right)$$

$$\int_0^{a+b} (M^a)^2 dz = \left[-\frac{(a+b-z)^3}{3} \right]_0^{a+b} = \frac{(a+b)^3}{3}$$

$$x = \frac{F a^2 \left(\frac{a}{3} + \frac{b}{2} \right)}{\frac{(a+b)^3}{3}} = F a^2 \frac{2a+3b}{2(a+b)^3}$$

$$V_B^b = \cancel{V_B^a} + 1 \cdot x = F a^2 \frac{2a+3b}{2(a+b)^3}$$

3.2